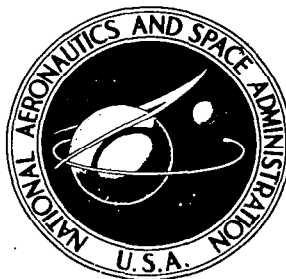


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**INLET SPILLAGE DRAG TESTS AND NUMERICAL
FLOW-FIELD ANALYSIS AT SUBSONIC AND
TRANSONIC SPEEDS OF A 1/8-SCALE,
TWO-DIMENSIONAL, EXTERNAL-COMPRESSION,
VARIABLE-GEOMETRY, SUPERSONIC
INLET CONFIGURATION**

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and R. L. Turner*

Prepared by

GENERAL DYNAMICS' CONVAIR AEROSPACE DIVISION

Fort Worth, Tex.

for Ames Research Center



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16. Abstract Accurate spillage drag and pressure data are presented for a realistic supersonic inlet configuration. Results are compared with predictions from a finite-differencing, inviscid analysis computer procedure. The analytical technique shows good promise for the evaluation of inlet drag but necessary refinements were identified. A detailed description of the analytical procedure is contained in the Appendix.					
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INLET SPILLAGE DRAG TESTS AND NUMERICAL FLOW-FIELD ANALYSIS
AT SUBSONIC AND TRANSONIC SPEEDS OF A 1/8-SCALE,
TWO-DIMENSIONAL, EXTERNAL-COMPRESSION, VARIABLE-
GEOMETRY, SUPERSONIC INLET CONFIGURATION

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General Dynamics' Convair Aerospace Division

SUMMARY

Inlet spillage drag tests were conducted in the NASA Ames Research Center's 6- by 6-foot wind tunnel with a 1/8-scale, two-dimensional, four-shock, horizontal-ramp, external-compression inlet model provided by General Dynamics' Convair Aerospace Division and a ducted force-balance system provided by the Fluidyne Engineering Corporation. The purpose of the investigation was to obtain accurate spillage drag measurements and pressure data on a realistic supersonic inlet configuration and to compare the results with a two-dimensional, finite-differencing, inviscid, flow-field-analysis computer procedure under development at the Convair Aerospace Division, Fort Worth, Texas.

Experimental data were obtained at four ramp positions, at capture-area ratios of from 0.40 to inlet choking, and at Mach numbers of from 0.55 to 1.39. All data were obtained at zero degrees angle of attack and at a nominal tunnel Reynolds number per foot of 2.5×10^6 .

Generally, the experimental data were consistent and provided the expected trend of decreasing spillage drag with increasing capture-area ratio and decreasing inlet throat area. Large losses in inlet total pressure recovery and increases in compressor face distortion were observed at high inlet-throat Mach numbers. The choking inlet-throat Mach number observed, based on geometric throat area, was 0.80 or less. These data provide a basis for determining the trade-off between inlet spillage drag and pressure recovery for practical design applications.

Computer data were generated for two subsonic test conditions and compared with the experimental results. The agreement in surface pressure distributions was excellent along the ramp surface, including the throat region and inside the cowl lip. Computed results along the external cowl surface were qualitatively correct but quantitatively less accurate. Computations with a finer mesh showed improvements in accuracy, which suggested that further improvement would be possible with an even finer mesh. A third iteration was not possible, however, because of core storage limitations of the CDC 6600 computer.

INTRODUCTION

Inlet spillage drag is currently recognized as an important consideration in the design and operation of variable-geometry, external-compression inlets for tactical and strategic supersonic military aircraft. Analytical techniques have not been developed nor do sufficient experimental data exist to predict inlet spillage drag adequately for supersonic inlets, particularly at subsonic and transonic speeds.

The purpose of this investigation was (1) to obtain accurate inlet spillage drag data on a typical two-dimensional, variable-geometry, supersonic inlet configuration over a wide range of geometry variations and subsonic and transonic test conditions, (2) to obtain external and internal pressure distributions to enable a detailed flow-field analysis, and (3) to compare the test data with predictions from a two-dimensional, finite-differencing, inviscid, flow-field-analysis computer procedure under development at General Dynamics' Convair Aerospace Division's Fort Worth Operation.

The testing was accomplished at the NASA Ames Research Center's 6- by 6-foot wind tunnel with a 1/8-scale, two-dimensional, four-shock, horizontal-ramp inlet model, provided by General Dynamics' Convair Aerospace Division, and a ducted force-balance system, provided by the FluidDyne Engineering Corporation. Both FluidDyne and General Dynamics personnel were present during the model installation and test period. FluidDyne was primarily responsible for the proper installation, calibration, and operation of the balance system, and General Dynamics was responsible for overall test direction. The test was conducted under contract NAS2-7210 during the period 26 March to 12 April 1973. Data were obtained at zero degrees angle of attack at Mach numbers of 0.55, 0.70, 0.85, 0.88,

1.20, and 1.39. The nominal test Reynolds number per foot was 2.5×10^6 , with variations to 1.5×10^6 and 3.5×10^6 for one configuration.

This report presents the total inlet drag (the sum of the additive drag and the external cowl drag forward of the balance windshield) in coefficient form for each data point, the average total pressure recovery at the simulated engine compressor face and a distortion parameter for each data point, and selected static pressure distributions on the ramps and lower cowl lip, illustrating the effects of capture-area ratio, ramp angle, and Mach number. Data for two test conditions were selected and compared with the finite-difference program predictions. The complete pressure data listings from which component drags could be calculated and compressor-face total pressure profiles determined are given in Reference 1.

Considerable attention was given to obtaining the highest level of accuracy possible with the test equipment. A discussion of data accuracy is presented herein.

A description of the finite-difference flow-field-analysis computer procedure under development at the Fort Worth Operation is given in the Appendix.

SYMBOLS

A_0	the area based on freestream conditions, required to accept the inlet mass flow
A_i	inlet capture area, 24.887 in.^2 (Model Scale, Configurations 1-5)
A_0/A_i	inlet capture area ratio
A	area, sq in.
C_D	inlet drag coefficient (includes external skin friction), $\frac{D}{q_0 A_i}$
$C_{D_{spill}}$	inlet spillage drag coefficient, $C_D(A_0/A_i \neq 1.0) - C_D(A_0/A_i = 1.0)$
C_F	cowl external-skin-friction drag coefficient, <u>Friction Drag</u> $q_0 A_i$

C_f	skin friction coefficient, $C_{fi}(1 + 0.1296 M_o^2)^{-0.648}$
C_{fi}	incompressible-skin-friction coefficient, $\frac{0.455}{(\log_{10} RN_i)^{2.58}}$
C_d	choked ASME nozzle discharge coefficient, $1 - 0.184 (R_{N6})^{-0.2}$
C_p	pressure coefficient, $P - P_o / q_o$
C_T	choked ASME nozzle thrust coefficient, $1 - 0.116 (R_{N6})^{-0.2}$
D	total inlet drag (additive drag + cowl drag)
D_f	Friction drag
D_{add}	additive drag
D_{cowl}	external cowl drag
$Dist$	distortion, $(P_{te_{max}} - P_{te_{min}}) / P_{te_{avg}}$
F	force, lb
g	acceleration due to gravity, ft/sec ²
H	measured force, lb
M	Mach number
\dot{m}	mass flow function, $W \sqrt{T_t / P_A}$
m	mass flow W/g
P	pressure, psia
p	pressure, psia
\bar{P}_{te} / P_{t_o}	compressor face total pressure recovery (average)
q	dynamic pressure, psi, $\gamma / 2 M^2 P_o$
R_{Ni}	inlet Reynolds number

R_{No} tunnel Reynolds number/ft

T temperature, °R

V velocity, ft/sec

W air weight flow, lb/sec

γ specific heat ratio

Subscripts

A ambient or freestream static pressure

e engine or compressor face

i inlet

o freestream

t stagnation

TH throat

w wetted

1 conic lip

2 balance strain gage

3 balance seal

4 inlet base

6 ASME nozzle system

Avg average

TEST EQUIPMENT AND EXPERIMENTAL METHODS

Facility Description

The test program was conducted in the NASA/Ames Research Center's 6- by 6-foot Supersonic Wind Tunnel. This is a closed-circuit single-return tunnel. It has an asymmetric sliding-block nozzle and a test section with perforated floor and ceiling to permit transonic testing. An eight-stage axial-flow compressor driven by two electric motors provides Mach numbers from 0.55 to 2.2. Details of the tunnel test section are shown in Figure 1.

For these tests, the inlet model was attached to the FluidDyne force balance (a flow-metering and force-measuring unit), which was supported by the tunnel sting body and its support system. The metric break on the model was at model station 28.5. A photograph of the installation is presented in Figure 2.

A 10-inch-I.D. pipe carried the air from the flow-metering unit along the tunnel floor for about 38 feet, then through the tunnel floor to the facility vacuum manifold (Figure 3).

Model Description

The model is a 1/8-scale simulation of a three-ramp, two-dimensional, variable throat external-compression supersonic inlet having a design Mach number of 2.2. Model variations consisted of changing the inlet ramp angles and throat area. The variations tested are shown in Figure 4. The model external contour was the same for each configuration. The internal and external shape of the inlet is shown by the cross-sections in Figure 5. The internal-duct-flow area distributions are given in Figure 6, and the nacelle normal area distribution in Figure 7. The wetted area for the nacelle is 588 in². Cowl and sideplate leading-edge geometry is presented in Figure 8. All dimensional data are for model scale.

An assembly drawing of the model and force balance is presented in Figure 9; a tunnel installation drawing is presented in Figure 10. Configuration changes were accomplished by removing one ramp section and replacing it with

another. The lower lip section, sideplates, and initial 7° ramp wedges were common to all configurations. Since the upper limit on test Mach number was 1.39, boundary-layer bleed was not incorporated on the model.

Balance Operation

The FluidDyne force balance is shown schematically on page 12. This flow-metering and force-measuring unit permits direct calculation of the drag on the inlet model. Isolation between the metric and nonmetric parts of the ducting system is provided by a thin rubber seal surrounding the flow metering nozzle. Calibration of the balance is described on page 8.

Inlet mass flow control was provided by a variable-area choke plate consisting of a stationary porous plate and an overlaid motor-driven movable plate. The variable-area choke plate was located a short distance upstream of the flow meter. By changing the flow area through the choke plate, the inlet back pressure was varied and, hence, the capture-area ratio. Several screens followed the choke plate to provide additional flow straightening. To insure that the ASME nozzle was choked during testing, inlet air was exhausted by the facility evacuation system.

FluidDyne personnel were primarily responsible for the proper installation, calibration, and operation of the balance system during the test.

Instrumentation

Model. - The location of the inlet static pressure instrumentation for inlet Configurations 1, 2, 4, and 5 is defined in Figure 11. The compressor-face rake numbering system is shown in Figure 12. The pressure instrumentation is summarized as follows:

<u>Configuration</u>	<u>Common Structure</u>		<u>Ramp</u>		<u>Total</u>
	<u>External</u>	<u>Internal</u>	<u>External</u>	<u>Internal</u>	
1	63	26	5	30	124
2				27	121
4				27	121
5				10	104
Compressor-face-rake totals					100
Compressor-face statics					4
					7

All inlet and compressor-face pressures were measured on the facility scanivalve system. Cycle time was set at 70 sec.

Pressure tubes were approximately 8 feet long. Tubes which originated at the model as 0.036-in. O.D. were spliced to 0.065-in.-O.D. tubing to reduce lag time, but most were 0.065-in.-O.D. for their complete length. A lag-time check was made at the beginning of the test to establish the required pressure stabilization time.

Balance. - The balance instrumentation consisted of one strain-gage bridge for the axial load, a digital voltmeter with a sensitivity of approximately 70 counts per pound of applied load for the balance output, and a mercury manometer for balance pressures - with the exception of P₇, which was referenced to a tunnel wall static pressure tap through a water-filled U tube.

The balance was calibrated before each run by applying a series of loads in 50-pound increments. Weights were applied axially up to 200 pounds, then removed in 50-pound increments. This process was repeated three times for each calibration. Data from the last two cycles were used to establish the gage factor.

ASME nozzle. - An ASME long-radius nozzle is an integral part of the flow-metering and force-measuring unit. Mass flow data from the nozzle were used to compute capture-area ratio. To insure that the nozzle was choked during the test, the inlet air was exhausted through a system of piping to the facility evacuation system.

Test Conditions

Range of operating conditions. - Test Mach numbers were 0.55, 0.70, 0.85, 0.88, 1.20, and 1.39. At each Mach number the choke plate was adjusted to provide from two to seven values of A_0/A_1 . All data were obtained with the model at zero degrees angle of attack. The nominal test Reynolds number per foot was 2.5×10^6 . On configuration 2, at Mach numbers 0.70 and 1.39, data were also obtained at Reynolds numbers per foot of nominally 1.5×10^6 and 3.5×10^6 . A summary of the test as run is given in Table I.

Test procedure. - At each Mach number, the choke plate was adjusted to obtain the desired mass flow through the

inlet; data were then recorded. The choke plate was then remotely adjusted for other mass flow rates, as desired. The time between data points was in excess of three minutes.

Tunnel pressures were recorded on digital readout mercury manometers. All inlet pressures were measured by the facility scanivalve system. Balance force data were recorded by an automatic printout device from the digital voltmeter. Balance pressures were recorded by photographing the manometer board.

Computations

Compressor-face conditions. - The average total pressure recovery, \bar{P}_{te}/P_{t0} , distortion $(P_{te_{max}} - P_{te_{min}})/\bar{P}_{te}$, and Mach number, M_e , at the simulated engine compressor face were computed from the static and total pressure instrumentation shown in Figure 12. If a tube was plugged or broken off, it was deleted from all listings and computations.

The average Mach number, M_e , at the compressor face is computed as follows:

$$M_e = \left[\frac{\left(\frac{\bar{P}_{te}}{\bar{P}_e} \right)^{2/7} - 1}{0.2} \right]^{1/2}$$

where \bar{P}_e is the average of four static pressures at the compressor face.

Inlet external skin-friction drag coefficient. - The inlet external skin-friction drag, which is a part of the measured balance force, is computed in coefficient form as follows:

$$C_F = \frac{\text{friction drag}}{q_0 A_i}$$

where

$$\text{friction drag} = C_f q_0 A_w$$

and where

$$A_w = 588 \text{ sq. in. for Configurations 1, 2, 4, and 5}$$

$$C_f = C_{fi} (1 + 0.1296 M_o^2)^{-0.648} \quad \text{(includes compressible effects (Ref. 2))}$$

$$C_{fi} = \frac{0.455}{(\log_{10} R_{Ni})^{2.58}} \quad \text{(the Prandtl-Schlichting equation for incompressible flow)}$$

$$R_{Ni} = \text{inlet Reynolds no.} = R_{No} \times 1.94$$

$$R_{No} = \text{tunnel Reynolds no./ft}$$

$$1.94 = \text{model average length in ft (Configs. 1, 2, 4, 5)}$$

Inlet capture ratio. - Capture-area ratio, defined as the ratio of the cross-sectional area of the captured free-stream tube area, A_o , to the inlet geometric reference area, A_i , is calculated as follows:

$$A_o/A_i = \left[\frac{\text{Inlet Mass Flow}}{\text{Mass Flow per Unit Area at Freestream Conditions}} \right] \quad \frac{1}{A_i} = \left[\frac{W_i}{\frac{W_o}{A_o}} \right] \frac{1}{A_i}$$

where

$$W_i = W_o$$

$$\text{Inlet Mass Flow, } W_i = \frac{(P/P_{tm})_6 C_{d6} A_6 P_{t6}}{\sqrt{T_{t6}}}$$

where

$$(P/P_{tm})_6 = \frac{P_6}{P_{t6}} \sqrt{\gamma/R} M_6 \left[1 + \frac{\gamma-1}{2} M_6^2 \right]^{1/2}$$

and C_{d6} , the discharge coefficient of the choked, standard, long-radius ASME nozzle used to meter the inlet flow, is a function of flow-meter Reynolds number,

$$C_{d6} = 1 - 0.184 (R_{N6})^{-0.2}$$

and $M_6 = 1.0$

For air, $\gamma = 1.4$

$$R = 1716.322 \text{ ft}^2/\text{sec}^2\text{-}^\circ\text{F}.$$

Therefore

$$(P/P_{t6})_6 = 0.53177$$

and

$$W_i = \frac{0.53177 C_{d6} A_6 P_{t6}}{\sqrt{T_{t6}}}$$

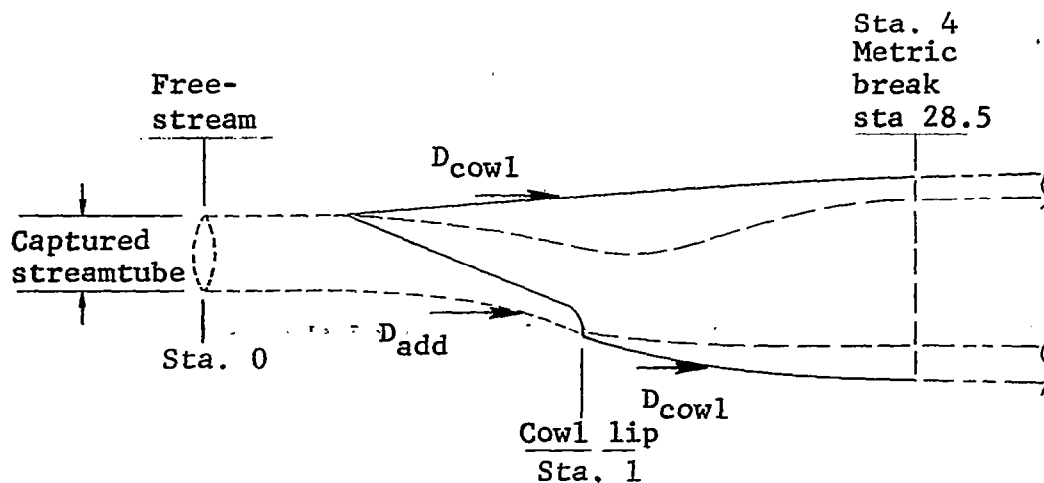
$$\dot{m}_o = \frac{W_o \sqrt{T_{t_o}}}{P_A A_o} = g \sqrt{\gamma/R} M_o \left[1 + \frac{\gamma-1}{2} M_o^2 \right]^{1/2}$$

$$\begin{aligned} W_o/A_o &= g \sqrt{\gamma/R} P_A / \sqrt{T_{t_o}} M_o \left[1 + \frac{\gamma-1}{2} M_o^2 \right]^{1/2} \\ &= 0.9189 P_A / \sqrt{T_{t_o}} M_o \left[1 + \frac{\gamma-1}{2} M_o^2 \right]^{1/2} \end{aligned}$$

Finally,

$$A_o/A_i = \left[\frac{0.53177 C_{d6} A_6 P_{t6} \sqrt{T_{t_o}}}{0.9189 M_o \sqrt{1 + 0.2 M_o^2} P_A \sqrt{T_{t6}}} \right] \frac{1}{A_i}$$

Inlet drag coefficient. - Total inlet drag, D , is defined as the sum of the axial forces (pressure and friction) acting on the external cowl surface between Stations 0 and 28.5 plus the pressure force acting on the unbounded captured streamtube between the freestream reference station and the cowl lip. The pressure force acting on the unbounded captured streamtube is commonly known as additive drag. The forces are shown by the following sketch.

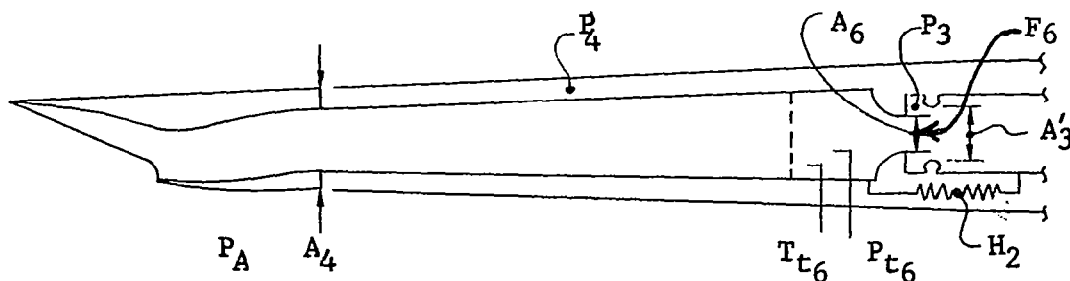


Total inlet drag is then defined as

$$D = D_{add} + D_{cowl} = \int_0^1 (P - P_A) dA + \int_1^4 (P - P_A) dA + D_f$$

$$D = \int_0^4 P dA - \int_0^4 P_A dA + D_f \quad (1)$$

A schematic of the model and balance arrangement is shown below.



The summation of the forces gives the equation

$$F_o + \int_0^4 P dA + D_f = H_2 + P_4 (A_4 - A_3') + P_3 (A_3' - A_6) + F_6 \quad (2)$$

where

$$F_o = \gamma M_o^2 P_A A_o + P_A A_o$$

and F_6 is the thrust of the choked ASME metering nozzle (Station 6), and A_3' is the effective area of the rubber seal isolating the metric and nonmetric parts. In equation form, F_6 is

$$\begin{aligned} F_6 &= P_6 A_6 + m_6 V_6 = P_6 A_6 (1 + \gamma C_{d6} C_{T6}) \\ &= 0.52828 P_{t6} A_6 (1 + \gamma C_{d6} C_{T6}) \end{aligned}$$

The discharge and thrust coefficients of the choked ASME nozzle are functions of the nozzle Reynolds number as follows:

$$C_{d6} = 1 - 0.184 (R_{N6})^{-0.2}$$

$$C_{T6} = 1 - 0.116 (R_{N6})^{-0.2}$$

Merging Equations (1) and (2) and rearranging,

$$D = H_2 + P_4 (A_4 - A_3') + P_3 (A_3' - A_6) + F_6 - \left[F_0 + \int_0^4 P_A dA \right]$$

Considering the last two terms in the equation,

$$\begin{aligned} F_0 + \int_0^4 P_A dA &= \gamma M_0^2 P_A A_0 + P_A A_0 + P_A (A_4 - A_0) \\ &= \gamma M_0^2 P_A A_0 + P_A A_4 \end{aligned}$$

Therefore,

$$D = H_2 + P_4 (A_4 - A_3') + P_3 (A_3' - A_6) + F_6 - \gamma M_0^2 P_A A_0 - P_A A_4$$

or, rearranging,

$$D = H_2 + F_6 - P_3 A_6 + (P_4 - P_A) A_4 + (P_3 - P_4) A_3' - 1.4 M_0^2 P_A A_0$$

The drag coefficient is obtained from the drag value by dividing by the product of the freestream dynamic pressure, q_0 , and the reference inlet area, A_i .

$$C_D = \frac{D}{q_0 A_i}$$

Data Accuracy

Error in facility and balance measurements. - The precision of measurements of both the basic-facility-measured and the balance-measured parameters are as follows:

Facility-Measured Pressures and Temperatures

$$P_{t_0} \pm .01 \text{ "Hg} = \pm .0049 \text{ psi}$$

$$P_A \pm .01 \text{ "Hg} = \pm .0049 \text{ psi}$$

$$P_{WALL} \pm .01 \text{ "Hg} = \pm .0049 \text{ psi}$$

$$P_x \pm 3/4 \text{ of } 1\% \text{ of } 12.5 \text{ psi} = \pm .094 \text{ psi}$$

$$T_{t_0} \pm 2^\circ\text{F}$$

Balance-Measured Pressures and Forces

$$\begin{aligned} P_4 &= P_{WALL} + P_7^* \\ &= (\pm .0049) + (\pm .0025) = \pm .0055 \text{ psi (RMS)} \end{aligned}$$

$$P_3 = \pm .017 \text{ psi (RMS)}$$

$$P_{t_6} = \pm .017 \text{ psi (RMS)}$$

$$H_2 \pm .25 \text{ lb (from force calibrations)}$$

On the basis of the facility values and a statistical analysis (RMS) of the possible errors in measurement, 68.3 percent of the data for the two ratios P_x/P_{t_0} and C_p should be within the limits shown below

	<u>$M_0 = 0.55$</u>	<u>$M_0 = 1.39$</u>
P_x/P_{t_0}	$\pm .007$	$\pm .011$
C_p	$\pm .026$	$\pm .044$

* $P_7 = (P_4 - P_{wall})$ - measured with water filled U tube manometer

The relatively large potential error in the scanivalve transducer output accounts for most of the above-noted potential error in the ratios.

Error in A_0 . - The two primary parameters provided solely by the balance system were A_0/A_1 , inlet capture-area ratio, and C_{D_1} , inlet drag coefficient. The equation used to calculate A_0 when the ASME flow nozzle is choked is shown below. During the test, the nozzle was choked at all data points but one, which was at a very low Reynolds number.

$$A_0 = \frac{0.53177 A_6 C_{d_6} P_{t_6}}{\dot{m}_0 P_A} \sqrt{\frac{T_{t_0}}{T_{t_6}}}$$

where \dot{m}_0 is a flow function, which for air, is

$$\dot{m}_0 = .9189 M_0 \sqrt{1 + .2 M_0^2}$$

and it is assumed that $T_{t_0} = T_{t_6}$.

The accuracy of the A_0 calculation can be determined by examining the effects of possible errors in the parameters of the equation. A_6 , the nozzle throat area, is a constant. Since C_{d_6} does not change in value in the fourth decimal place even at up to a 2-percent change in RN_6 , it is eliminated as an error source. Individual errors in P_{t_6} , P_A , and M_0 could cause an error in A_0 , as listed below. Thus, most errors in A_0 would be quite small.

<u>Error</u>	<u>Percent Error in A_0</u>	
	<u>$M_0 = 0.55$</u>	<u>$M_0 = 1.39$</u>
$M_0 \pm .001$	$\pm 0.193\%$	$\pm 0.027\%$
$P_{t_6} \pm .017 \text{ psi}$	$\pm 0.161\%$	$\pm 0.361\%$
$P_A \pm .0049 \text{ psi}$	$\pm 0.046\%$	$\pm 0.181\%$
RMS of error %	$\pm 0.255\%$	$\pm 0.405\%$

Error in C_D . - The balance output proved to be quite linear. For one typical calibration involving 16 readings, the average displacement of a point from a selected calibration straight line was 0.143 lb. So that the sensitivity of the inlet drag equation to various errors could be evaluated, a particular data point was chosen for examination. The example which follows shows the magnitude of the various equation terms and the effect of errors on these terms. By combining the potential error (in lb) for each of the equation parts into an RMS value, a numerical value is obtained which is statistically representative of repeatability within which at least 68.3 percent of the data should fall.

Inspection of the plotted data that follows will show that probably no more than two or three data points fall outside this limit and that data scatter from a smoothly faired curve appears to be no more than one half of the RMS value calculated.

Example: Data correlation No. 144 ($M_O = 0.547$, $A_O/A_i = 0.9955$ Config. 5)

$$D = 6.183 \text{ lb} = H_2 + F_6 - P_3 A_6 + (P_4 - P_A) A_4 \\ + (P_3 - P_4) A_3' - 1.4 M_O^2 A_O P_A$$

<u>Term</u>	<u>Numerical Value</u>	<u>RMS Error in Pressure (psi)</u>	<u>Error in Force (lb)</u>
$H_2 =$	80.87		$\pm .25$
$F_6 = .52828 P_{t6} A_6 (1 + \gamma C_{D6} C_{T6})$	313.81057	$\pm .017$	$\pm .5069$
$P_3 A_6 = P_3 (23.758)$	81.41867	$\pm .017$	$\pm .4038$
$(P_4 - P_A) A_4 = (P_4 - P_A) 60.5$	1.2204	$\pm .00736$	$\pm .4452$
$(P_3 - P_4) A_3' = (P_3 - P_4) 29.46$	-201.75186	$\pm .01786$	$\pm .5261$
$\gamma M_O^2 A_O P_A =$	106.54930	$\pm .017$ ($\pm .001 M_O$)	$\pm .2873$

$$\text{RMS} = \sqrt{\sum \text{Errors}^2} \\ = 1.02 \text{ lb}$$

$$\text{Error in } C_D = \frac{1.02}{2.154 \times 24.887}$$

$$= 0.019$$

Zero Shift. - During the test of each configuration, a zero shift was encountered between the initial balance reading (reading before tunnel airflow) and the final reading (reading after tunnel shutdown and repressurization to atmospheric conditions). In each case, for the four inlet configurations, the shift was in the same direction. When converted to pounds of axial force and coefficients at a nominal Mach number of 0.85, the shifts were as follows:

<u>Configuration</u>	<u>Axial Force Shift (lb)</u>	<u>C_D</u>
1	+ .556	.008
2	+ .812	.011
4	+ .670	.009
5	+ .855	.012

$$\text{Max } \Delta = 0.855 - 0.556 = 0.299 \text{ lb}$$

As a possible source of these shifts, the temperature change during the course of a test was considered. An examination of the variation during tests of a given configuration showed no more than a 12°F variation. In addition, the final data of each test was at Mach 0.55, where the stagnation temperature was 77°F ±2°, which meant that the final soaking temperatures of the balance before each test termination was very close to the ambient temperature at which the balance was calibrated.

After the final test was complete, in an attempt to find the source of the shift, the model and the windshield were removed from the balance. After removal of these pieces, the balance zero immediately returned to that of the pre-run calibration. The conclusion reached was that the instrumentation tubing touching or rubbing against the windshield was responsible for the shift. It was also reasoned that, during actual operation, the effect of this rubbing would be minimized as a result of the vibration imposed on the balance by both the internal and external air streams. Thus, for final data reduction, the initial zero was used.

It is the conclusion of those conducting the test that the absolute inlet drag level is closer to the correct value when the initial zero is employed.

The plots in Figure 13 for each configuration present the increment in inlet drag coefficient that should be added to the listed data if the final zeros were to be used rather than the initial zeros.

If the final rather than the initial zeros had been employed, the total drag coefficient would be approximately 0.006 to 0.016 less, depending on Mach number and configuration.

EXPERIMENTAL RESULTS

Total Inlet Drag

Total inlet drag coefficient, C_D , is presented as a function of inlet capture-area ratio, A_0/A_1 , for Mach numbers of 0.55, 0.70, 0.85, 0.88, 1.2, and 1.39 in Figure 14. Inlet configuration (or ramp angle) is presented as a variable at each Mach number.

For each configuration, inlet mass flow was varied, with the upper limit determined by the inlet throat area or, as in the case of Configuration 5, by the balance choke plate maximum flow area. At each Mach number a locus of choke points for each configuration is established and extrapolated to unity capture area ratio. The unity-capture-area-ratio drag levels so determined are plotted as a function of Mach number in Figure 15. Although the extrapolation is somewhat arbitrary, the curve generated is smooth and, therefore, the spillage drag should be of reasonable accuracy.

Generally the data of Figure 14 are well behaved and provide the expected results of decreasing drag with increasing capture-area ratio and decreasing inlet throat area. The exception to the trend of most of the data is at Mach 1.39, where the drag of Configuration 5 is lower than that of Configuration 1. The reason is not obvious but could be associated with the complicated flow field set up by the re-expansion of the flow at the intersection of the first and second ramps on Configuration 5 and the resultant strong terminal shock and possible boundary-layer separation on the ramps.

At Mach numbers of 0.70 and above, and at a given inlet capture-area ratio, a significant inlet drag reduction can be realized by operating the inlet as near the choke point as possible. A tradeoff of course exists between the reduced inlet drag and reduced thrust due to pressure recovery degradation. (The effect of inlet throat area on pressure recovery and distortion is presented in Figure 18 and discussed subsequently).

The estimated cowl skin-friction drag is shown in Figure 16.

The effect of Reynolds number on total inlet drag is shown in Figure 17 for Configuration 2. Reynolds number variations from approximately 1.5×10^6 to 3.5×10^6 were made at Mach numbers of 0.70 and 1.39. Inlet drag coefficient is plotted as a function of Reynolds number at capture-area ratios of 0.45 and 0.62. Increasing Reynolds number decreases the measured drag up

to a Reynolds number per foot of about 2.5×10^6 at Mach 0.70 and 3.5×10^6 at Mach 1.39.

Pressure Recovery and Distortion

The effects of inlet capture-area ratio and throat area on total pressure recovery and distortion at the simulated engine compressor face is shown in Figure 18. The vertical line of capture-area ratio for each configuration corresponds to the capture-area ratio at which the inlet throat becomes choked or where further reductions in inlet back pressure will not produce any further increase in inlet mass flow.

Lines of constant theoretical throat Mach number, based on geometric throat area and assuming inviscid flow, are superimposed on the plotted data and show a choking Mach number of approximately 0.80 or less for each configuration at each freestream Mach number. A total pressure recovery at the inlet throat of 1.0 was assumed in calculating throat Mach number. It is obvious that, for inlet design and analysis purposes, a throat Mach number of less than 0.80 should be assumed to preclude large losses in total pressure recovery and high distortion.

The low total-pressure recovery for Configuration 5 at Mach 1.39 is due to the re-expansion of the flow at the intersection of the first and second ramps and the high total-pressure losses associated with the resulting strong terminal shock.

Reynolds number effects on pressure recovery and distortion (Mach 0.70 and 1.39) were not significant, based on the limited data obtained.

Ramp and Cowl Static Pressure Distribution

The effects of capture-area ratio, ramp angle, and Mach number on the ramp centerline static pressures are presented in Figures 19, 20, and 21, respectively. The lack of variation of ramp static pressure away from the inlet centerline is shown in Figure 22. Plots of the lower cowl centerline external pressures are presented in Figures 23, 24, and 25. The effect of capture-area ratio is shown for Configuration 1 at Mach 0.85; the effect of ramp angle is shown at nominally 0.60 capture-area ratio and Mach 0.85; and the effect of Mach number is shown for Configuration 1 at nominally 0.60 capture-area ratio. The remainder of the pressure data is reported in Reference 1.

The effect of decreasing inlet capture-area ratio on the ramp pressures at Mach 0.85 (Figure 19) is to increase the static pressure on the ramps forward to the first-ramp leading edge, with the biggest increase occurring on the third ramp and near the inlet throat. The high negative pressure coefficients near the inlet throat at 0.715 capture-area ratio are indicative of choked flow.

Increasing ramp angle or decreasing throat area at a constant capture-area ratio (Figure 20) has the effect of increasing pressure on the first and second ramp and decreasing pressures on the third ramp. Configuration 2 produces negative pressure coefficients near the inlet throat, again indicative of choked flow.

A large increase in ramp pressure occurs at Mach 1.20 (Figure 21), which partly results in the increase in drag noted earlier. A detached normal shock is produced ahead of the first-ramp leading edge at this Mach number. The shock is attached to the first-ramp leading edge at Mach 1.39, with the terminal-shock pressure rise clearly evident on the second ramp at approximately station 5.0.

In the case of the cowl lower centerline pressures, decreasing the capture-area ratio moves the captured streamline stagnation point further inside the lip, increases the velocity of the airflow adjacent to the external cowl surface, and results in a decrease in cowl pressure (Figure 23). The increase in pressure coefficient very near the lip leading edge at 0.493 and 0.448 capture-area ratios could be due to the local formation of a separation bubble at these low capture-area ratios.

Decreasing the inlet ramp angle (Figure 24) has the same effect as reducing the inlet capture-area ratio in that the captured-streamline stagnation point moves further inside the cowl lip and results in lower cowl pressure coefficients near the lip leading edge.

The effect of Mach number on cowl pressure distributions (Figure 25) shows high cowl pressure coefficients at Mach 1.2 and low pressure coefficients near the cowl leading edge at Mach 1.39.

COMPARISON OF EXPERIMENTAL DATA WITH FINITE-DIFFERENCE FLOW-FIELD SOLUTIONS

The Fort Worth Operation has developed a versatile finite-difference computer code capable of computing the entire flow field about inlets such as those tested in this program. The detailed pressure distributions acquired in the tests offered an excellent basis for an evaluation of the method of analysis upon which the computer code is based. These flow-field solutions, used for comparison with the experimental measurements, were computed as a part of the Convair Aerospace Division's IRAD program. Both the method of analysis and the computer code are described in detail in the Appendix; the application to these inlet flow fields is described in the following paragraphs.

The finite-difference solutions obtained and the corresponding test conditions are summarized in Table II. Each solution was obtained with a general patch arrangement, as illustrated in Figure 26. Each utilized the method of Godunov, with which the contractor has had the most experience and which has proven by virtue of its combined accuracy and stability to have the best overall performance of all the finite-difference methods incorporated in the computer code (see Appendix). Cell-node coordinates were hand-loaded along each patch boundary, both along the solid surfaces and in the far field. All interior cell-node points were automatically generated from these boundary coordinates by a routine built into the program. Because of the high design Mach number (2.2) of the tested inlet, the cowl lip radius is small compared to the inlet height. This necessitates the use of very small cells in the vicinity of the cowl lip. Yet the overall flow field must extend far enough from the inlet to minimize the impact of the far-field boundary conditions on the solution accuracy. Both of these criteria must be met within a core storage limitation of approximately 2000 total cells on the CDC-6600 computer utilized. Fortunately, the program permits wide variations in cell size, but, even so, the net result is a mesh that is not as fine as desired in any area of the flow field.

The tested inlet is essentially two-dimensional and was analysed as such in the finite-difference program. However, the subsonic duct must make a transition from the 2-D inlet to the circular compressor face. The geometry effects in this area can only be approximated by the 2-D analysis. This was done by maintaining actual contour on the ramp side and by adjusting the inside contour of the cowl to give the same 2-D flow area as does the

test inlet at corresponding stations. This results in essentially no contour modification in the first 7 inches of the 19 inches of the subsonic duct included in the computational control volume. Since the inlet height is approximately 5 inches, this was felt to be an adequate representation of the subsonic duct portion of the flow field.

Further details of the finite-difference solutions, including boundary conditions applied, are described below for each of the three cases analyzed.

Case I

Case I was selected before the current test program was started. It was chosen as a typical condition tested during the 1971 research conducted at Fluidyne Engineering Corporation with the same inlet model. Specifically, the test case chosen was for a freestream Mach number of 0.707, a capture-area ratio of 0.492, and the inlet ramp in Configuration 2.

The complete computational control volume employed is illustrated in Figure 27. (The patch numbering system is as given in Figure 26.) Portions of six patches in the vicinity of the inlet are given in Figure 28. Yet more detail in the region of the cowl lip is given in Figure 29. As can be seen from this figure, the cowl lip is treated as sharp. The computer code is quite capable of handling a body-oriented blunt-lip mesh; however, because of the scale of the lip radius to the overall inlet size and because the use of a blunt-lip mesh would have required more cells in the lip region (to the disadvantage of other portions of the flow field), the sharp-lip approach was chosen. Obviously, pressures predicted in the cells nearest the lip will be of questionable accuracy.

The Case I finite-difference solution was started from "free-stream" initial conditions (i.e., each flow property in each cell is initialized at the freestream value). Both the "characteristic time" and "cell-skipping" methods (see Appendix) were employed to reduce computation time. A total of 3200 time passes were computed before the solution was deemed to have achieved a true steady-state condition. Less iterations would have been needed had not boundary-condition changes been required after the first 1000 iterations. The final boundary conditions employed were:

- o Upstream boundaries: Density, horizontal velocity, and vertical velocity (actually zero) fixed at freestream values; pressure by linear extrapolation.
- o Top boundaries: Pressure fixed at the freestream value; velocity components and density by zeroth-order extrapolation.
- o Downstream (external) boundaries: Same conditions as top boundaries.
- o Solid boundaries: Pressure determined by Godunov shock-wave-analogy algorithm (see Appendix) as modified to account for surface curvature effects. All fluxes across boundary are zero.
- o Subsonic-duct downstream boundary: Horizontal velocity fixed at ideal value determined by inlet capture-area ratio, freestream Mach number, and boundary-to-inlet-area ratio; vertical velocity set to zero; pressure and density obtained by zeroth-order extrapolation.

Computed pressure distributions for the ramp, external cowl, and internal cowl lip are presented in Figures 30, 31, and 32, respectively. Also given on these figures are the experimental data from the test point run at the same condition - nominally Mach 0.7, capture-area ratio of 0.5 (the specific conditions are given in Table II). The computed and measured ramp surface pressure distributions of Figure 30 show good qualitative agreement, but, generally, higher pressures were predicted than were measured. This occurs both on the ramps and in the throat region. Agreement between calculations and experiment improves with nacelle station along each straight ramp section. This is not surprising considering the few computational cells located along each ramp (three on the first ramp and two on the second). Additional cells in this vicinity (impossible within the computational environment employed) would almost certainly have improved the agreement in this region. That higher throat pressure coefficient were predicted by the inviscid analysis than were measured is consistent with boundary-layer displacement effects, but better agreement was anticipated.

For the predicted and measured pressures along the external cowl surface (Figure 31), again, the finite-difference solution matches the qualitative nature of the experimental results, giving a negative pressure region just aft of the lip and little pressure variation aft of nacelle station 12. However, the experimental data indicate that C_p 's approach zero along the aft portion of the nacelle, while the computed solution does not achieve this logical asymptote. Clearly, the computed solution is inadequate along the external cowl and will result in a large error in the calculated lip suction force.

In the comparison of predicted and measured pressures just inside the cowl lip (Figure 32), considering the approximation of the cowl lip as sharp rather than blunt, the agreement shown in the figure is excellent.

In summary, the agreement between measurements and predictions was quite good with the exception of the external cowl surface. Solution of other flow fields, particularly around airfoils, with the same computer code have shown the solution accuracy to be a strong function of the number of cells used, particularly in expansion regions. Unfortunately, the mesh employed in Case I fully used the storagage capacity on the CDC 6600; however, a rearrangement of the cells was possible. Such a change was made, and the resulting solution is described as Case Ia below.

A plot of the flow-field streamlines for the Case I final solution is presented in Figure 33. The coordinates for this plot were obtained by integration of the mass flux along column boundaries. These calculations were performed within the computer code, and the resulting coordinates were hand-plotted. That the capture streamline corresponds to 99.2 percent of the input desired capture area is a measure of the error (0.8 percent) between the specified capture-area ratio and the value actually achieved at the final solution. This is quite good considering the rather indirect method in which this condition is imposed on the solution.

The value of a variety of inlet drag items (including additive drag, lower cowl drag, and total inlet drag) were determined or computed from both the experimental data and the computer solution. A comparison of these results is presented in Table III. For each item, the drag is presented in terms of a drag coefficient, based on the inlet area and freestream dynamic pressure. The experimental values were computed as follows: ramp drag is by pressure integration along the ramp, using centerline tap data, back to the point on the ramp that an estimated normal to the

flow passing through the cowl lip would intersect the ramp surface; additive drag is ramp drag plus a momentum difference term from freestream to the estimated inlet face location, based on one-dimensional isentropic relationships; lower cowl drag is by pressure integration, using lower cowl centerline tap data, from the stagnation point to the metric break; the drag designated "lower cowl + additive" is simply the sum of those two drags; the total drag is from the force balance, with bookkeeping corrections, and does include appropriate friction effects. While the experimental total drag includes pressure forces on the entire external surface, the computer model allows a cowl drag contribution from only one surface, that being the external surface of the lower cowl. The computed drags from the finite-difference solution were arrived at as follows: ramp drag is by pressure integration along the ramp to the same inlet-face point used in reducing the experimental data; additive drag is by pressure integration along the actual computed capture streamline; lower cowl drag is by pressure integration along the cowl from the cowl lip to the model metric break location; "lower cowl + additive" drag is again by addition of those terms; skin friction is a computed friction drag for the entire model external surface to the metric break.

As expected from the ramp pressure distribution comparison in Figure 30, the computed ramp drag exceeds the experimental values; numerically, it is about 39 percent higher. The computed momentum difference, having a C_D contribution of 0.005 to additive drag, agrees with the difference between ramp and additive drags determined from the computer solution; thus the computer solution additive drag is higher than the experimental value by the difference in ramp drag. As discussed above, very large discrepancies between measured and computed cowl pressures (see Figure 31) exist, and these are reflected in the large difference in lower cowl drag values. A portion of this difference is expected, however, because the two-dimensional analysis requires all spillage to occur over this one surface; spillage actually occurs over the other sides of the inlet, with corresponding lip suction effects; the experimental data confirms this effect. The combined lower cowl and additive drag shows the computed value to be low by 20 percent; this is partially due to the spillage effect discussed, but in fact is relatively good agreement and results from the offsetting direction of the errors in additive and cowl drags. The agreement between experimental and analytical total drags is quite good, the computed value being 10 percent higher than the corrected force-balance value. This agreement is, in part, fortuitous but does indicate that the sideplate suction effects are significant and that the two-dimensional analysis

shows good potential in the evaluation of what is in reality a three-dimensional flow field about a nominally two-dimensional inlet. No spillage-drag comparison was possible because the unity-capture case was not computed. Having such a solution would allow more detailed assessment of the analytical technique because the accuracy of computed drag increments could be determined.

Case Ia

In an attempt to improve the solution along the cowl surface, and yet utilize the Case I solution as a starting condition, patches 3 and 4 were compressed axially in the vicinity of the cowl lip. The overall computational control volume for the revised mesh is shown in Figure 34. The revised mesh in the vicinity of the inlet is shown in Figure 35, and more detail of the revised cell structure near the cowl lip is shown in Figure 36. The modifications are most obvious through a comparison of Figure 36 with Figure 29.

With the Case I solution as an initial condition, an additional 550 time passes were computed. While this did not yield a steady-state condition, it did result in certain improvements in the solution. The ramp pressure distribution for the revised mesh is shown in Figure 37. Comparison of this distribution with that of Figure 30 shows only slight improvement of the pressure distributions on the first two ramps (where the mesh remains unchanged) but shows a significant improvement in the throat region, where the agreement between experiment and computation is now quite good. This effect is felt to be due to the more accurate treatment of the flow field in the cowl-lip region permitted by the revised cell arrangement.

The cowl pressure distribution for the revised mesh is shown in Figure 38. While improvements are evident (see Figure 31), especially in the leading portion of the cowl, the agreement is not yet satisfactory. As stated above, this solution is not yet steady state; examination of the transient trends in the solution shows that all computed data points are continuing to approach the experimental pressure distribution. The changes shown between the Case Ia and Case I cowl pressure distributions are definitely the result of the mesh modifications and are not merely a function of the additional calculations performed. This is illustrated by the data of Figure 39, which shows the transient history at a typical location on the cowl before and after the mesh revisions. While some slight solution drift is still occurring prior to time

pass 3200, the marked change in the solution occurs in response to the mesh change. It is estimated that an additional 1.5 hours of computer time might be required for a final steady-state result. However, examination of the solution trends indicates that still further mesh refinements (i.e., more cells) would be required to achieve the ultimately desired accuracy.

The cowl lip internal pressure distribution for the revised mesh is shown in Figure 40, and, when contrasted with Figure 32, indicates yet further improvement in this region also.

Case II

The second condition chosen for solution with the finite-difference program is at the same freestream Mach number and with the same ramp configuration but at an increased capture-area ratio, which resulted in choking during the test runs. Three test points were run at the chosen nominal conditions of 0.7 Mach number and 0.62 capture-area ratio. These test points, listed in Table II, differed only in tunnel Reynolds number. The nominal (intermediate) test Reynolds number case has been chosen as a basis for evaluation of the finite-difference solution. However, excluding the cowl-lip region, the effect of Reynolds number on the pressure distributions is small; therefore, selected experimental data points for the higher and lower Reynolds number cases have been included in the comparisons to show the effect in the cowl lip region.

The revised mesh used for case Ia was used for Case II without modification, and the results of the Ia solution were used as an initial condition. Steady state was achieved after 1950 time passes. Comparisons between computed and experimental pressure distributions along the ramp surface, external cowl, and internal cowl are presented in Figures 41, 42, and 43, respectively.

The computed and measured ramp surface pressure distributions of Figure 41 show generally good qualitative agreement. The characteristics of the computed solution along the straight ramp sections is very similar to that of solutions I and Ia, and the same explanation given for the solution I results applies.

As stated, this case was chosen because the mass flow was near that at which the inlet choked. This is best seen by reference to Figure 18(b), which shows large reductions in pressure recovery in going from 0.59 to 0.62 capture-area ratio, and large variations in pressure recovery at the 0.62 value with changes in

tunnel Reynolds number. Thus the comparison of pressure distribution near the throat is of great interest. As seen in Figure 41, the computed solution did not yield as low C_p values in the throat as were measured. This is as expected, however, since at these near-sonic conditions the flow is very sensitive to the boundary-layer displacement effects, and the smaller effective flow area results in lower measured C_p values. In both the test run and the computed solution, supersonic flow is indicated by the throat C_p 's; a weak normal shock at nacelle station 10.0 is predicted in the computer solution but not evidenced, at least upstream of station 10.4, in the test data.

A comparison of computed and measured cowl external pressures is given in Figure 42. Good qualitative agreement is seen but, as in Case I, the predicted C_p 's are generally lower than those measured. This is especially true near nacelle station 10.0 and would result in an optimistic lip suction force prediction. A closer inspection of the cowl contour used in the finite-difference solution shows a very slight irregularity in the specified coordinates in this region; as expected, the inviscid solution is much more sensitive to such details than is the actual viscous flow. Better specification of the contour and more cells along the cowl would have yielded an improved solution, but machine storage capacity limitations prevented this.

The pressure distributions inside the cowl lip (Figure 43) do not agree as well as in the Case I solution. As stated previously, this is a very small region close to the lip, and the sharp lip assumed in the computed solution does limit accuracy in this region. It should be noted, however, that the change from large positive C_p 's in this region in Case I ($A_0/A_1 = 0.50$) to negative C_p 's in Case II ($A_0/A_1 = 0.62$) is predicted by the finite-difference solution.

CONCLUDING REMARKS

Accurate inlet spillage-drag data were obtained on a realistic two-dimensional, variable-geometry, external-compression, supersonic inlet configuration over a wide range of inlet compression-surface angles, mass flows, and subsonic and transonic flight speeds.

High inlet-capture-area ratios were obtained, enabling a credible extrapolation of the measured inlet drag to a capture-area ratio of 1.0.

Generally, the experimental data were consistent and provided the expected trend of decreasing spillage drag with increasing capture-area ratio and decreasing inlet throat area.

Large losses in inlet total-pressure recovery and increases in compressor-face distortion were observed at high inlet-throat Mach numbers. The choking inlet-throat Mach number observed, based on geometric throat area, was 0.80 or less. These data provide a basis for determining the tradeoff between inlet spillage drag and pressure recovery for practical design applications.

The computed flow-field solutions agreed reasonably well with the measured pressure distributions for the corresponding inlet flow conditions. It was established that finer meshes will result in yet more accurate solutions, but the desired refinements will require a computer having more core storage capacity than the CDC-6600 employed. The analytical technique shows good promise for the evaluation of inlet drag through use of a two-dimensional model, but further drag prediction comparisons are needed.

Since the computer code can handle subsonic, supersonic, and mixed flows, the subsonic solutions reported do not represent a complete utilization and checkout of its capabilities. Further evaluation of the full potential of the code should be made, preferably in a larger, faster-computing environment such as the CDC-7600 or, even more ideally, the NASA Ames Iliac 4. Other test cases from this experimental study provide an excellent basis for such an evaluation.

APPENDIX

DESCRIPTION OF THE FINITE-DIFFERENCE METHOD OF ANALYSIS AND THE COMPUTER PROGRAM

The computer procedure employed to provide the analytical solutions utilized in this study was formulated to solve numerically (using explicit finite-difference techniques) the equations describing the transient, two-dimensional or axisymmetric flow of an inviscid, compressible, perfect gas. The rationale employed in the development of the computer procedure was to provide a computational framework that could be used to treat a wide variety of problem types. To this end, a number of options are included, permitting, for example, selection of the particular differencing scheme to be used, control of the calculation process, and specification of the type and volume of output data. The versatility of the program is derived from (1) the availability of several finite-difference schemes, (2) the flexibility of the flow-field mesh arrangement, and (3) the variety of boundary conditions available.

Thus, while the finite-difference method of Godunov (Reference 3) was employed with this procedure to calculate inlet nacelle flow fields for the current effort, it has also been used with this procedure to calculate airfoil flow fields. Also, with this same computer procedure, other finite-difference methods (i.e., the MacCormack, Brailovskaya, and donor-cell methods - References 4, 5, and 6, respectively) have been used to calculate simpler flow fields. In addition, a closely related computer code with more limited geometric capabilities but with the viscous and heat-conduction terms of the Navier-Stokes equations has been used successfully to calculate subsonic, supersonic, and hypersonic attached boundary-layer flows (both laminar and turbulent) and to calculate laminar separated flows. The descriptions which follow are oriented toward the aspects of the analysis employed to generate the solutions utilized in this study.

Method of Analysis

The basic approach is to solve, using finite-difference techniques explicit in time, the equations that describe the transient flow of a compressible, inviscid, perfect gas. The desired steady-state flow field is a result of the solution

progressing asymptotically from an initial condition to the steady-state condition. The basic equations are applied as follows:

1. The flow field is divided into a finite number of discrete cells, with fluid properties assumed constant within each cell.
2. The governing equations are applied in integral form to each cell as a control volume to describe the time rate of change of mass, momentum, and energy within the cell in terms of the transport of these properties (fluxes) across the cell boundaries.
3. The fluid properties at the boundaries between cells are used to evaluate the fluxes. These properties are calculated by any one of several simple algorithms, which are described in detail below.
4. Appropriate boundary conditions are applied at the extremities of the overall flow-field control volume.

Governing equations. - Use is made of the continuity, momentum, and energy equations in integral form. These equations, in non-dimensionalized form, are as follows:

Continuity:

$$\frac{\partial}{\partial t} \int_{\text{c.v. (control volume)}} \rho \, dv = - \int_{\text{c.s. (control surface)}} \rho \vec{V} \cdot \vec{ds} \quad (1)$$

Momentum:

$$\frac{\partial}{\partial t} \int_{\text{c.v.}} \rho \vec{V} \cdot dv = - \int_{\text{c.s.}} \vec{V} (\rho \vec{V} \cdot \vec{ds}) - \int_{\text{c.s.}} p \vec{ds} \quad (2)$$

Energy:

$$\frac{\partial}{\partial t} \int_{c.v.} \rho E dv = - \int_{c.s.} (E + p/\rho) \rho \vec{V} \cdot \vec{ds} \quad (3)$$

where $E = \left(\frac{1}{\gamma-1} \right) \left(\frac{p}{\rho} \right) + \frac{\vec{V}^2}{2}$ and γ is the ratio of specific heats. ρ , \vec{V} , E , p , and t represent density, velocity, total energy, pressure, and time, respectively.

The quantities in these equations are dimensionless. The following list presents the reference quantities used to make them dimensionless.

<u>Dimensional quantity</u>	<u>Reference quantity</u>
Length	L, reference length
Velocity	U, freestream velocity
Time	U/L
Density	ρ_0 , freestream density
Pressure	$\rho_0 U^2$
Energy	U^2

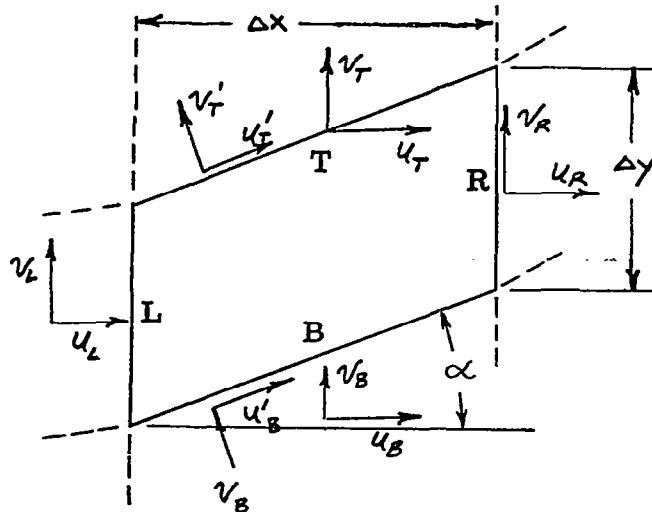
Formulation of Finite-Difference Equations. - Application of the continuity, momentum, and energy equations to an individual cell results in the basic finite-difference equations used in the analysis. For one-step techniques such as Godunov, these take the form

$$\phi_i^{t+\Delta t} = \phi_i^t + \frac{\Delta t}{V} \cdot F_i \quad (4)$$

where V is the cell volume and Δt is the computational time step. For two-dimensional flow, ϕ_i and F_i take the form shown below, where u and v are the horizontal and vertical components of \vec{V} and \vec{i}_x and \vec{i}_y are unit vectors in the horizontal and vertical directions.

i	ϕ_i	F_i
1	ϕ	$-\int_{c.s.} \rho \vec{V} \cdot d\vec{s}$
2	ϕu	$\left(-\int_{c.s.} V(\phi \vec{V} \cdot d\vec{s}) - \int_{c.s.} p d\vec{s} \right) \cdot \vec{i}_x$
3	ϕv	$\left(-\int_{c.s.} \vec{V}(\phi \vec{V} \cdot d\vec{s}) - \int_{c.s.} p d\vec{s} \right) \cdot \vec{i}_y$
4	ϕE	$-\int_{c.s.} (E + p/\rho) \rho \vec{V} \cdot d\vec{s}$

The entire flow field consists of cells which are arbitrary quadrilaterals; however, to illustrate the application of these equations, the expressions for F_i have been expanded for a simpler, parallelogram-shaped cell as follows:

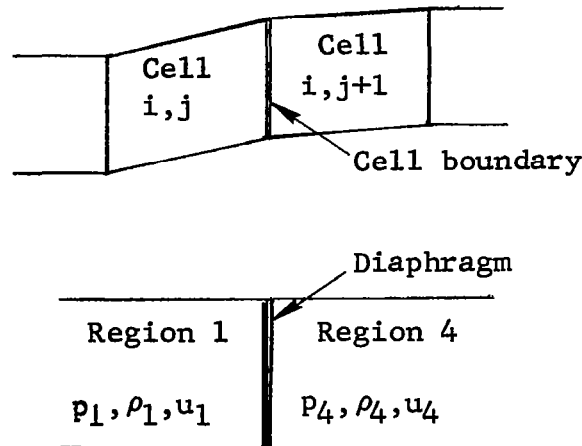


where u' and v' represent horizontal and normal velocity components at the inclined boundaries. The resulting form of the F_i are shown below.

i	ϕ_i	F_i
1	ρ	$\frac{(\rho u)_L - (\rho u)_R}{\Delta x} + \frac{(\rho v')_B - (\rho v')_T}{\Delta y \cos \alpha}$
2	ρu	$\begin{aligned} & \frac{(\rho u^2)_L - (\rho u^2)_R}{\Delta x} + \frac{(\rho v'u)_B - (\rho v'u)_T}{\Delta y \cos \alpha} \\ & + \frac{P_L - P_R}{\Delta x} + \frac{(P_T - P_B) \tan \alpha}{\Delta y} \end{aligned}$
3	ρv	$\begin{aligned} & \frac{(\rho uv)_L - (\rho uv)_R}{\Delta x} + \frac{(\rho v'v')_B - (\rho v'v')_T}{\Delta y \cos \alpha} \\ & + \frac{P_B - P_T}{\Delta y} \end{aligned}$
4	ρE	$\begin{aligned} & \frac{[\rho u(E + P/\rho)]_L - [\rho u(E + P/\rho)]_R}{\Delta x} \\ & + \frac{[\rho v'(E + P/\rho)]_B - [\rho v'(E + P/\rho)]_T}{\Delta y \cos \alpha} \end{aligned}$

Calculation of Fluid Properties at Cell Boundaries. - Equation 4 describes the fluid properties in the cells (it is applied once per cell per time step) at time $t + \Delta t$ in terms of the properties in the cells at time t and the transport of these properties across the cell boundaries. A number of methods for evaluating the cell boundary properties are proposed in the literature. Four of these techniques (References 3 through 6) that are compatible with the basic program structure (applying the conservation equations to arbitrary cells) have been incorporated into the computer program. The Godunov scheme has exhibited the best overall behavior, especially in terms of numerical stability, and consequently has been employed to generate the solutions in this study.

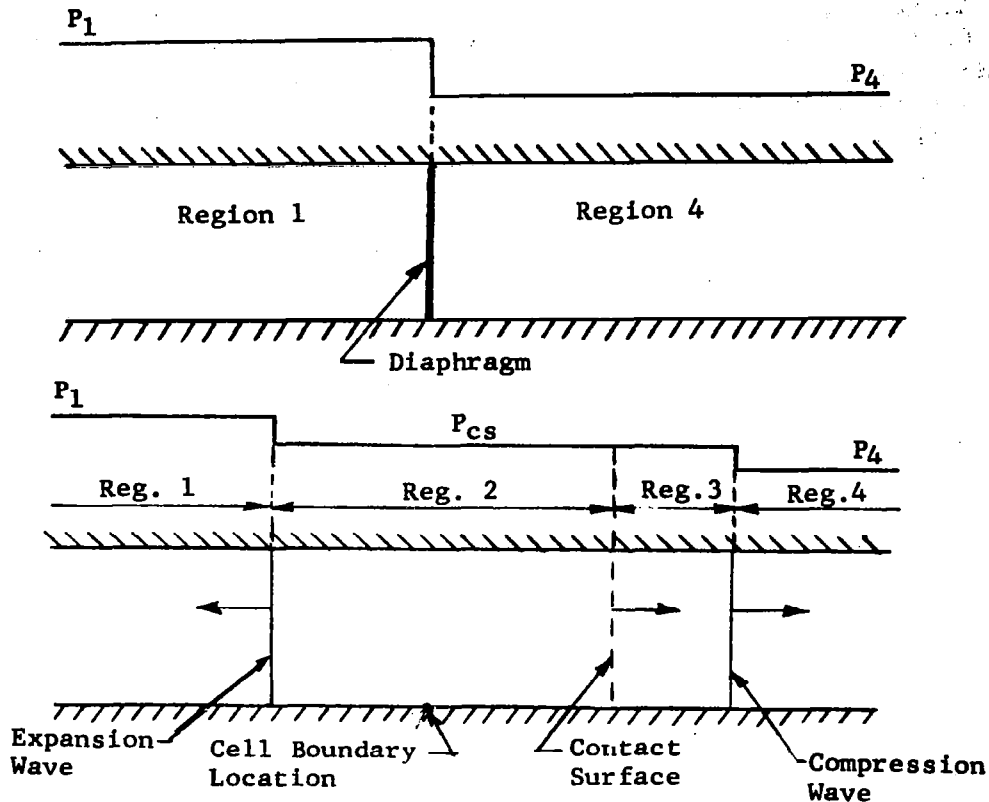
Basically, the Godunov method evaluates the fluid properties (velocity normal to cell boundary, p , and ρ) at a cell boundary by considering the adjacent cells and their common boundary to be analogous to the one-dimensional shock-tube problem as shown in the following sketch.



The location of the diaphragm of the shock tube represents the cell boundary. The properties on either side of the diaphragm are given by the components of the properties in the respective cells at time t . If the diaphragm is suddenly removed, a wave pattern is established in the tube that comprises a compression wave, an expansion wave, and a contact surface, as illustrated below.

The solution of the one-dimensional equations of fluid motion yields the fluid properties in each of the four regions. The fluid properties at the cell boundary are then defined by recognizing the region of flow that exists at the cell boundary.

Flow-Field Boundary Conditions. - Since the conservation equations governing compressible, time-dependent flows are parabolic, they require specified values for p , ρ , u , and v , or their derivatives, at every cell boundary on the flow-field control-volume boundary. Because steady-state solutions are being sought, steady-state boundary conditions are required. Proper specification of boundary conditions is essential if a physically valid solution is to be obtained; improperly specified boundary conditions can produce incompatibilities that may cause disturbances to propagate throughout the flow field and



even destroy the stability of the solution. Therefore, special attention must be given to the boundary conditions, and the optimum set of boundary conditions will vary from problem to problem and from one finite-difference technique to another.

Basically, three types of boundaries must be considered:

1. Inflow boundaries
2. Solid boundaries
3. Permeable boundaries.

Boundary conditions for each type of boundary are implemented in the computer programs by the equivalent of surrounding the flow-field control volume by a perimeter of imaginary (or "image") cells, and defining the required properties at the centers of these cells.

Specification of inflow boundary conditions for supersonic flows is straightforward. Since disturbances do not propagate upstream in a supersonic flow, the inflow boundary is not influenced by the downstream flow. The freestream properties are thus the values to be specified. For subsonic flows, however, disturbances from downstream can propagate upstream and influence conditions at the inflow boundary. It is therefore inappropriate to specify all of the flow properties at a subsonic inflow boundary. The technique employed is to set the velocity at the inflow boundary equal to the freestream velocity. The pressure is then determined by first-order extrapolation from the interior flow field. The density at the inflow boundary is then computed by requiring that the total enthalpy at the inflow boundary be equal to the freestream total enthalpy. In order to minimize the effects of errors introduced by this boundary condition, the inflow boundary is placed as far from the body as is feasible within the restraints imposed by the mesh size.

For a solid boundary, the only appropriate boundary condition is that the normal component of velocity be zero. Wall pressure is, however, specifically required by the computational process, and the calculation of wall pressure has proven to be very critical; different methods have been successful with particular finite-difference techniques. For the Godunov method, the shock-wave-analogy algorithm used between interior cells is applied. This calculation has been generalized to incorporate surface-curvature effects as suggested in Reference 7.

The correct values for the flow properties at the permeable boundaries (i.e., the downstream and lateral boundaries) are generally not known in advance. For supersonic outflow, the solution is insensitive to outflow boundary conditions as long as they do not introduce instabilities. For subsonic outflow, however, conditions must be imposed which, in some sense, least disturb the interior (upstream) flow. The approach taken is to compute properties at the permeable (outflow) boundaries by extrapolation from the interior flow field. As for inflow boundaries, the permeable boundaries are located sufficiently far from the body to minimize the effects of errors introduced by the selected boundary conditions. A specific exception is the subsonic duct downstream boundary in an inlet flow field. Here it is impossible to move the boundary to minimize the effects on the solution. In fact, sufficient conditions must be imposed to achieve the desired capture-area ratio. In subsonic flows (or supersonic flows that can be considered as isentropic), the boundary condition used with the most success

is one in which the axial velocity is fixed at the ideal value based on isentropic flow (assumed uniform at the outflow boundary) and the pressure and density are extrapolated (zeroth order). The velocity component parallel to the outflow boundary is set to zero. For supersonic flows (where a loss of total pressure is expected) the capture-area ratio, continuity equation, and total energy equation can be combined to yield the ideal values of pressure, density, and axial velocity which should exist if the flow were uniform at the outflow boundary. These values, along with a zero velocity parallel to the boundary, are imposed as the boundary conditions at each cell on the outflow boundary.

Flow-Field Initial Conditions. - Inasmuch as the asymptotic steady-state solution depends only on the boundary conditions imposed, the specification of initial conditions should affect only the time required to obtain a solution. (However, if initial conditions are exceptionally inconsistent with the boundary conditions, startup problems may result and preclude a steady-state solution.) Successful use has been made of impulsive initial conditions (i.e., starting from freestream conditions), input "best guess" initial conditions (based on intuition or prior knowledge), or the final solutions of previously calculated flow fields as initial conditions.

Computer Program Description

The computer program which implements the method of analysis described above is designated as General Dynamics' Convair Aerospace Division Procedure TP4. So that core storage requirements are reduced, the computer procedure is logically divided into three separate programs, designated TP4-I, TP4-II, and TP4-III. The TP4-I program performs flow-field-mesh generation and initial-condition setup; TP4-II performs the actual flow-field calculations; TP4-III performs additional computations (such as streamline locations) on the flow-field data when a final solution is achieved. Unless otherwise stated, the information which follows refers to the overall capabilities of the three separate TP4 programs.

The versatility and flexibility of the program are derived primarily from (1) the flexible mesh-patch structure, (2) user control of boundary conditions through input data, (3) comprehensive output data, and (4) special features that enhance the

efficiency and increase the utility of the program. These features combine to yield a program that is definitely "user-oriented." Control of the finite-difference technique employed is also available through input data, but because the solutions reported here are derived with the Godunov method, this aspect of the program will be discussed in no further detail. Also, the flow field is specified as either rectangular or axisymmetric by a single input option code.

The mesh-patch structure, boundary-condition options, special features, and input/output are discussed in the following paragraphs.

The Mesh-Patch Arrangement. - The development of a procedure for treating a wide variety of flow geometries requires a versatile mesh-geometry arrangement to treat complex body shapes. This capability is provided through the use of a mesh-patch arrangement. With this arrangement, complex mesh geometries are built up from simpler sub-meshes or "patches." The following simple example illustrates the use of the mesh-patch arrangement.

Consider the flow through a two-dimensional channel having a step as illustrated in Figure 44(a). A mesh arrangement consisting of three patches is shown in Figure 44(b). Each patch consists of a logically (but not necessarily geometrically) rectangular array of cells, i.e., each patch consists of M columns and N rows of quadrilateral cells. (M and N may be different for each patch except for certain restrictions on adjoining patches.) Thus, each patch has four boundaries, as illustrated in Figure 44(c).

The program allows for meshes comprising up to 20 patches. With this capability, mesh geometries can be constructed for quite complex flow fields. Sample mesh-patch arrangements for some types of problems are illustrated schematically in Figure 45.

One of the attractive features of the finite-difference scheme employed is the use of a non-orthogonal body-oriented mesh geometry. This feature facilitates the application of boundary conditions along arbitrary surfaces; however, it also presents the user with the frequently arduous task of defining the non-uniform mesh geometry. The mesh geometry is defined by specification of the coordinates of the corner points (or "nodes") of the quadrilateral cells. For a general non-orthogonal, non-uniform mesh, this often involves the specification of coordinates for several thousand points. The probability for errors

in generating, transcribing, and keypunching such a large number of coordinates is quite high.

One way to reduce the probability for errors is to reduce the amount of input geometry data required. This can be accomplished by generating the coordinates of all the interior nodes of a mesh patch with the program. This technique requires only the coordinates of node points on the four patch boundaries. Briefly, the interior node coordinates are generated for an arbitrary four-sided region (in a plane) from a set of known points along each of the four boundaries. The known boundary points are first mapped onto the corresponding boundaries of a unit square. Straight-line connection of these boundary points is then used to define interior node points in the unit square. The interior points within the unit square are then transformed back to the physical plane by use of a technique based on a surface-definition method of Coons (Reference 8).

With this geometry-generation method, only the points on the patch boundaries are required. For curved surfaces, these point coordinates must be furnished. For patch boundaries that are straight lines, however, an additional simplification of the input has been accomplished by a technique for subdividing a straight line into a monotonically increasing (or decreasing) set of intervals. Through this technique only a few input parameters are necessary to generate the point-coordinate data for a straight patch boundary.

A sample application of the mesh-geometry generation scheme is presented in Figure 46. The four boundaries of this sample mesh patch are defined as follows:

Boundary 1: The straight line from (-0.25, 0) to (-0.75, 0)

Boundary 2: The straight line from (0, 0.5) to (0, 1.2)

Boundary 3: The second quadrant of the ellipse

$$\left(\frac{x}{0.25}\right)^2 + \left(\frac{y}{0.5}\right)^2 = 1$$

Boundary 4: The second quadrant of the ellipse

$$\left(\frac{x}{0.75}\right)^2 + \left(\frac{y}{1.2}\right)^2 = 1$$

The coordinates for the points along Boundaries 3 and 4 were generated from the above relations and input directly. The points along Boundaries 1 and 2 were generated by use of the technique developed for straight patch boundaries. The coordinates of the interior points were generated by the mesh-geometry generation scheme.

The geometry plot in Figure 46 was generated by an SC-4020 computer recorder. Such computer-generated plots provide a rapid and graphic means of assessing the suitability of the computer-generated mesh geometry as well as of detecting geometry input errors.

The Boundary-Condition Options. - The mesh-patch arrangement described above requires the specification of boundary conditions for each patch boundary. So that maximum flexibility would be provided, the program was coded to permit the selection of any of several available boundary conditions for each of the four boundaries of each patch. In addition, it was necessary to provide for the calculation of the cell boundary fluxes on those boundaries common to two adjoining patches.

The calculation of the properties on a common boundary is accomplished on a cell-to-cell basis within the computer program. That is, each segment of the common patch boundary is shared by two (and only two) cells. Thus the two patches adjacent to the common boundary must have the same number of cells along that boundary, and the cell spacing along that boundary must be the same for each patch. Once the cells involved are identified, the cell boundary fluxes are computed by use of the same scheme used for interior cell boundaries.

The boundary condition applied at any patch boundary not shared with another patch can be selected via input option from any of the following types: For inflow boundaries, options for subsonic or supersonic inflow are available. For outflow boundaries, extrapolation techniques are generally employed, and both zeroth- and first-order (linear) extrapolation options are available. A special boundary condition is provided for inlet flow fields for use at the subsonic duct outflow boundary so that the inlet captures the desired mass flow. For solid surfaces, the pressure on the boundary may be determined by extrapolation from the flow field adjacent to the surface or may be computed by use of the Godunov scheme to determine the pressure required to cancel the normal component of momentum in the adjacent cell. For the latter case, surface curvature effects

may also be included. In addition, boundary-condition options for the axis of symmetry for axisymmetric flows and other special conditions are also available.

The optimum boundary-condition options may vary from problem to problem and from one finite-difference scheme to another. The program allows the user to select the boundary condition most appropriate for a given problem from an array of boundary-condition options. In addition, the program has been so coded that additional boundary-condition options may be incorporated in a straightforward manner.

Special Features. - Several special features have been included in the inviscid flow-field program to enhance its efficiency and increase its utility.

All of the data required to restart the solution may be written on magnetic tape at selected intervals during a computer run. This capability allows lengthy problems to be run as a series of lower-risk computer runs and, also, ensures that an entire run will not be lost because of an abnormal termination (e.g., exceeding the run-time estimate). This feature also permits the user to analyze intermediate results and identify potential problems before proceeding with the solution. When restarting with data from a previously generated magnetic tape, the user may alter any of the options controlling the computations (e.g., boundary-condition options).

The solution may be obtained as a true time-dependent phenomenon by the use of the same time step for each cell in the mesh or as a characteristic-time (non-uniform time step) solution wherein each cell is advanced by a time step based on its own local stability conditions. The latter approach conserves computer time when only a steady-state solution is required, especially when large variations in cell size are present.

Another tool to reduce computer time, a "cell-skipping" scheme, is available. Cells in which the properties are changing much less rapidly than in the most active portions of the flow field are intermittently omitted from calculational update. The skipping frequency is controlled by relative fluctuation sizes and by an input limit on the number of successive times an individual cell may be skipped.

If it is desired to begin a solution from a non-uniform initial flow field, the Mach number, local flow angle, total

pressure ratio (relative to the freestream value), and total temperature ratio may be input at corner cells of each patch (or sub-patch if further definition is desired) and the initial conditions at each cell will be computed by interpolation from these points. This feature is utilized to reduce computation time or to minimize the possibility of severe startup transients causing instabilities.

A streamline-tracing routine for the generation of coordinates of points on selected streamlines has been developed for nacelle-type flow fields.

Input/Output. - The flow-field program input and output information is summarized as follows:

Input data:

- Flow-field control volume and mesh arrangement.
- Freestream conditions.
- Options controlling mesh-patch boundary conditions.
- Initial conditions for the flow field.
- Options controlling output, use of special features, convergence criteria, etc.

Output data:

- Fluid pressure, density, velocity components, Mach number, velocity vector direction and magnitude, and total enthalpy for each cell in the mesh at selected intervals.
- Similar data for selected cells for each time pass.
- Fluid pressure, density, velocity components, Mach number, and pressure coefficient for selected patch boundaries, e.g., solid surfaces.
- Printed data for pressure, density, and/or Mach-number contours.
- Printed data for streamline traces.

- 1953

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TABLE I. - AS-RUN TEST PROGRAM -
1/8-SCALE INLET DRAG MODEL (A-FAA/GD) TEST NO. 66-695

Angle of Attack = 0°

Run No.	M ₀	Reynolds No./ft	Config. Code	A ₀ /A _i Schedule
1				
2				
3				
4				
5				
		DATA NOT APPLICABLE		
6	1.40	2.5 x 10 ⁶	1	.77, .74, .68, .63, .58, .53
7	1.20	2.5	1	.72, .68, .62, .57, .53, .48
8	0.88	2.5	1	.71, .67, .62, .56, .52, .46
9	0.85	2.5	1	.71, .66, .60, .54, .49, .45
10	0.70	2.5	1	.76, .69, .65, .59, .54, .50
11	0.55	2.5	1	.87, .84, .79, .75, .70, .59
12	1.40	1.5	2	.63, .46
13	1.40	2.5	2	.62, .58, .53, .49, .45, .44
14	1.40	3.5	2	.43, .61
15	1.20	2.5	2	.58, .53, .48, .44, .41
16	0.88	2.5	2	.57, .54, .50, .46, .41
17	0.85	2.5	2	.58, .53, .49, .45, .41
18	0.70	1.5	2	.62, .45
19	0.70	2.5	2	.62, .59, .54, .49, .44
20	0.70	3.5	2	.43, .62
21	0.55	2.5	2	.71, .65, .60, .54, .51
22	--	--	Data Not Valid	--
23	1.40	2.5	4	.49, .47, .45, .42
24	1.20	2.5	4	.46, .43, .40
25	0.88	2.5	4	.46, .43, .40
26	0.85	2.5	4	.46, .45, .42, .40
27	0.70	2.5	4	.50, .46, .43
28	0.55	2.5	4	.57, .55, .53, .51, .50
29	1.40	2.5	5	.86, .81, .77, .71, .66, .59, .59
30	1.20	2.5	5	.83, .78, .76, .71, .66, .60
31	0.88	2.5	5	.83, .81, .78, .73, .68, .62
32	0.85	2.5	5	.83, .79, .73, .69, .63, .59
33	0.70	2.5	5	.89, .86, .82, .76, .71, .65
34	0.55	2.5	5	1.02, 1.00, .94, .89, .83, .76

TABLE II. - SUMMARY OF FINITE-DIFFERENCE
COMPARISON CASES

Item	Case		
	I	Ia	II
Nominal Conditions			
Ramp configuration	2	2	2
Nominal Mach number	0.7	0.7	0.7
Nominal capture-area ratio	0.50	0.50	0.62
Figure (No.) showing results	30-33	37-40	41-43
Test Conditions			
Run numbers	19	19	18,19*,20
Mach number	0.700	0.700	0.705,0.701*, 0.699
Capture-area ratio	0.491	0.491	0.621,0.618*, 0.621
Reynolds number/ft x 10 ⁻⁶	2.64	2.64	1.55,2.64*,3.67
Computer Solution Conditions			
Mach number	0.707	0.707	0.700
Capture-area ratio	0.493	0.493	0.62
Figure (No.) showing mesh	27-29	34-36	34-36
Number of cells	1785	1785	1785
Initial conditions	Freestream	Case I Solution	Case Ia Solution
Number of time passes	3200	550	1950
Computer time, hr	5.0	0.6	2.0

*Conditions selected for comparison in Case II

TABLE III. - COMPARISON OF MEASURED AND
COMPUTED DRAGS FOR CASE I

Drag Item	Drag Coefficient, C_D	
	Experimental	Computed
Ramp	0.150	0.208
Additive	0.155	0.213
Lower Cowl	-0.033	-0.115
Lower Cowl + Additive	0.123	0.098
Computed Skin Friction	--	0.088
Total Drag	0.169	0.186

Side elevation diagram of the test section. The diagram shows a horizontal cross-section of a wind tunnel or similar facility. Key components and dimensions are labeled:

- Test section (Perforated floor & ceiling)**: The main horizontal section.
- Observation window**: A circular window on the left side, with a diameter of 46" D.
- Schlieren window**: A circular window on the right side, with a diameter of 46" D.
- Sliding nozzle block**: A hatched rectangular block at the bottom right.
- Sting body**: A central vertical structure with a diameter of 4" and a length of 22' 36".
- Rear push rod** and **Forward push rod**: Two rods at the bottom right, with a distance of 88" between them.
- Dimensions**:
 - Horizontal dimensions: 14'-5", 8"-11", 33", 33", 47", 36.072", 36", 69", 88".
 - Vertical dimensions: 36", 36", 36".
 - Angles: 18°, 13°.

Figure 1.- Ames 6- by 6-Foot Supersonic Wind Tunnel test section and model-support system details.

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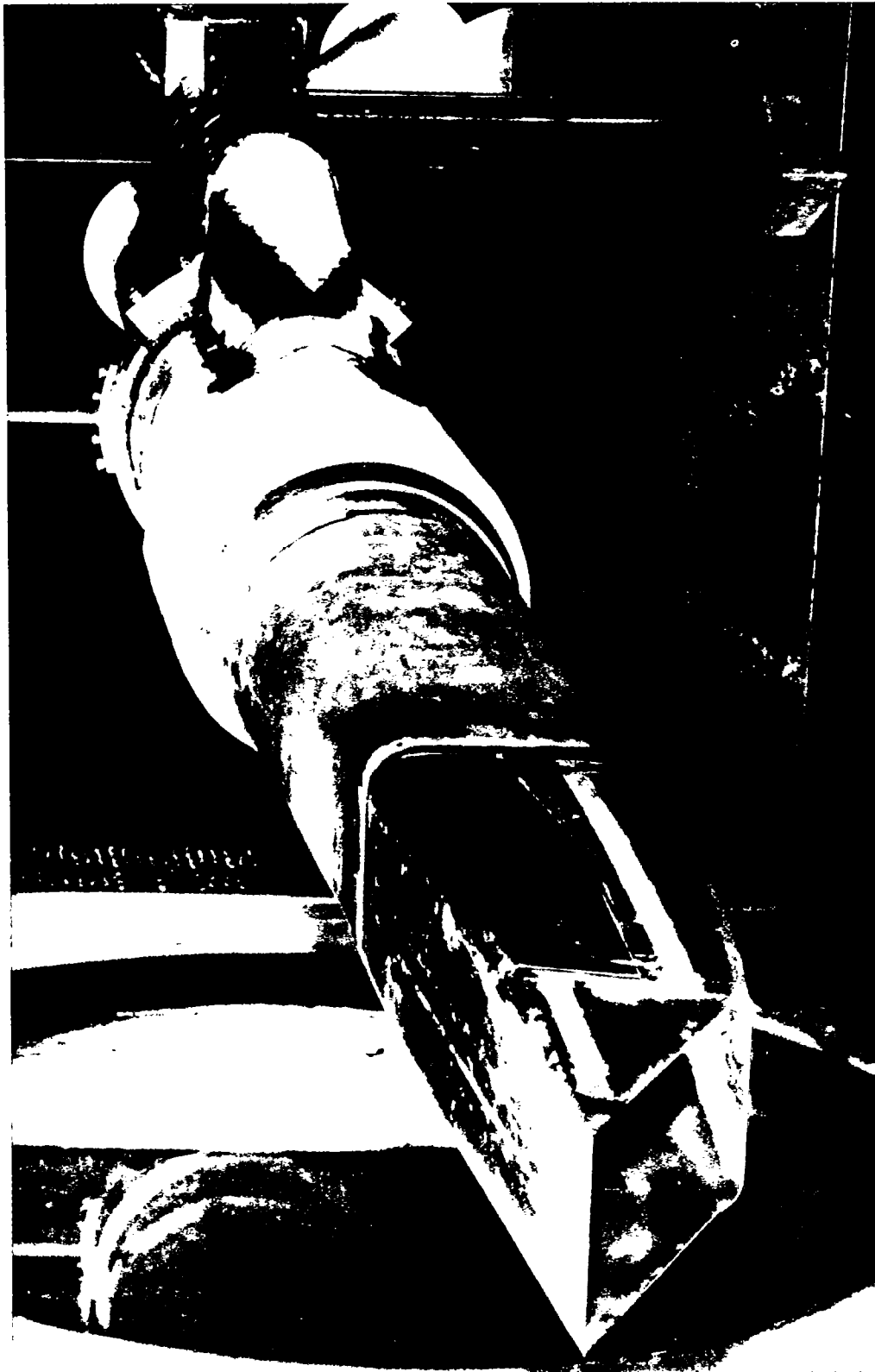


Figure 2.- Inlet drag model (Config. 5) mounted in the Ames
6x6-Foot Supersonic Wind Tunnel.



Figure 3.- Vacuum line from model to tunnel evacuation system.

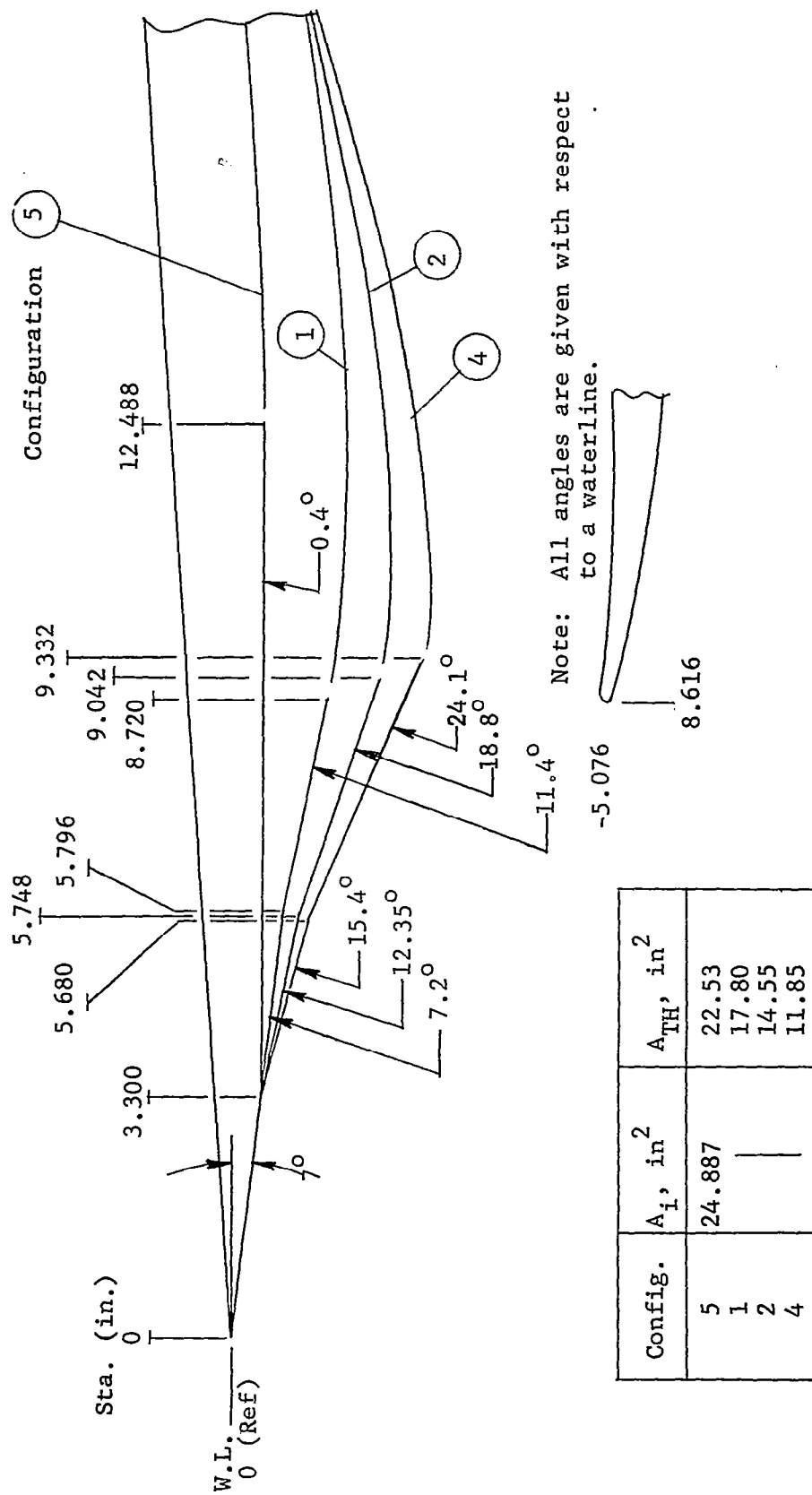
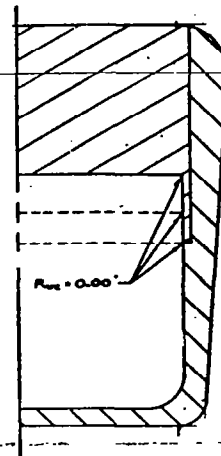
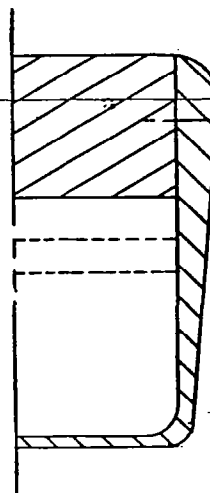
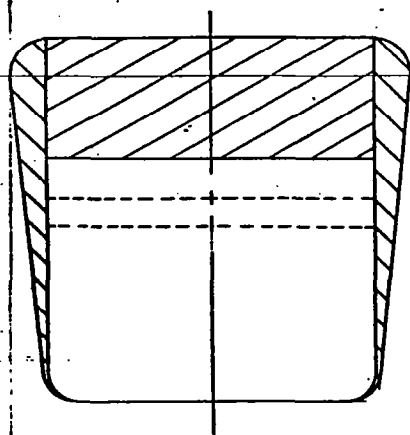
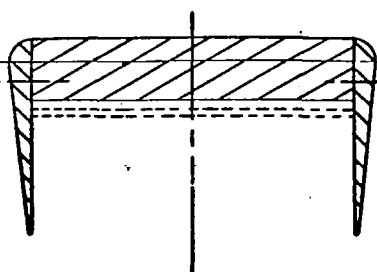


Figure 4.- Ramp geometry of inlets.

O REF. WL FOR SECTIONS

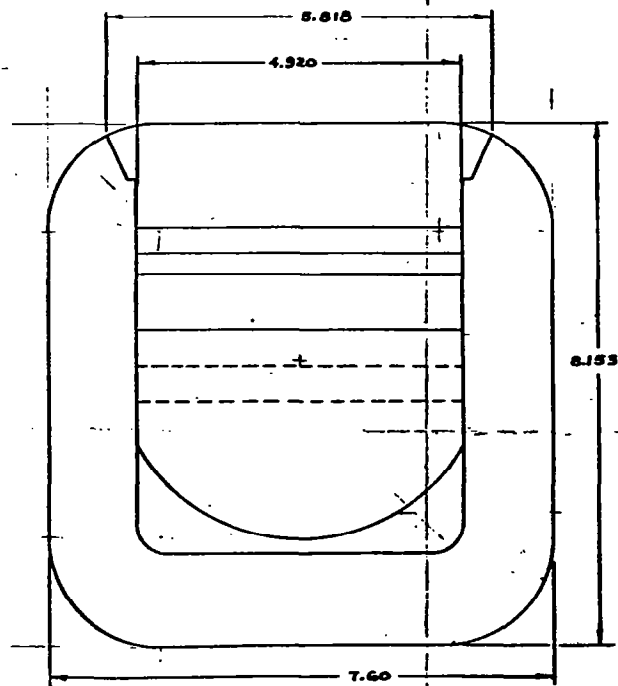


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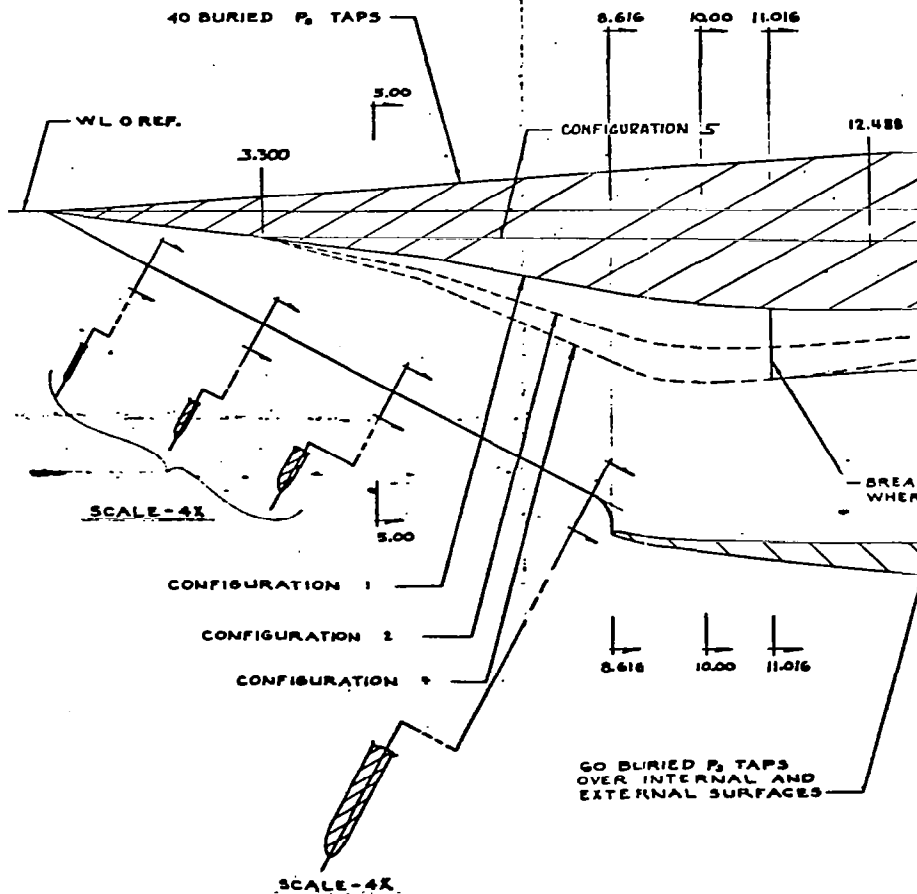
10.00

11.016
(R_{min} BEGINS)



FRONT VIEW

40 BURIED P₂ TAPS



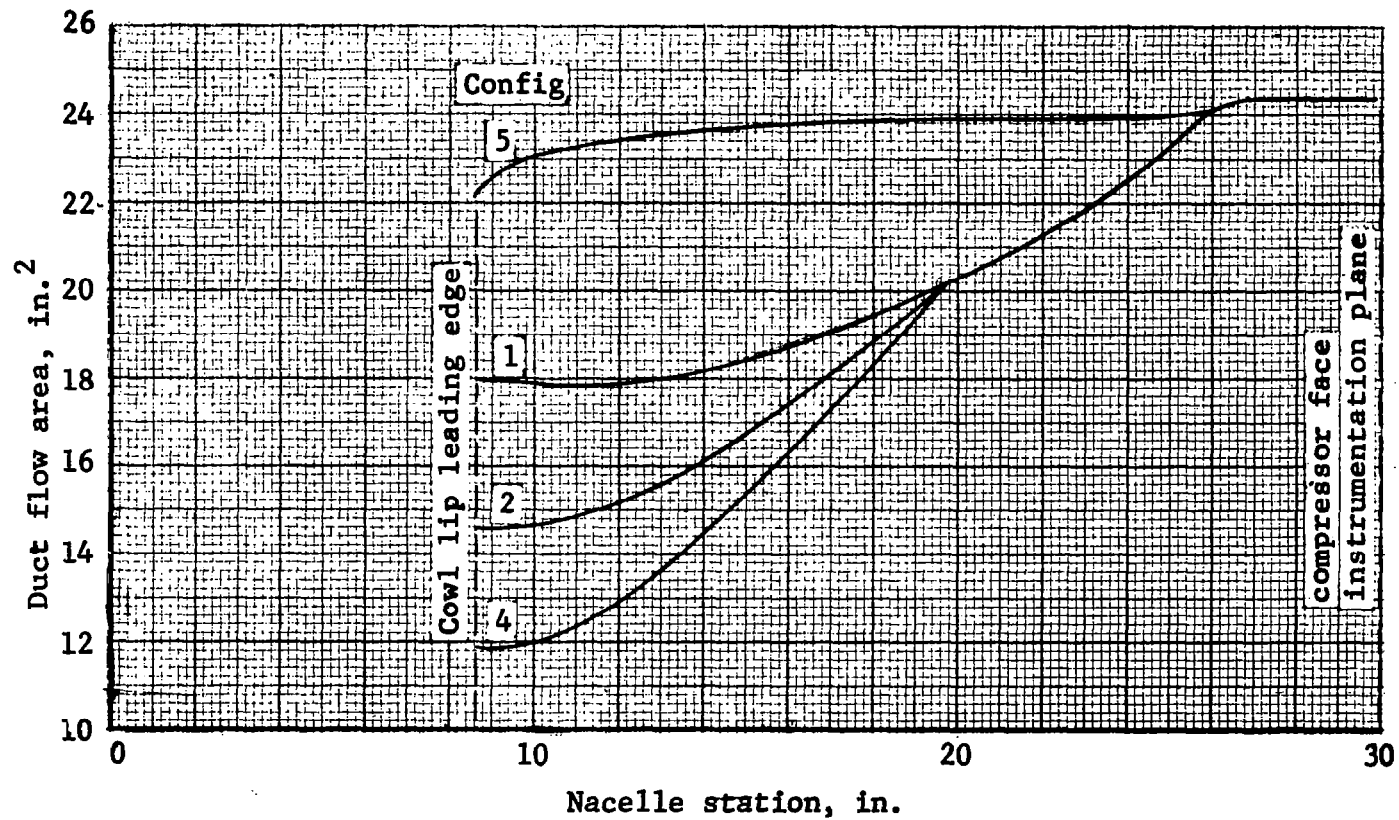


Figure 6.- Model duct flow area distribution.

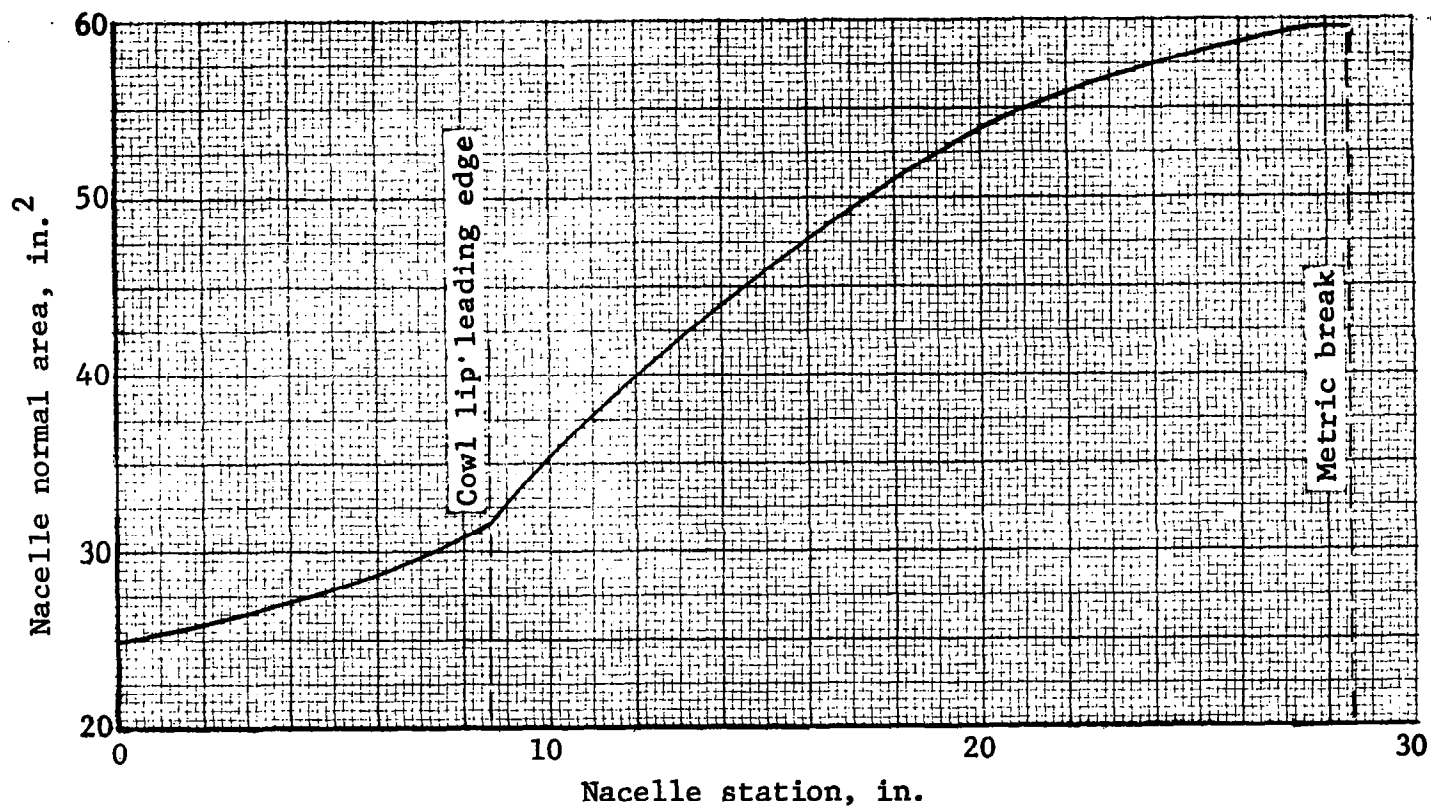
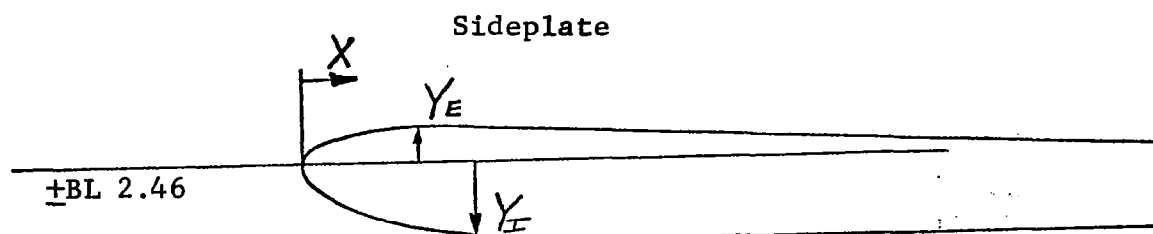


Figure 7.- Model nacelle normal area distribution.



<u>X (in.)</u>	<u>Y_E (in.)</u>	<u>Y_I (in.)</u>
0	0	0
.0030	.0056	.0095
.0060	.0078	.0137
.0120	.0110	.0196
.0240	.0157	.0275
.0600	.0226	.0415
.0864	.0240	.0474
.1200	.0239	.0518
.1800	—	.0541
.2592	—	.0560

Cowl

<u>Station</u>	<u>WL - Internal</u>	<u>Station</u>	<u>WL - External</u>
8.616	-5.0760	8.616	-5.0760
8.621	-5.0674	8.620	-5.0849
8.632	-5.0596	8.630	-5.0934
8.646	-5.0546	8.641	-5.0998
8.658	-5.0518	8.658	-5.1074
8.711	-5.0492	8.688	-5.1192
8.803	-5.0638	8.810	-5.1621
8.832	-5.0700	8.935	-5.1982

Figure 8 Cowl and Sideplate leading-edge geometry

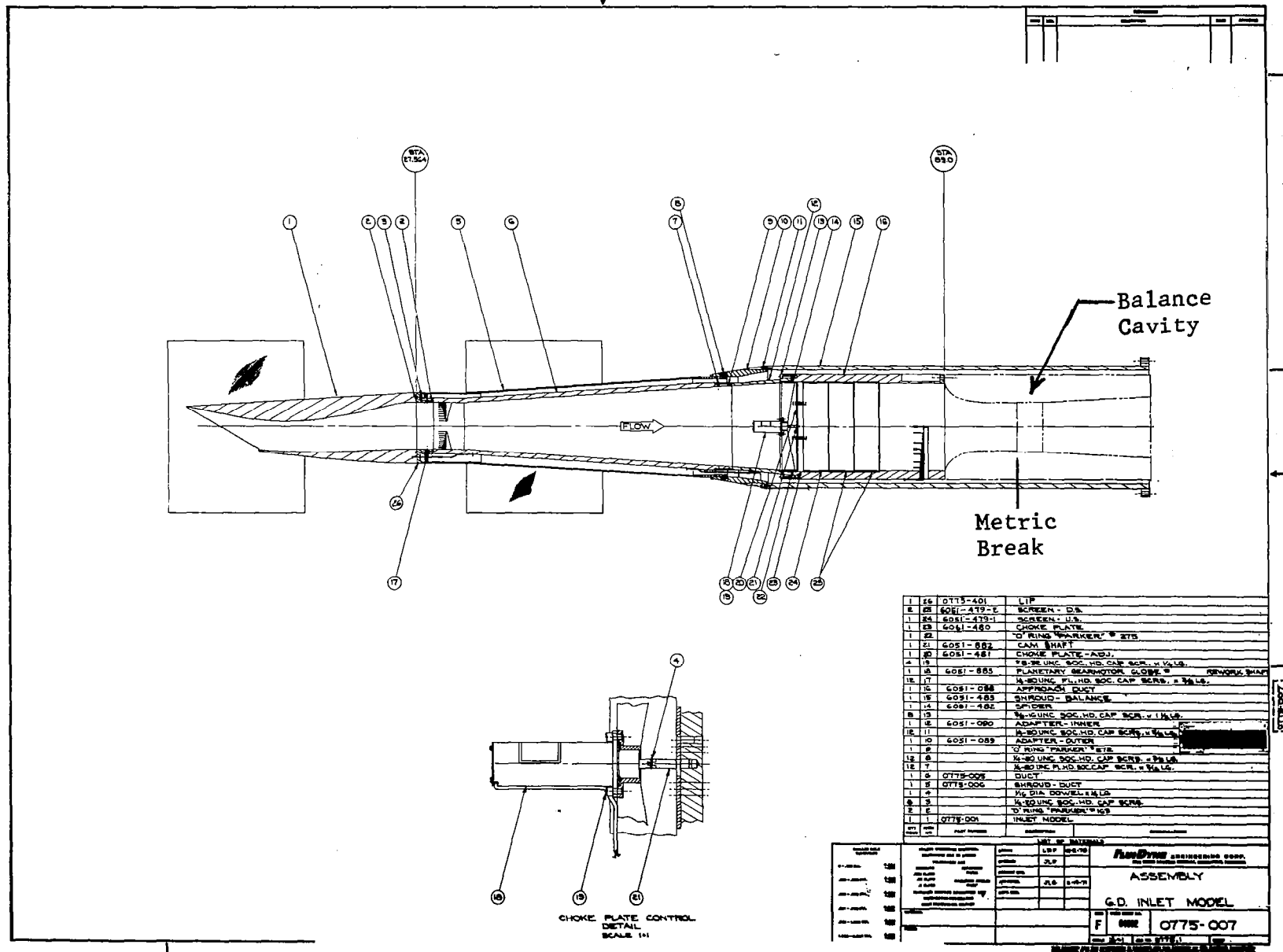
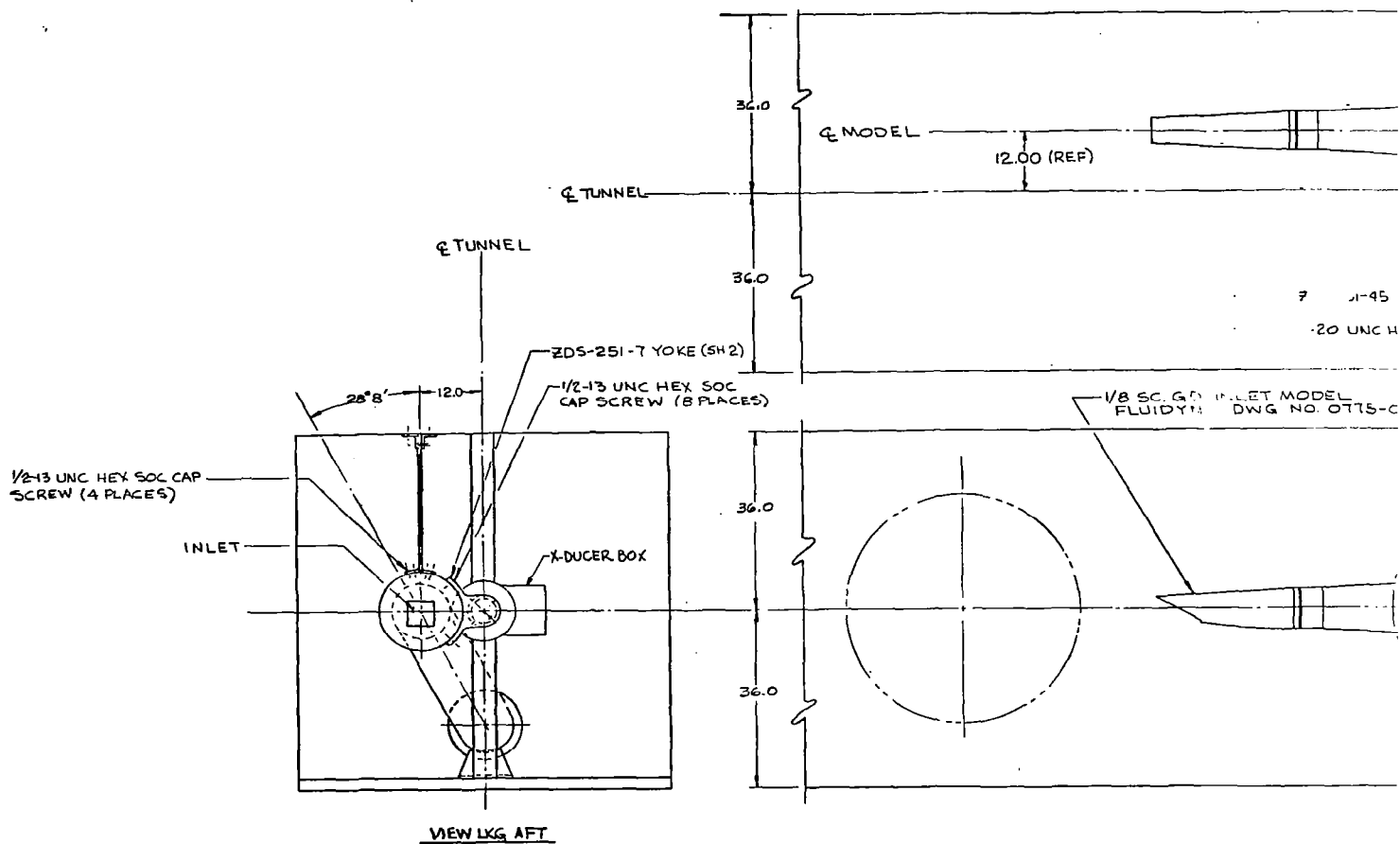


Figure 9.- Model and force balance assembly.



PARTIAL INSTALLATION DRAW
MODEL AND BALANCE IN AMES

CONFIG. 5 INSTRUMENTATION

INTERNAL TAP LOCATIONS

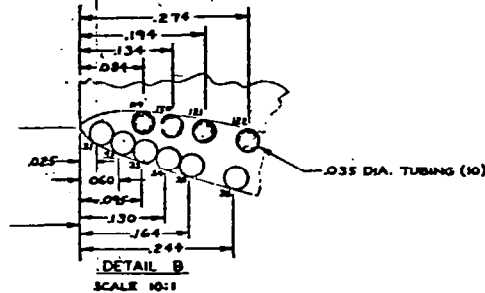
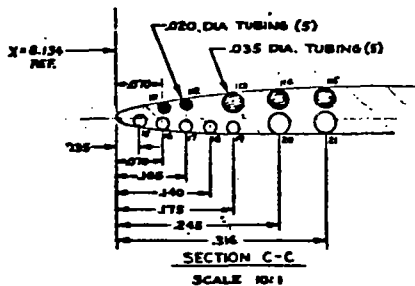
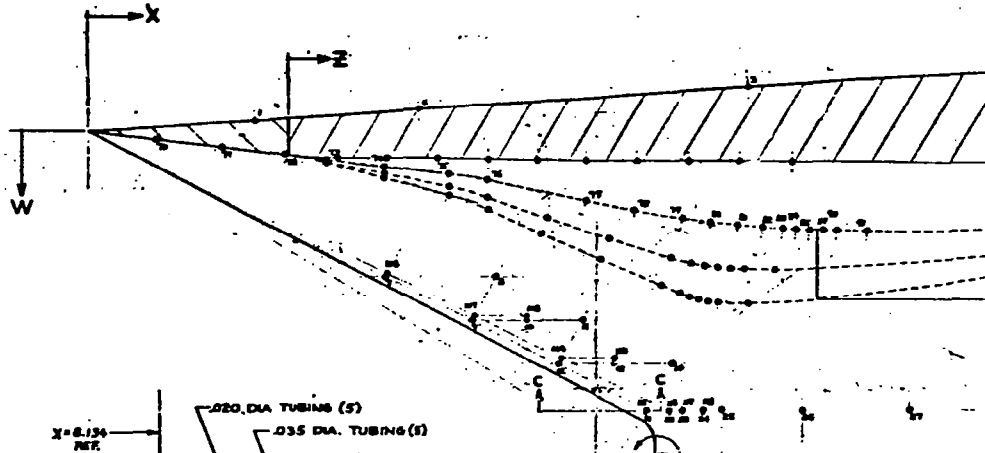
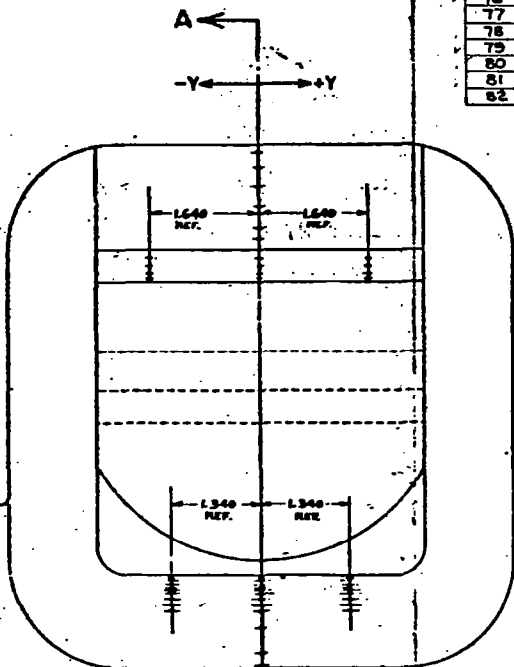
TAP NO.	X	Z	Y
73	3.753	7.63	0
74	4.515	11.525	0
75	5.278	2.288	0
76	6.040	3.050	0
77	6.803	3.813	0
78	7.566	4.576	0
79	8.328	5.338	0
80	9.091	6.101	0
81	9.853	6.863	0
82	10.616	7.626	0

EXTERNAL TAP LOCATIONS

TAP NO.	X	Z	Y
2	5.000	2.01	0
3	10.000	7.01	0
4	15.000	12.01	0
5	20.000	17.01	0
6	25.000	22.01	0

NOTES:

1. 0 DENOTES INTERNAL TAP
2. USE .035 DIA. TUBING UNLESS SIZED TUBE IS USED JUMPER
3. TAP NUMBER 69 NOT USED.



COML INSTRUMENTATION

EXTERNAL TAP LOCATIONS

TAP NO.	X	W	Y
1	2.50	0	0
7	4.47	2.204	0
8	6.17	2.294	0
9	5.79	2.182	0
10	6.64	2.182	0
11	7.49	2.979	0
12	7.12	3.676	0
13	7.97	3.676	0
14	8.80	3.676	0
15	8.17	4.367	0
16	8.19	4.367	0
17	8.23	4.360	0
18	8.28	4.362	0
19	8.31	4.350	0
20	8.38	4.373	0
21	8.45	4.385	0
22	8.49	4.381	0
23	8.44	4.341	0
24	9.24	4.380	0
25	9.61	4.364	0
26	10.81	4.367	0
27	12.41	4.364	0
28	15.00	4.380	0
29	19.49	4.386	0
30	24.99	4.380	0
31	8.64	0	0
32	8.68	0	0
33	8.72	0	0
34	8.75	0	0
35	8.79	0	0
36	8.83	0	0
37	9.28	0	0
38	9.36	0	0
39	9.60	0	0
40	10.11	0	0
41	10.81	0	0
42	11.61	0	0
43	12.41	0	0
44	13.70	0	0
45	14.99	0	0
46	17.50	0	0
47	20.00	0	0
48	25.00	0	0
49	8.64	-1.340	0
50	8.75	0	0
51	8.86	0	0

TAP NO.	X	W	Y
52	9.11	0	0
53	9.36	0	0
54	9.67	0	0
55	10.11	0	0
56	10.81	0	0
57	11.62	0	0
58	12.42	-1.340	0
59	8.67	+1.340	0
60	8.75	0	0
61	8.86	0	0
62	9.12	0	0
63	9.36	0	0
64	9.63	0	0
65	10.10	0	0
66	10.81	0	0
67	11.61	0	0
68	12.41	+1.340	0

INTERNAL TAP LOCATIONS

TAP NO.	X	W	Y
70	1.00	0	0
71	2.00	0	0
72	2.95	0	0
86	1.00	-1.640	0
87	1.99	-1.640	0
88	2.94	-1.640	0
96	1.00	+1.640	0
97	2.00	+1.640	0
98	2.94	+1.640	0
106	4.50	2.230	0
107	5.84	2.912	0
108	6.64	2.910	0
109	7.18	3.583	0
110	7.98	3.581	0
111	8.21	4.378	0
112	8.25	4.378	0
113	8.32	4.381	0
114	8.39	4.378	0
115	8.46	4.378	0
116	8.59	4.382	0
117	8.64	4.371	0
118	8.74	4.378	0
119	8.70	0	0
120	8.75	0	0
121	8.81	0	0
122	8.90	0	0

CONFIG. 1 INSTRUMENTATION

INTERNAL TAP LOCATIONS

TAP NO.	X	Z	Y
73	3.58	.59	0
74	4.43	1.46	0
75	5.45	2.46	0
76	6.05	3.06	0
77	7.56	4.57	0
78	8.29	5.30	0
79	9.02	6.03	0
80	9.41	6.42	0
81	9.80	6.81	0
82	10.20	7.21	0
83	10.70	7.61	0
84	10.70	7.71	0
85	11.10	8.11	0
90	11.30	8.31	0
91	11.74	8.75	0
92	9.58	.59	-1.640
93	9.45	1.46	0
94	5.45	2.46	0
95	6.05	3.06	0
99	7.36	4.57	0
100	9.02	6.03	0
101	10.20	7.21	-1.640
102	3.58	.59	+1.640
103	4.45	1.46	0
104	5.45	2.46	0
105	6.05	3.06	0
123	7.36	4.57	0
124	9.02	6.03	0
125	10.20	7.21	+1.640

EXTERNAL TAP LOCATIONS

TAP NO.	X	Z	Y
2	5.00	2.01	0
3	10.00	7.01	0
4	15.00	12.01	0
5	20.00	17.01	0
6	25.00	22.01	0

CONFIG. 2 INSTRUMENTATION

INTERNAL TAP LOCATIONS

TAP NO.	X	Z	Y
73	3.58	.59	0
74	4.45	1.46	0
75	5.45	2.46	0
76	6.05	3.06	0
77	6.98	3.99	0
78	7.91	4.92	0
79	8.85	5.86	0
80	9.13	6.14	0
84	9.33	6.34	0
85	9.53	6.54	0
89	9.73	6.74	0
90	9.93	6.94	0
91	10.38	7.39	0
92	3.58	.59	-1.640
93	4.45	1.46	0
94	5.45	2.46	0
95	6.05	3.06	0
99	6.98	3.99	0
100	7.92	4.92	0
101	8.85	5.86	-1.640
102	3.58	.59	+1.640
103	4.45	1.46	0
104	5.45	2.46	0
105	6.05	3.06	0
123	6.98	3.99	0
124	7.91	4.92	0
125	8.85	5.86	+1.640

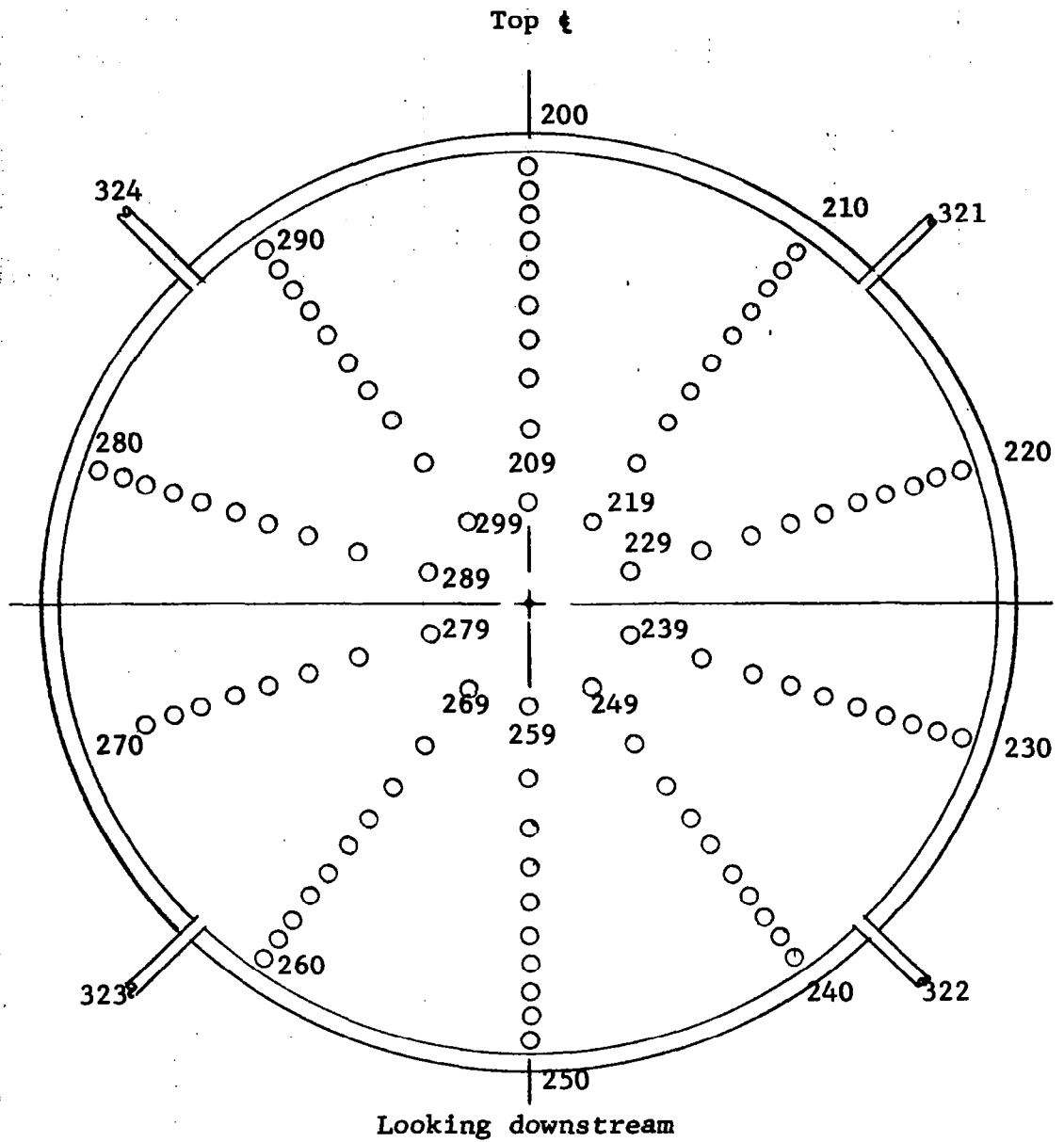


Figure 12.- Instrumentation at compressor face station.

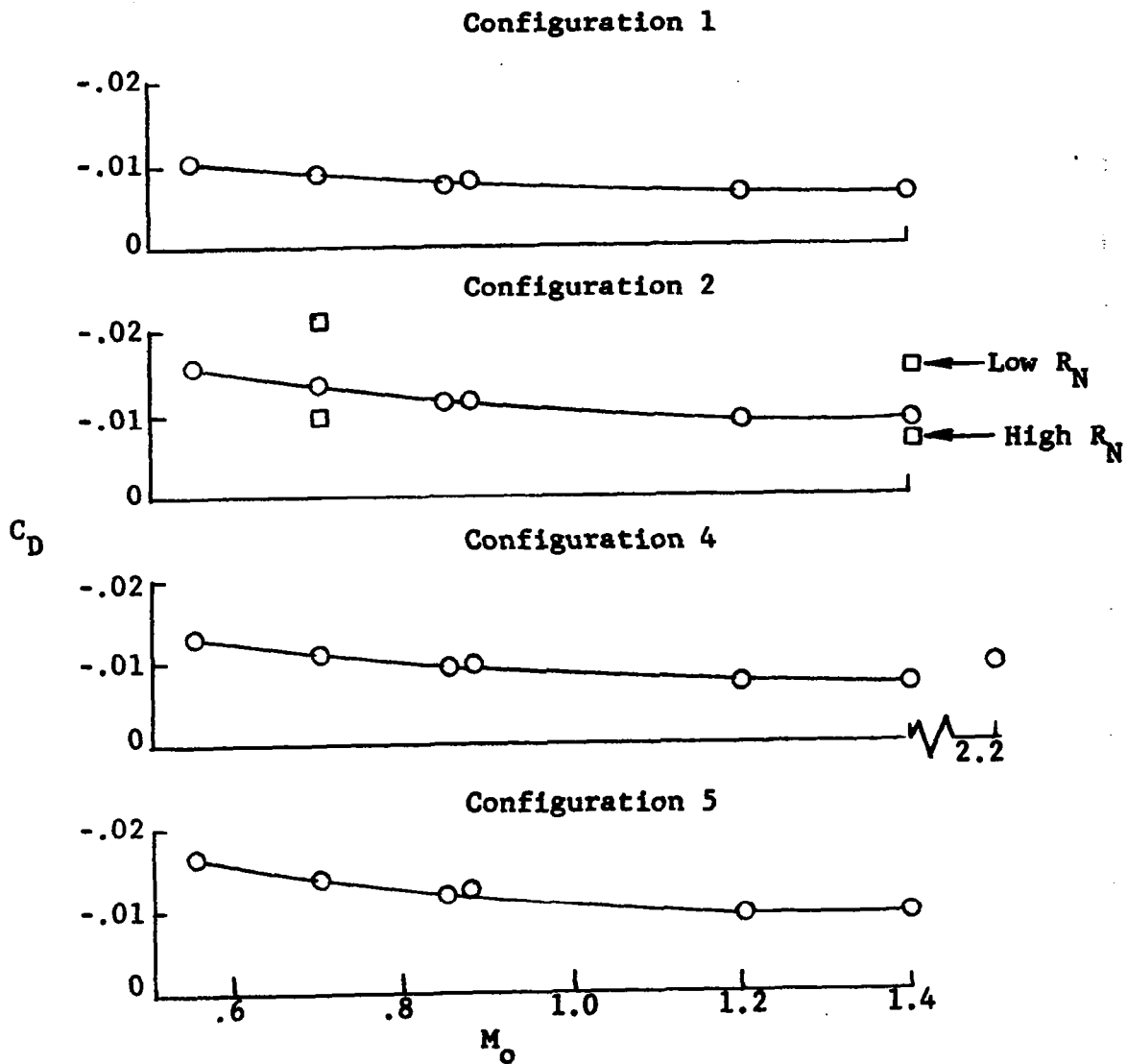


Figure 13. - Zero-shift corrections to drag coefficient.

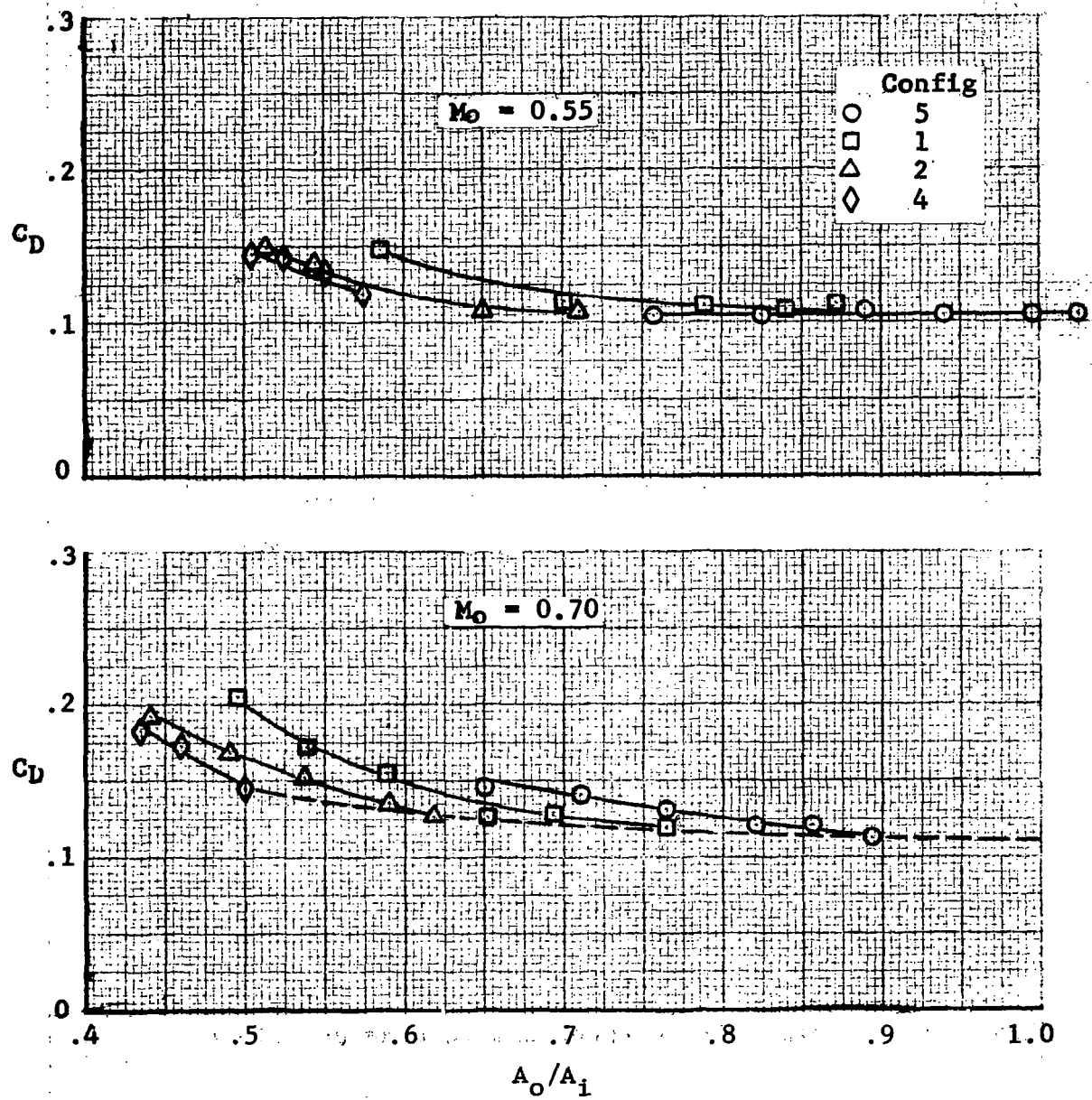


Figure 14.- Total inlet drag.

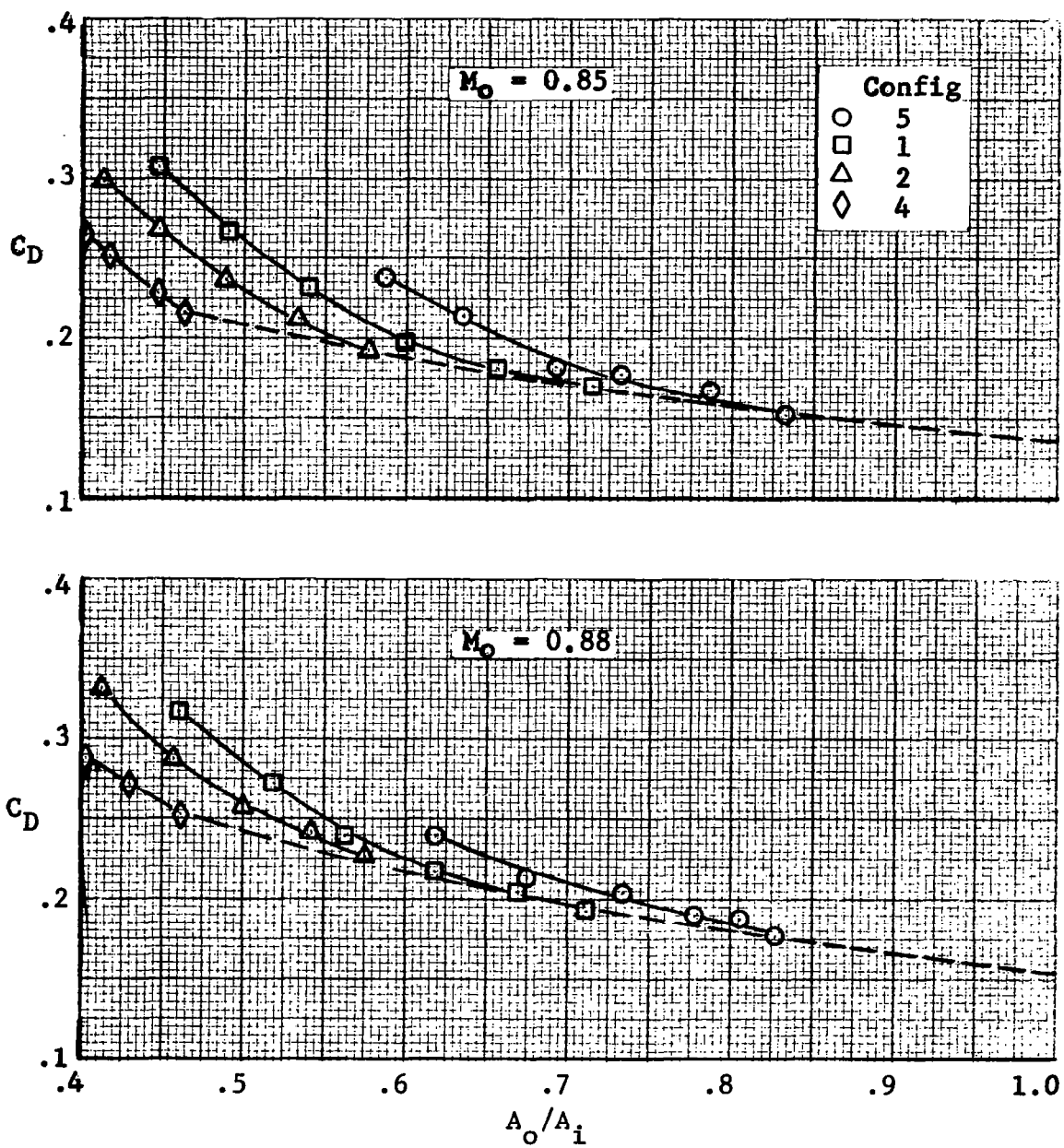


Figure 14.- Continued.

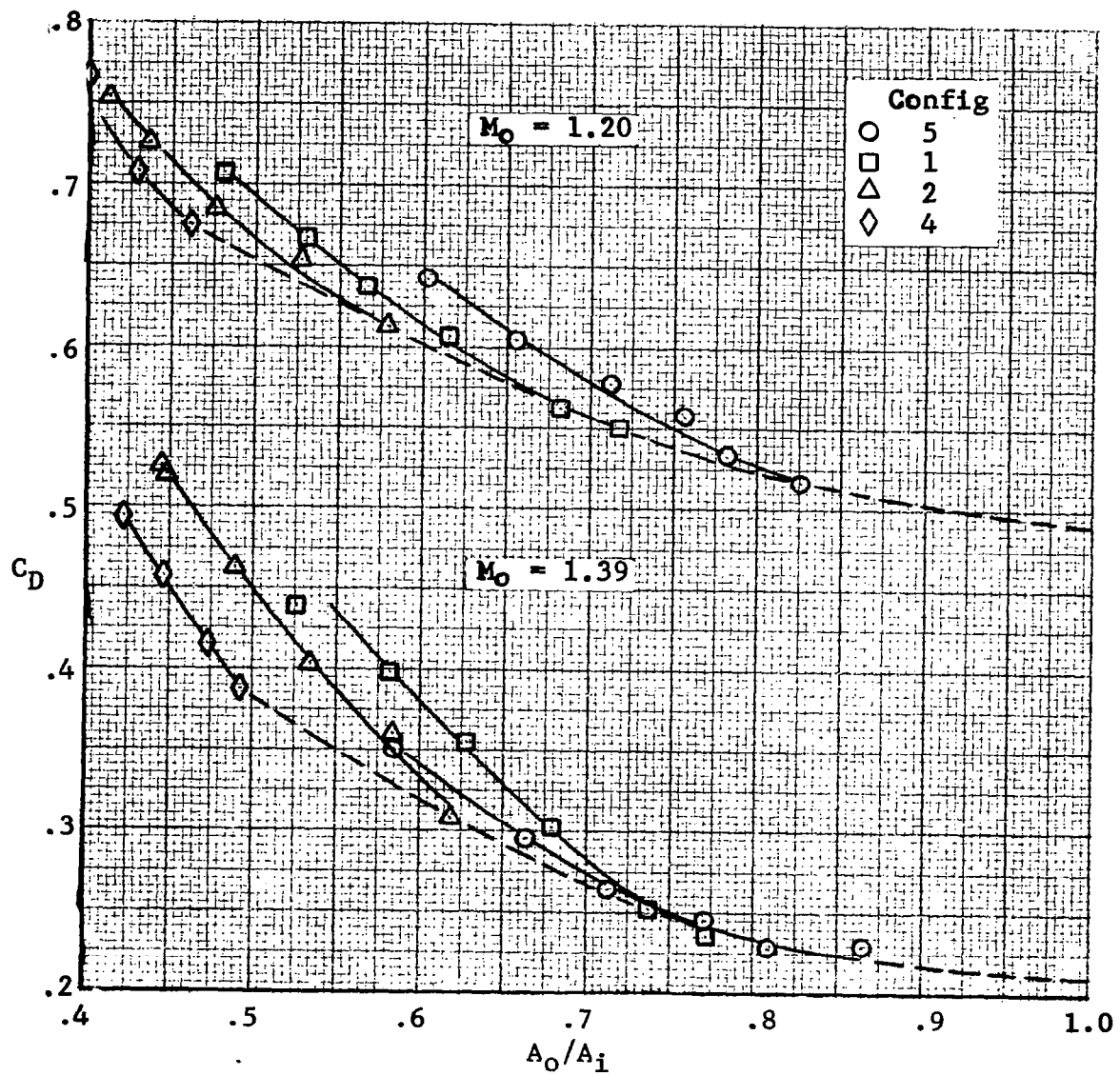


Figure 14.- Concluded.

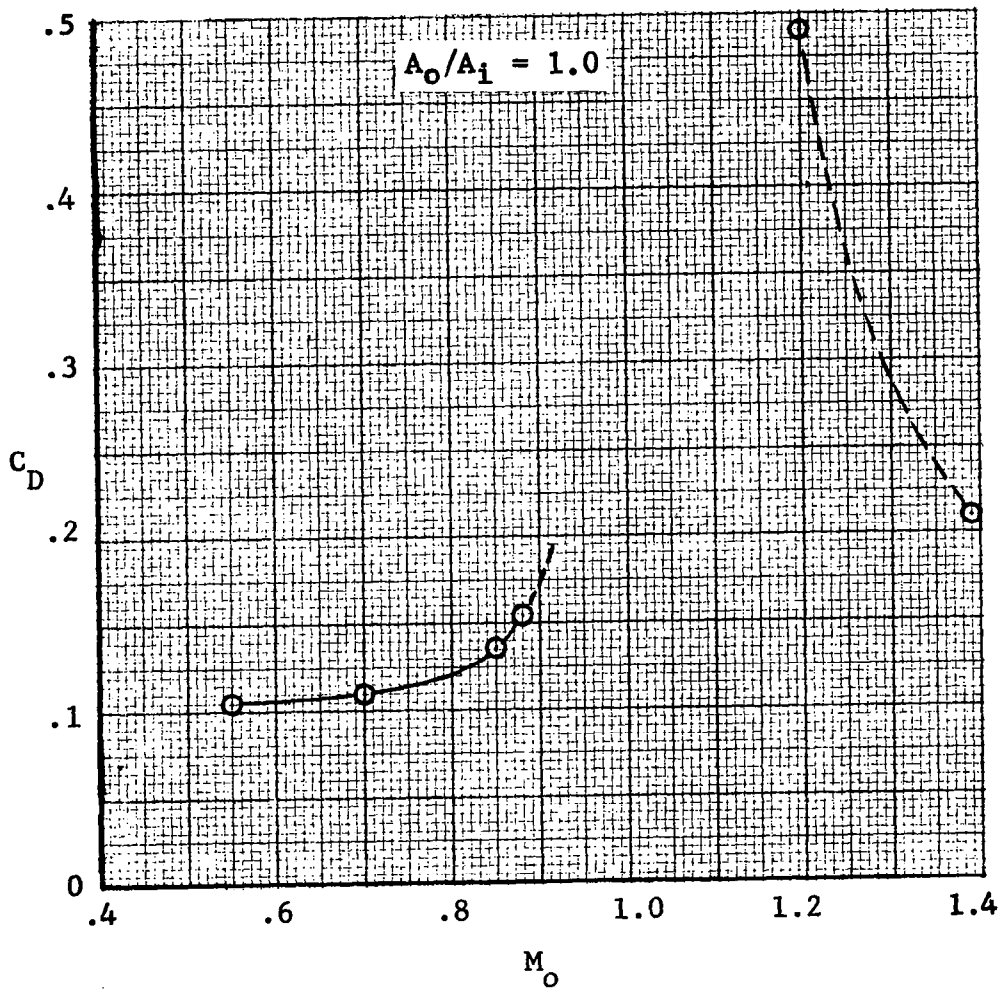


Figure 15.- Effect of Mach number on total inlet drag at $A_o/A_i = 1.0$.

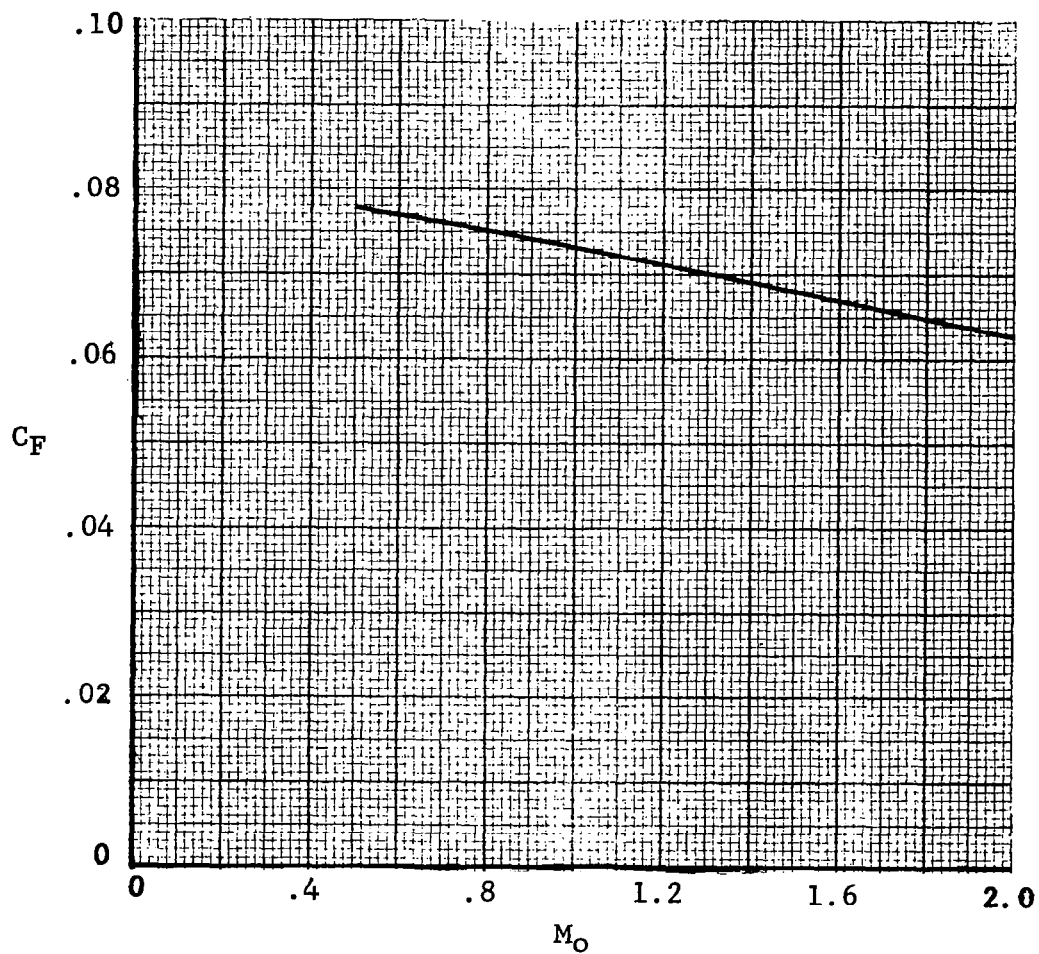


Figure 16. - Estimated cowl skin friction drag coefficient.

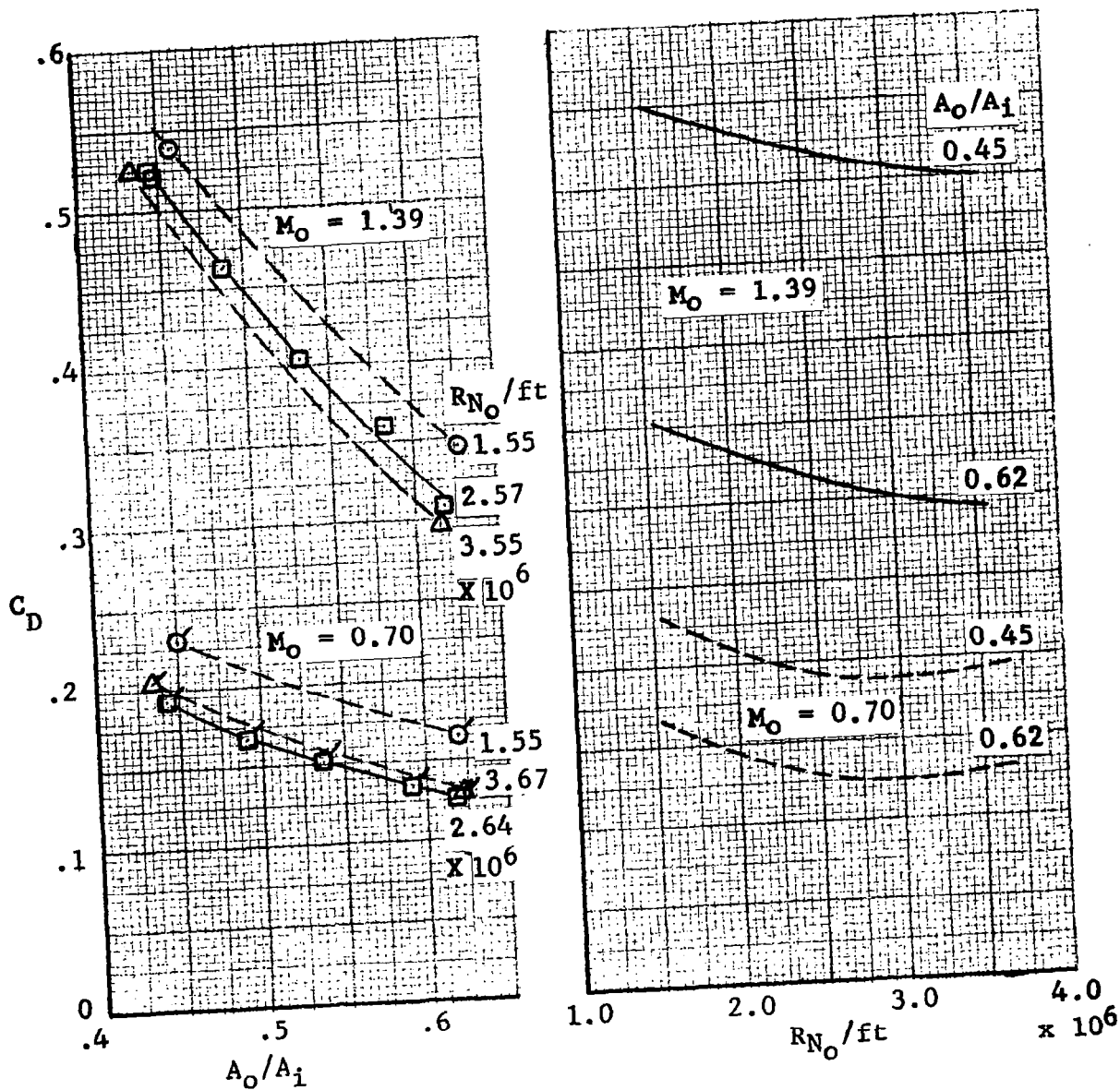
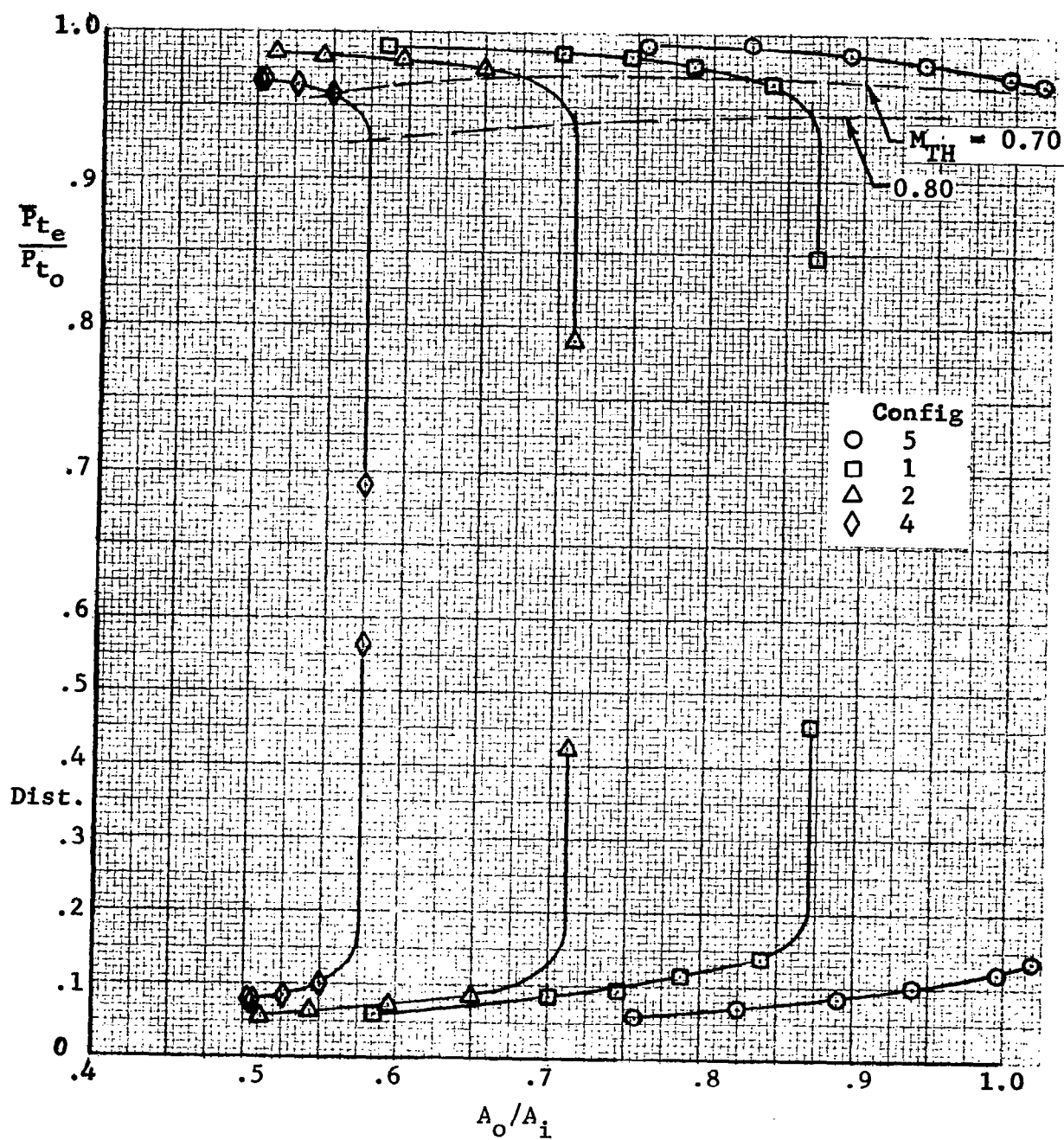
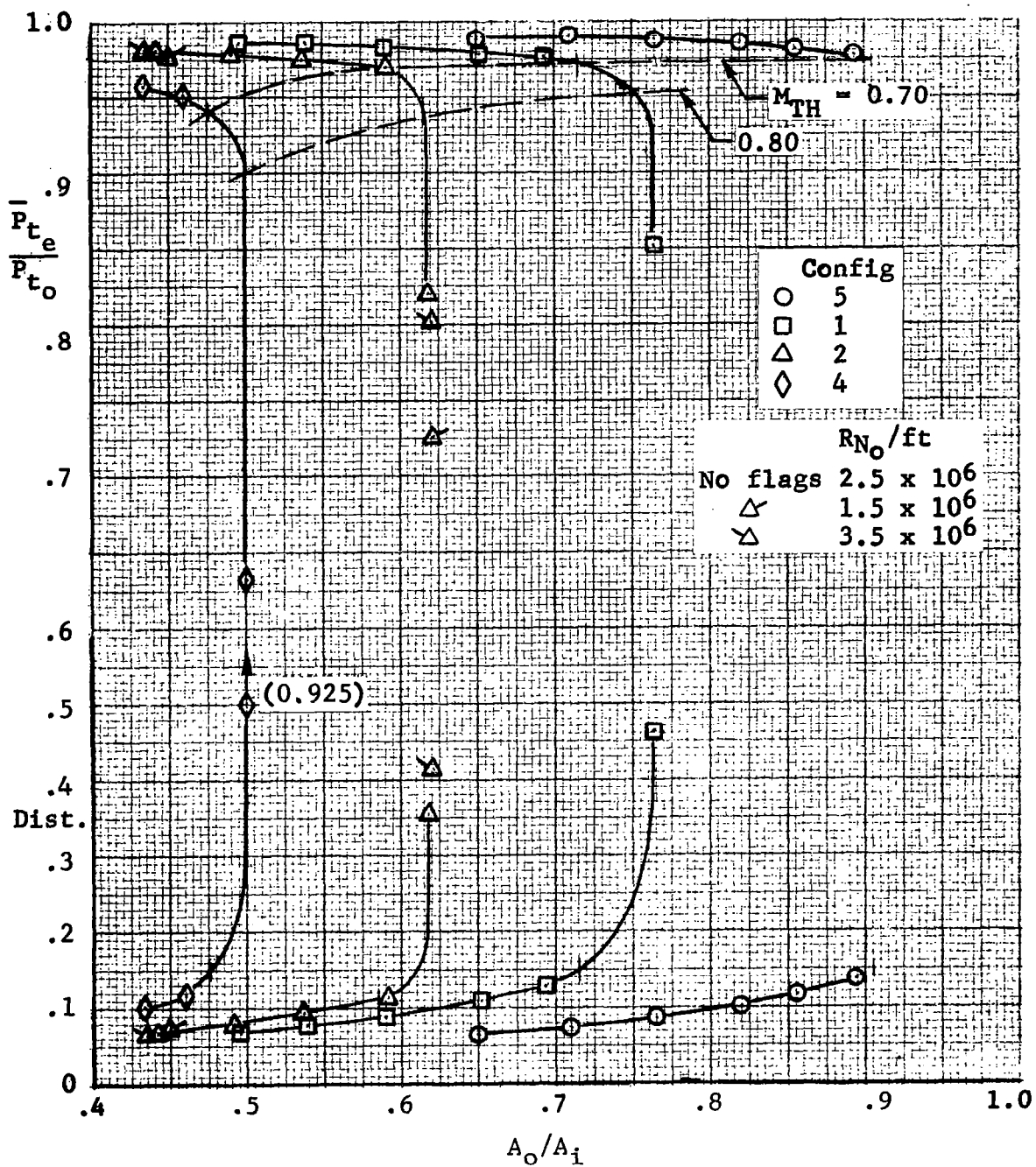


Figure 17.- Effect of Reynolds number on total inlet drag.
Config. 2.



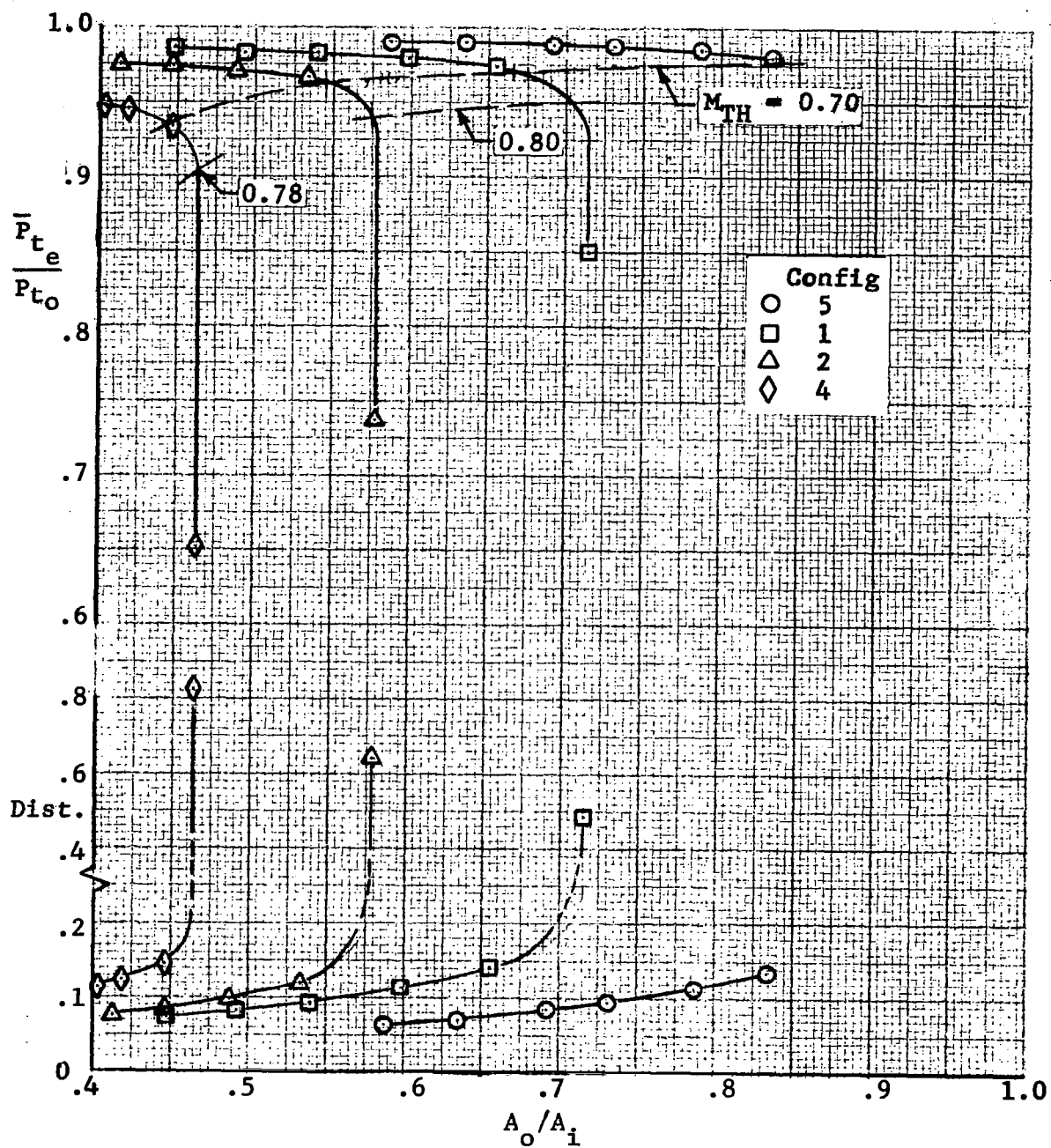
(a) $M_o = 0.55$

Figure 18.- Inlet pressure recovery and distortion.



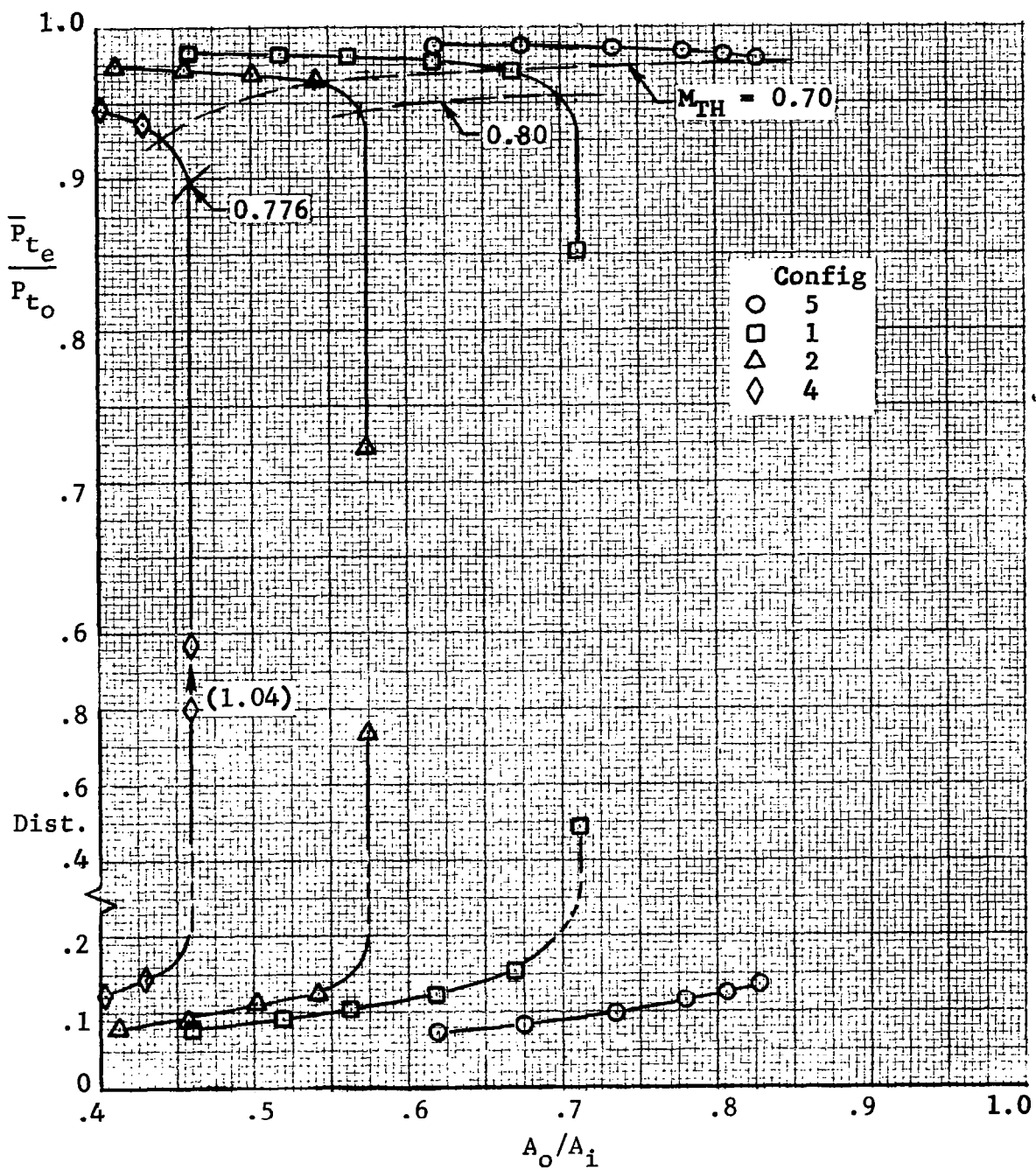
(b) $M_o = 0.70$

Figure 18.- Continued.



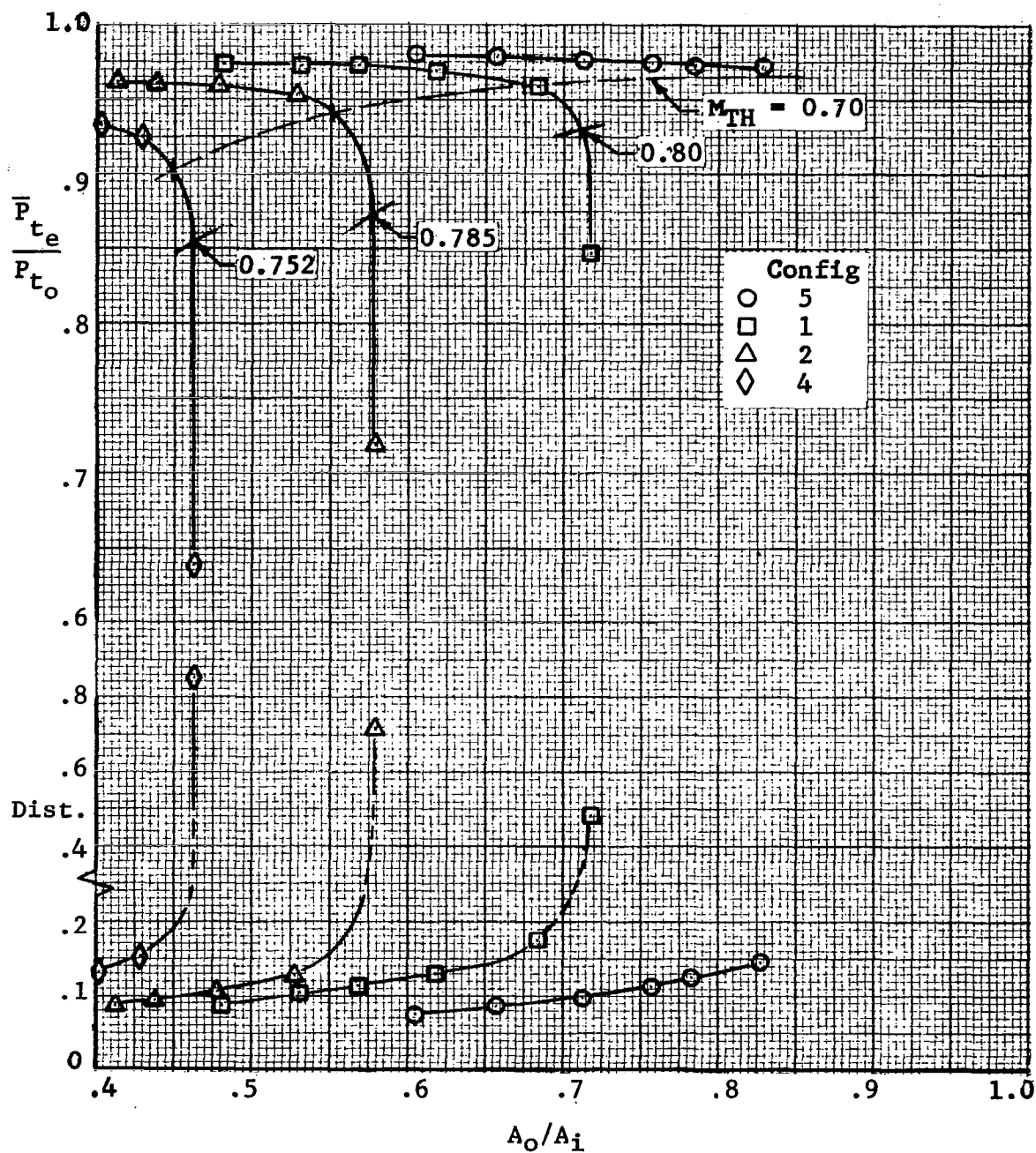
(c) $M_o = 0.85$

Figure 18.- Continued.



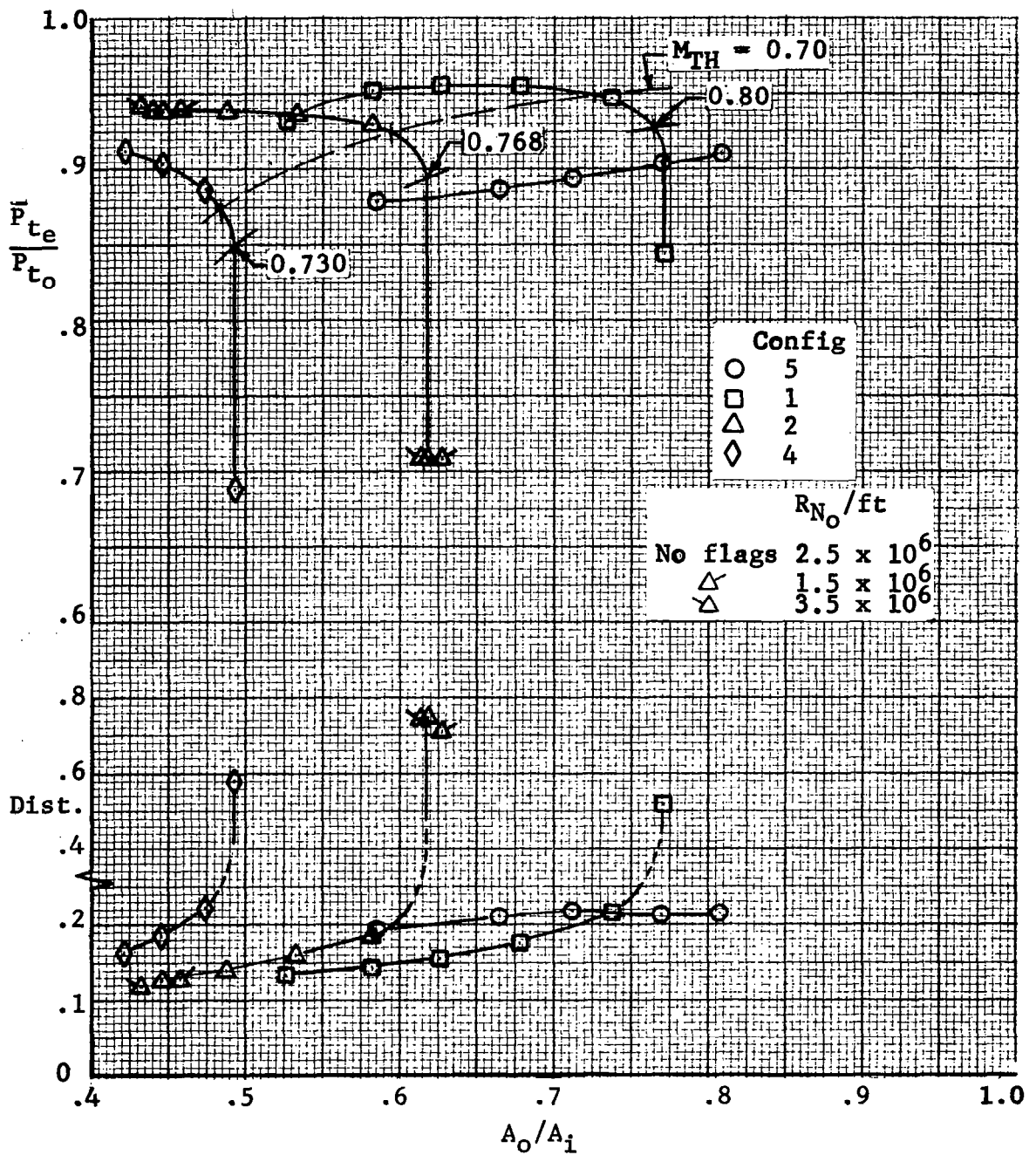
(d) $M_0 = 0.88$

Figure 18.- Continued.



(e) $M_0 = 1.20$

Figure 18.- Continued.



(f) $M_0 = 1.39$

Figure 18.- Concluded.

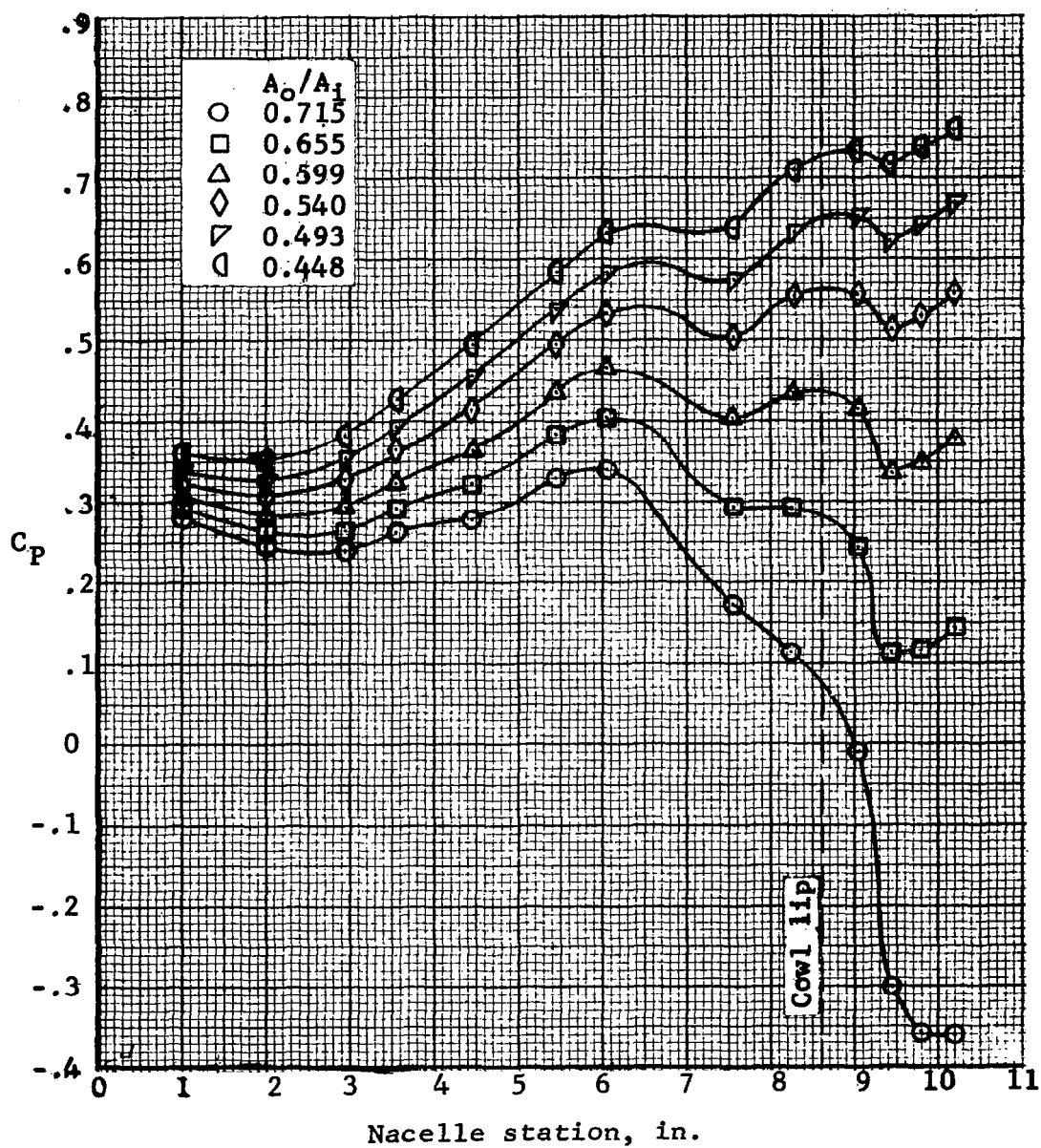


Figure 19.- Effect of capture area ratio on ramp centerline pressure distribution, Config. 1, $M_0 = 0.85$.

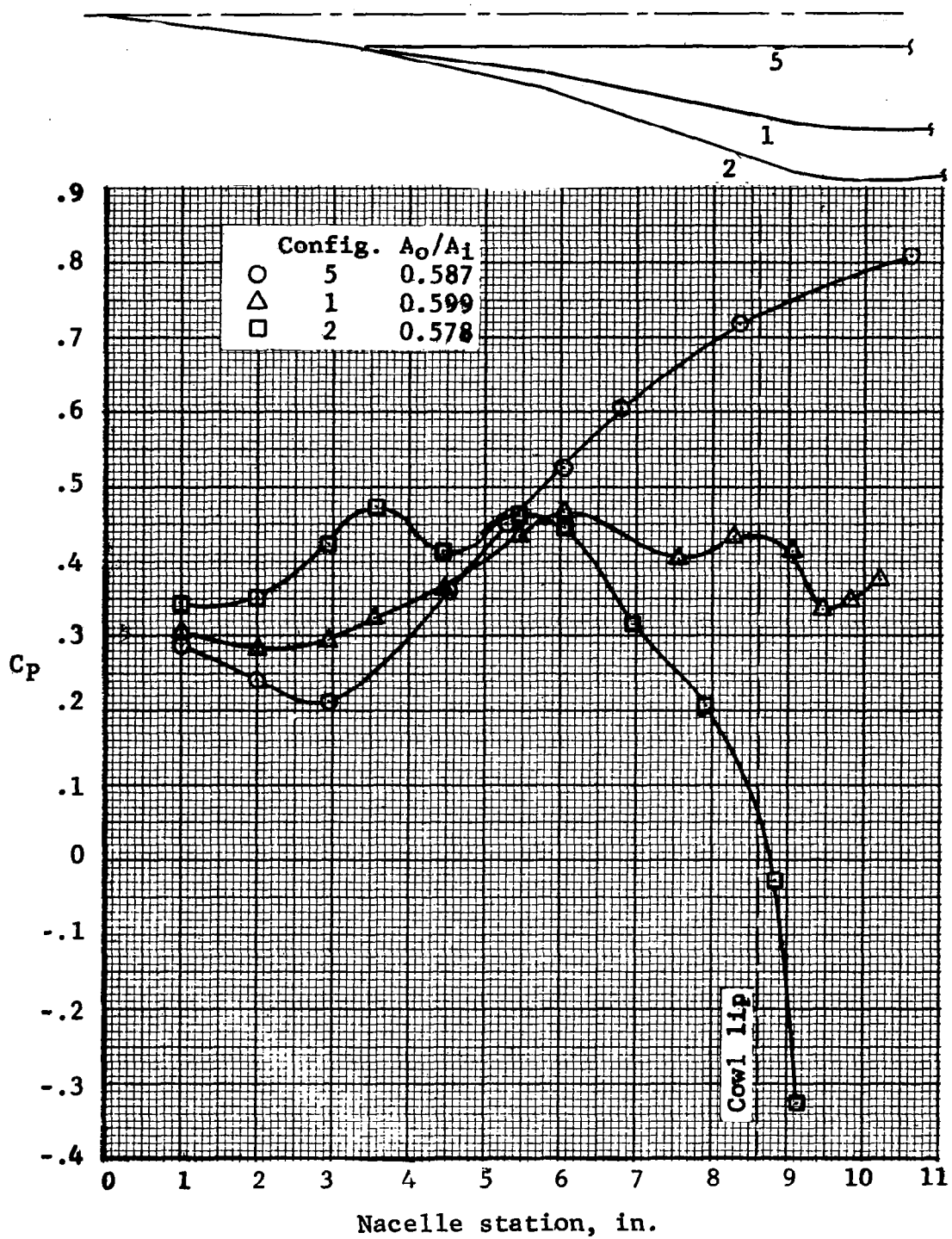


Figure 20.- Effect of ramp angle on ramp centerline pressure distribution, $M_0 = 0.85$.

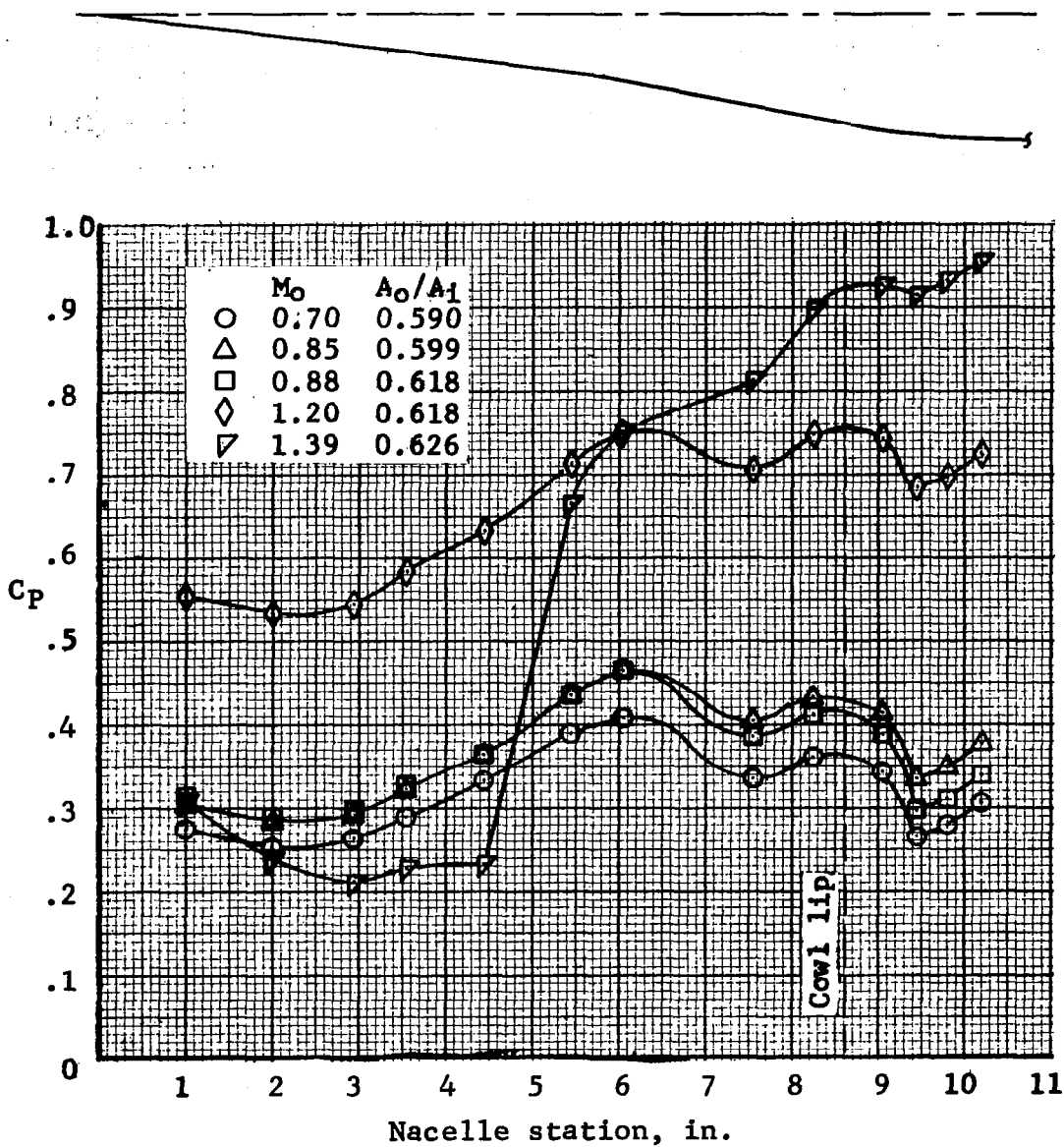


Figure 21.- Effect of Mach number on ramp centerline pressure distribution, Config. 1.

1.64 In. Off
Centerline

Centerline

A_0/A_1

▲
▲
●

○
△
□

.715
.579
.448

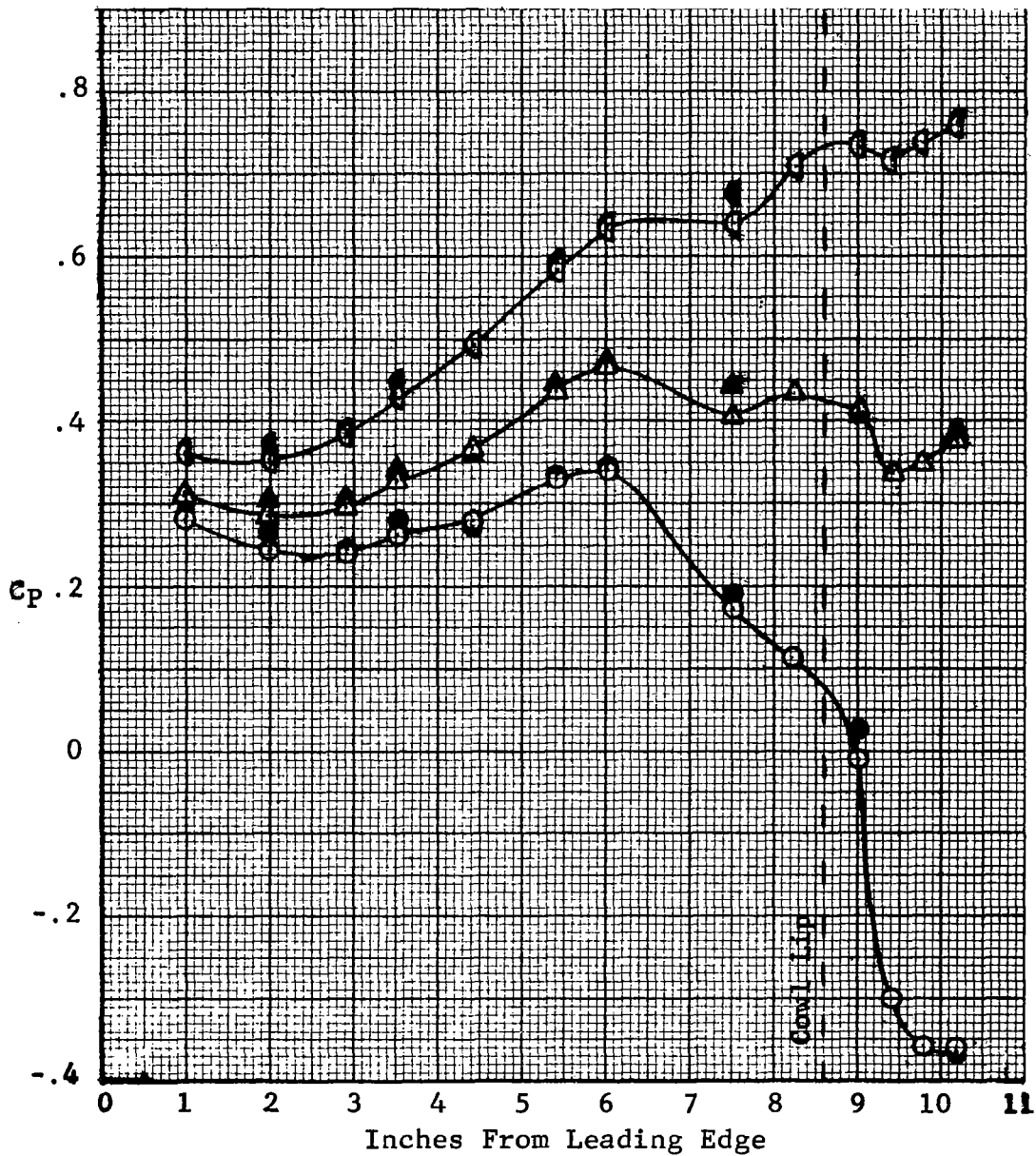


Figure 22. - Comparison between centerline and off-centerline ramp pressure distributions, Config. 1, $M_0 = .85$.

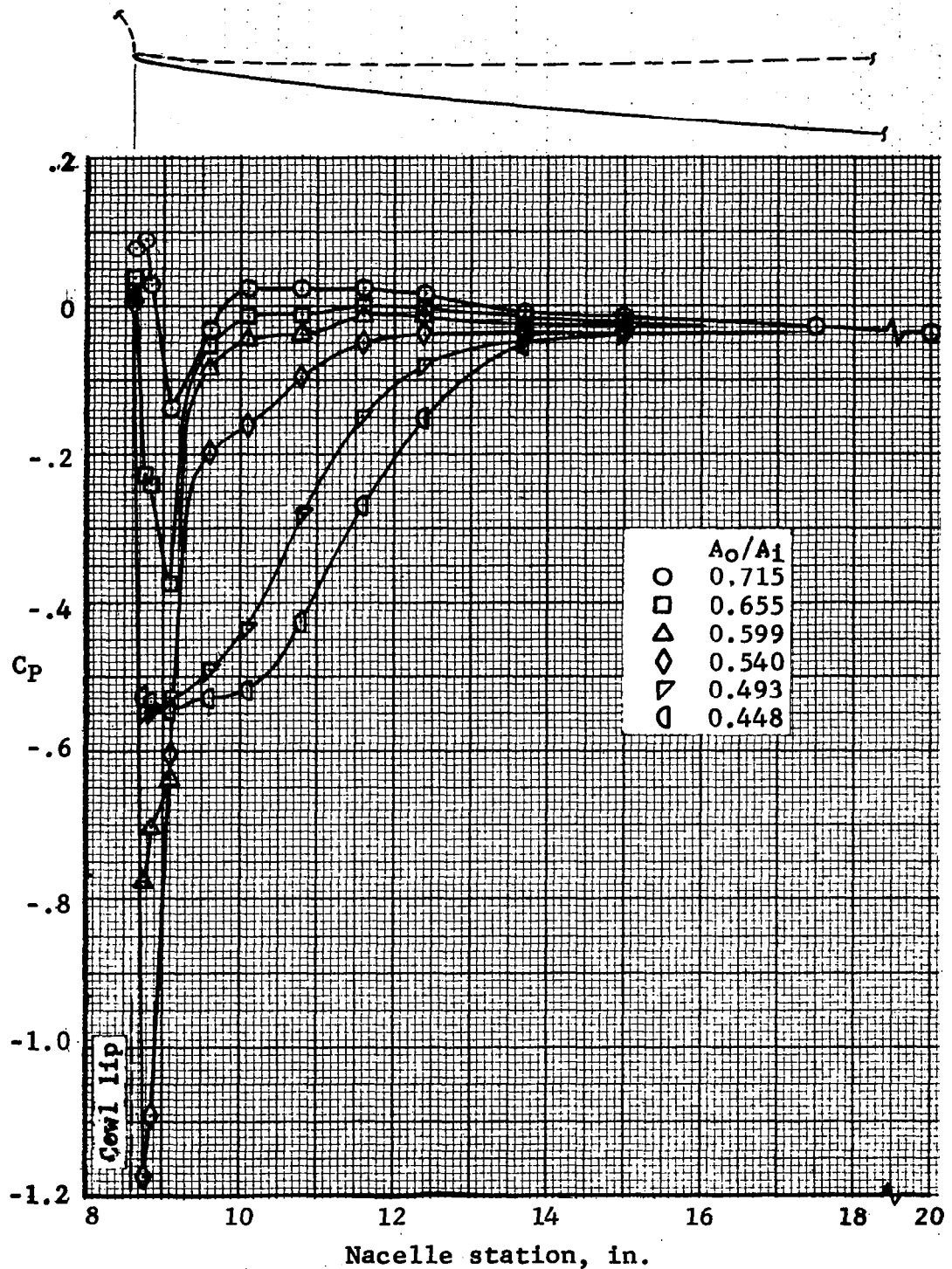


Figure 23.- Effect of capture area ratio on cowl lower centerline pressure distribution, Config. 1, $M_0 = 0.85$.

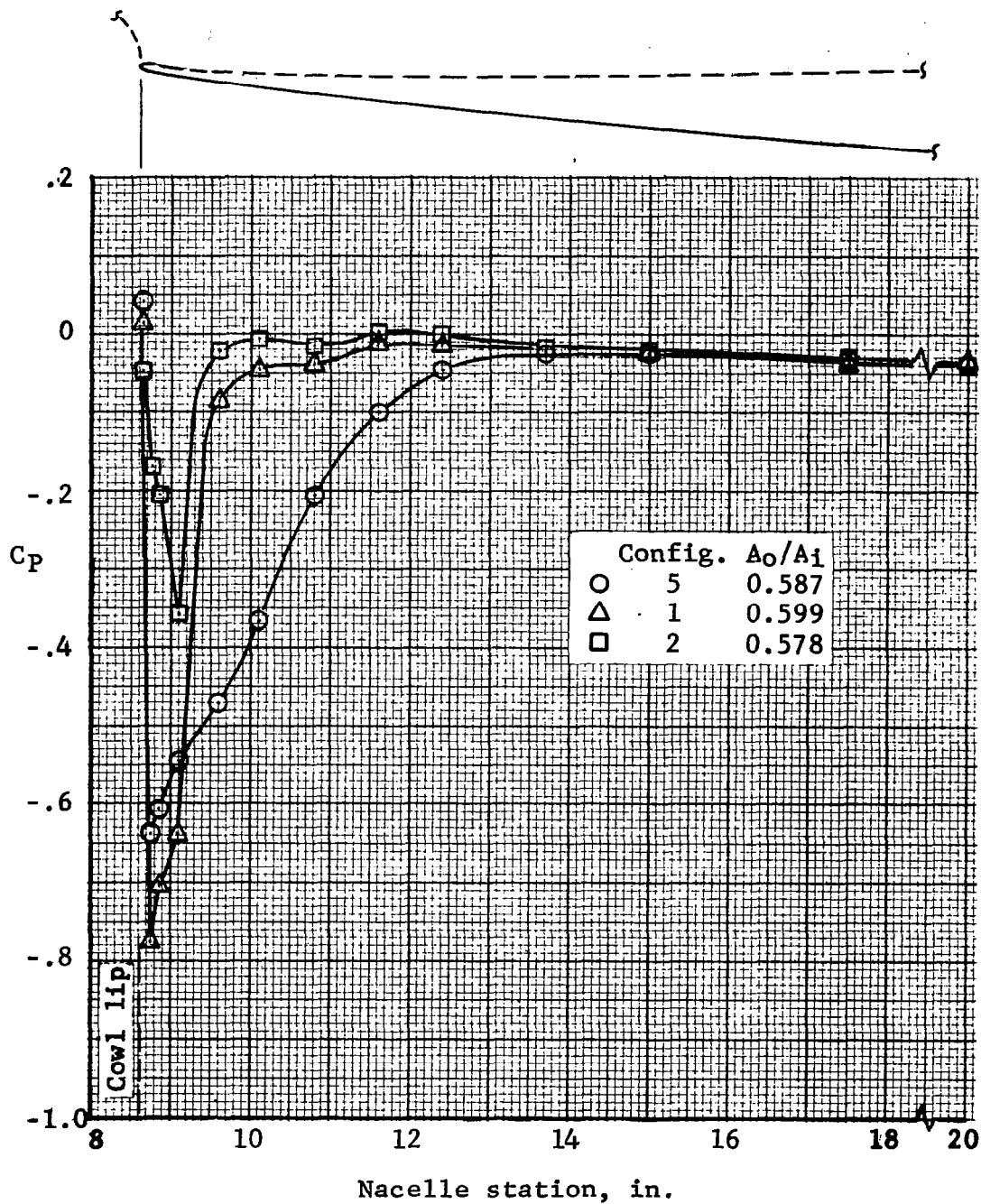


Figure 24.- Effect of ramp angle on cowl lower centerline pressure distribution, $M_0 = 0.85$.

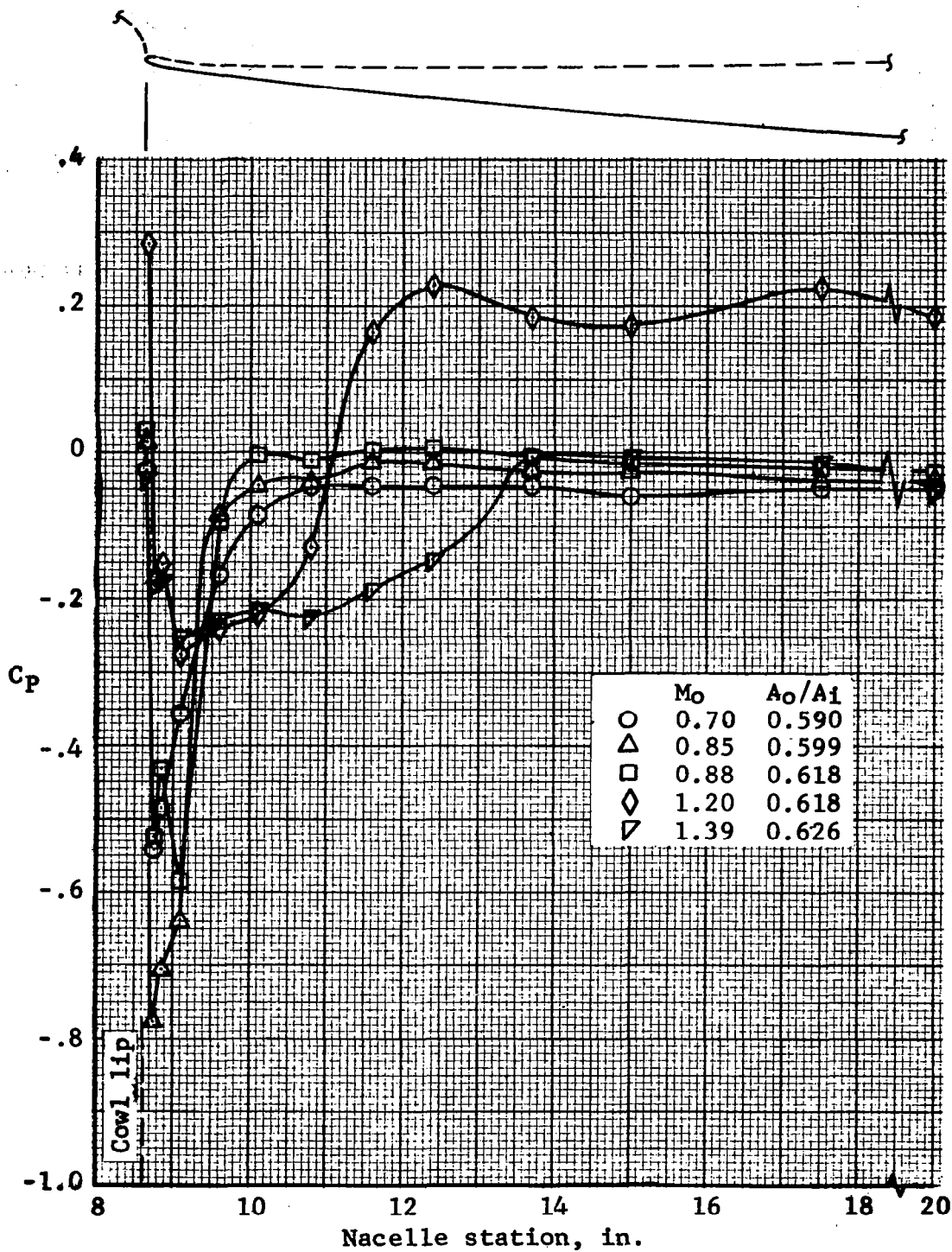


Figure 25.- Effect of Mach number on cowl lower centerline pressure distribution, Config. 1.

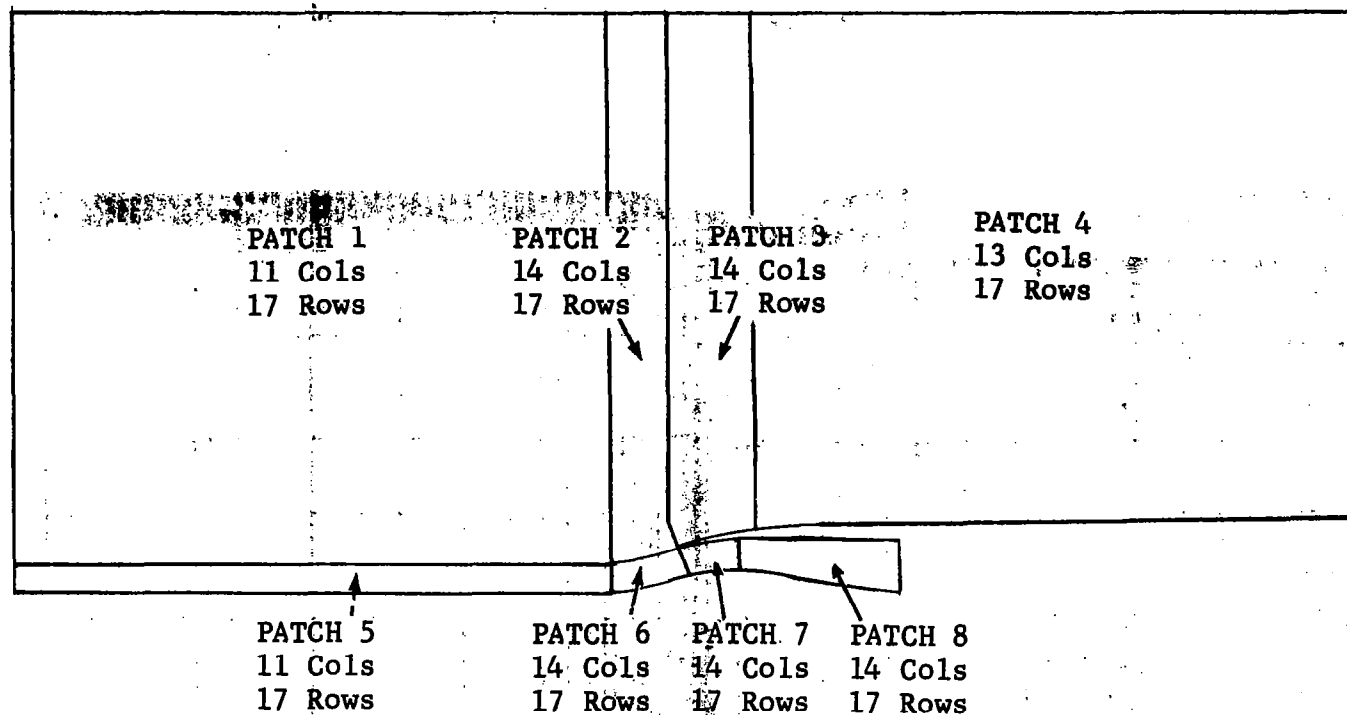


Figure 26.- Computational patch arrangement.

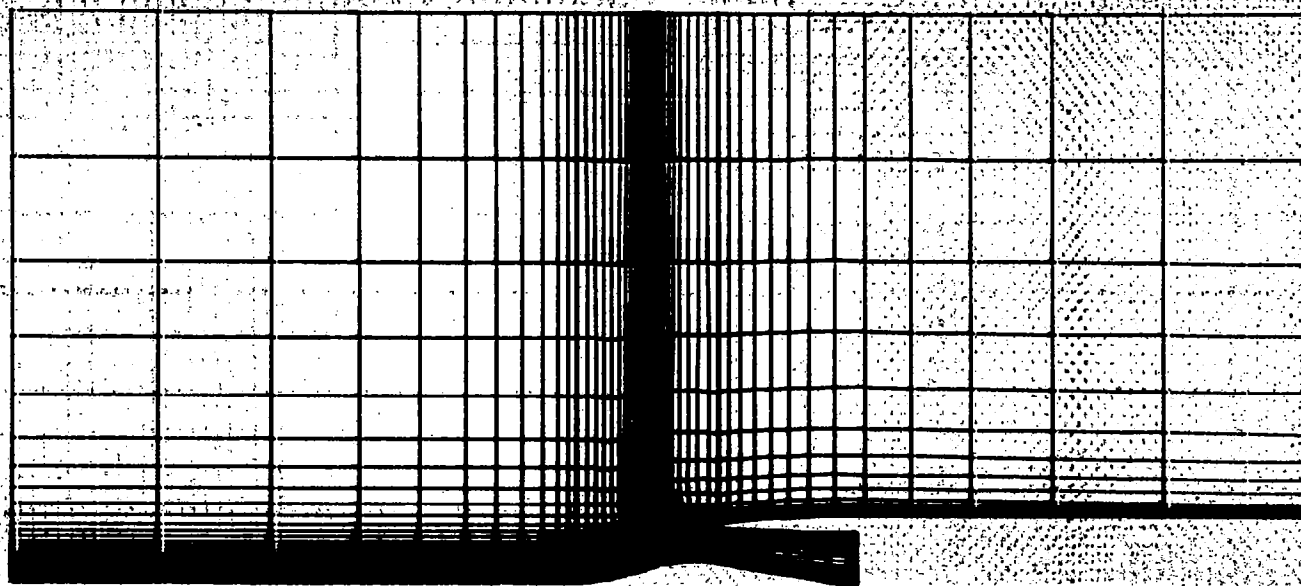


Figure 27.- Computational control volume.

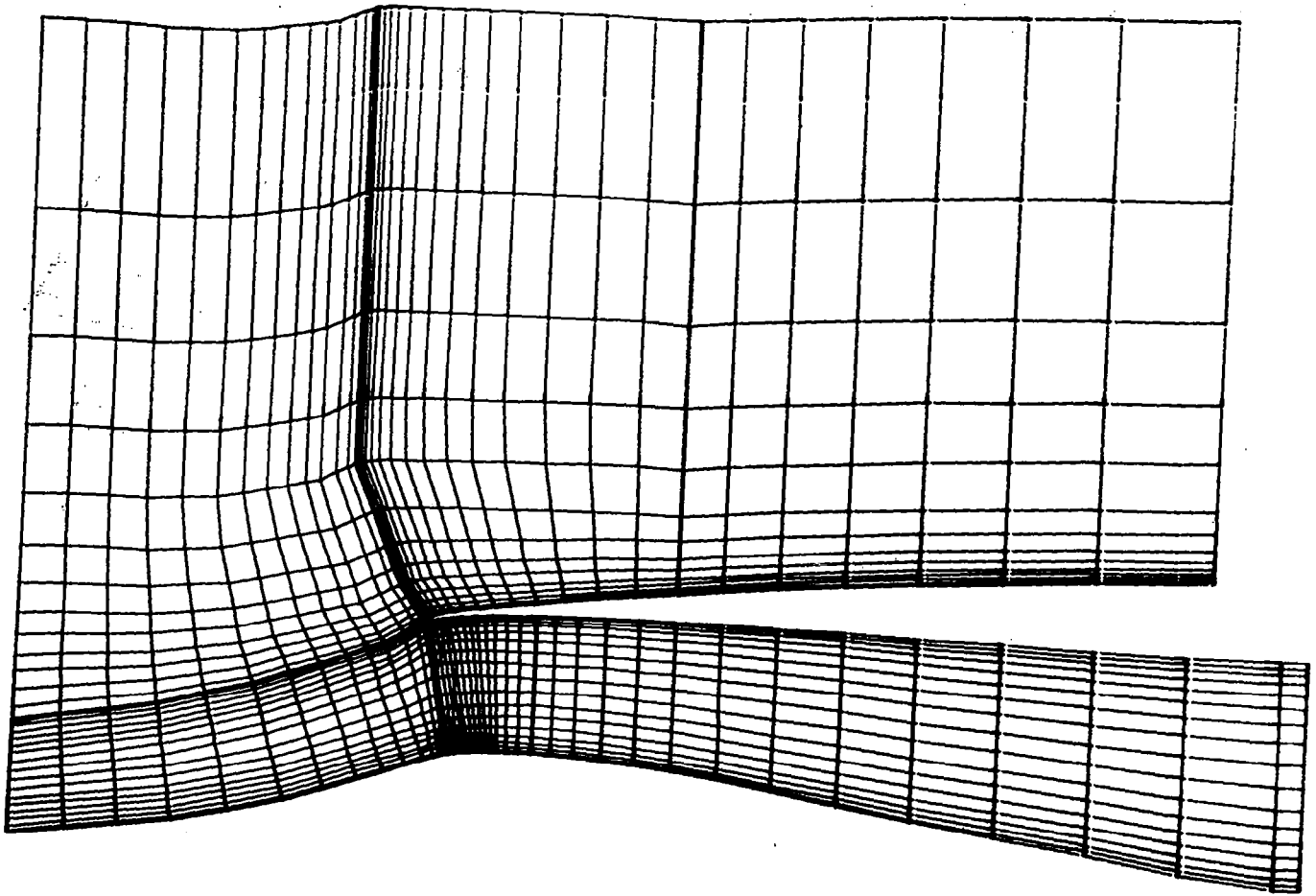


Figure 28.- Computational cell arrangement near inlet.

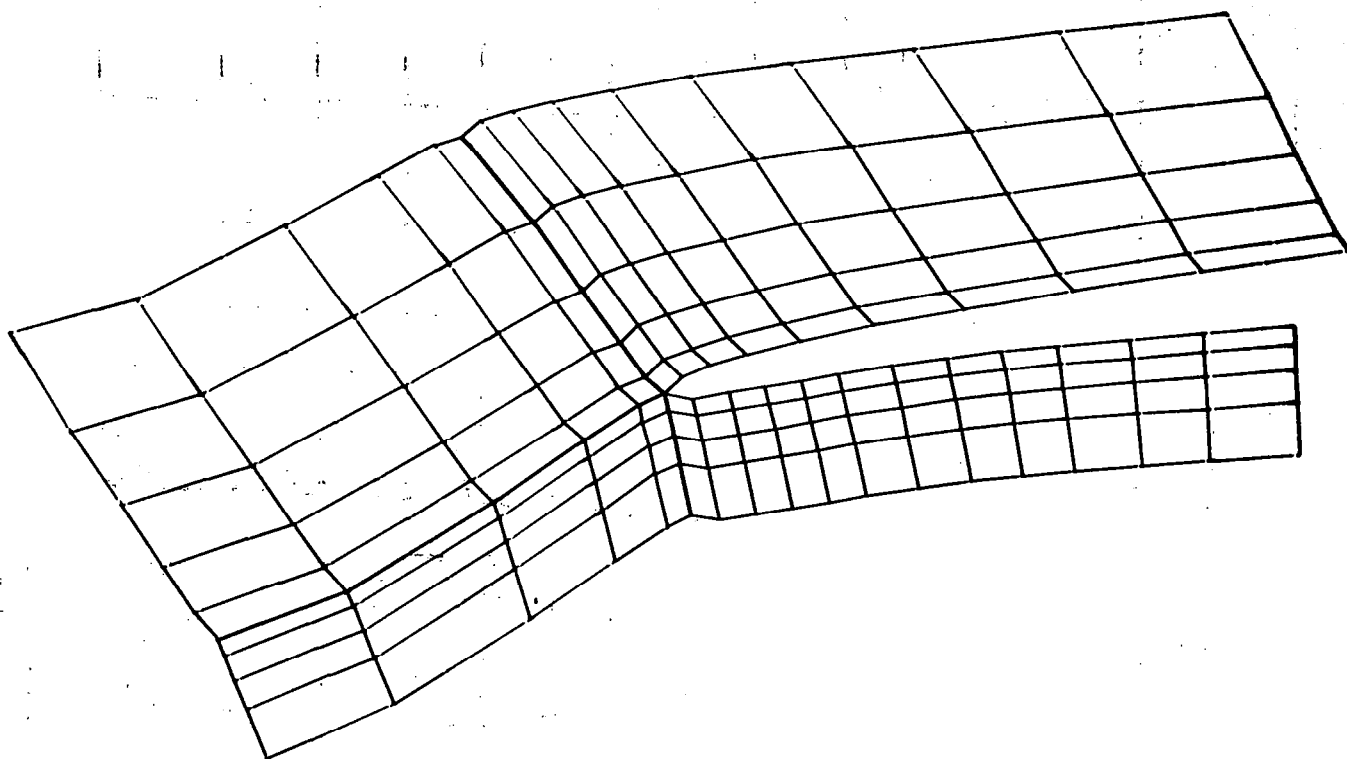


Figure 29.- Computational cell arrangement near cowl lip.

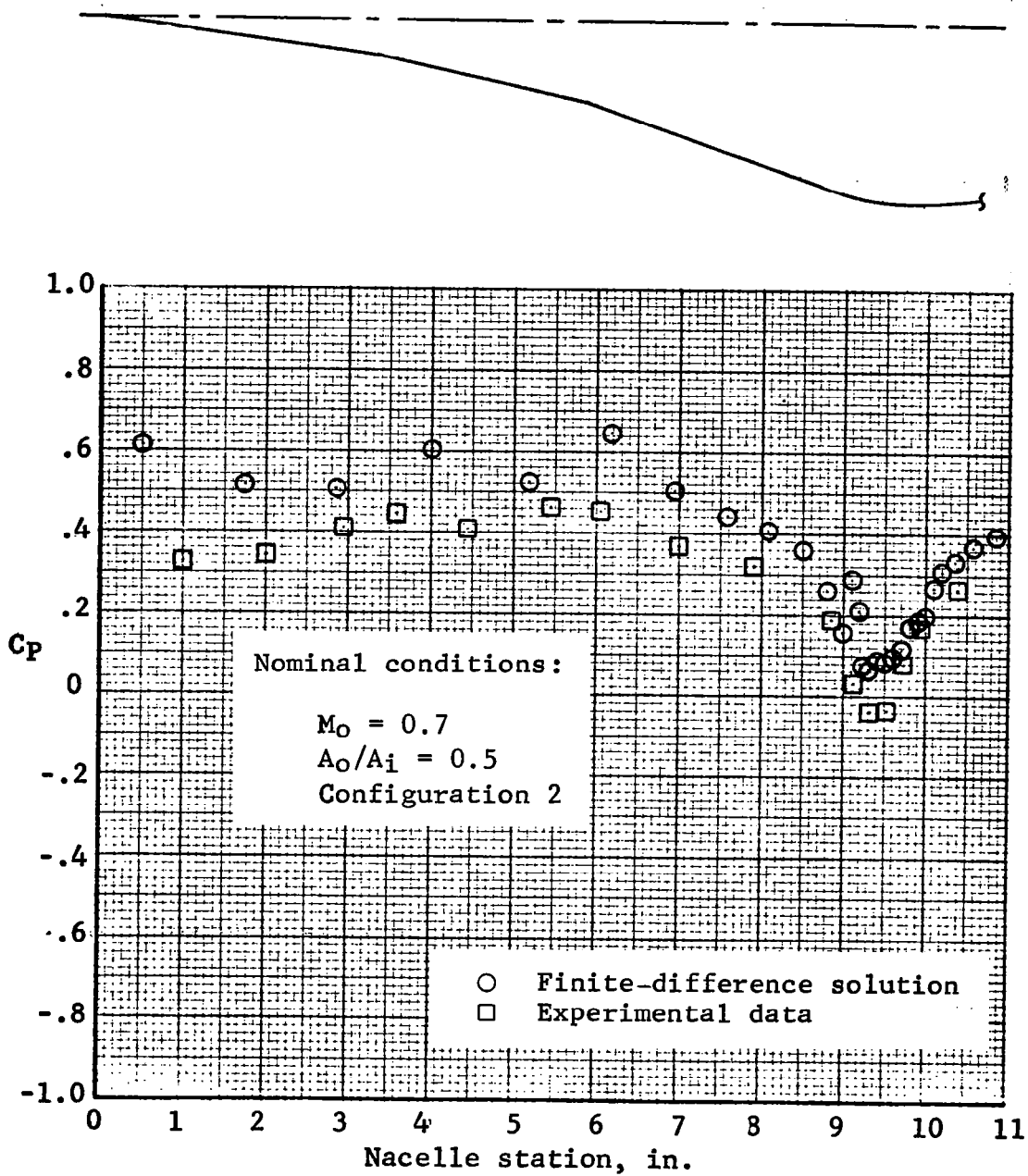


Figure 30.- Ramp pressure distribution, Case I.

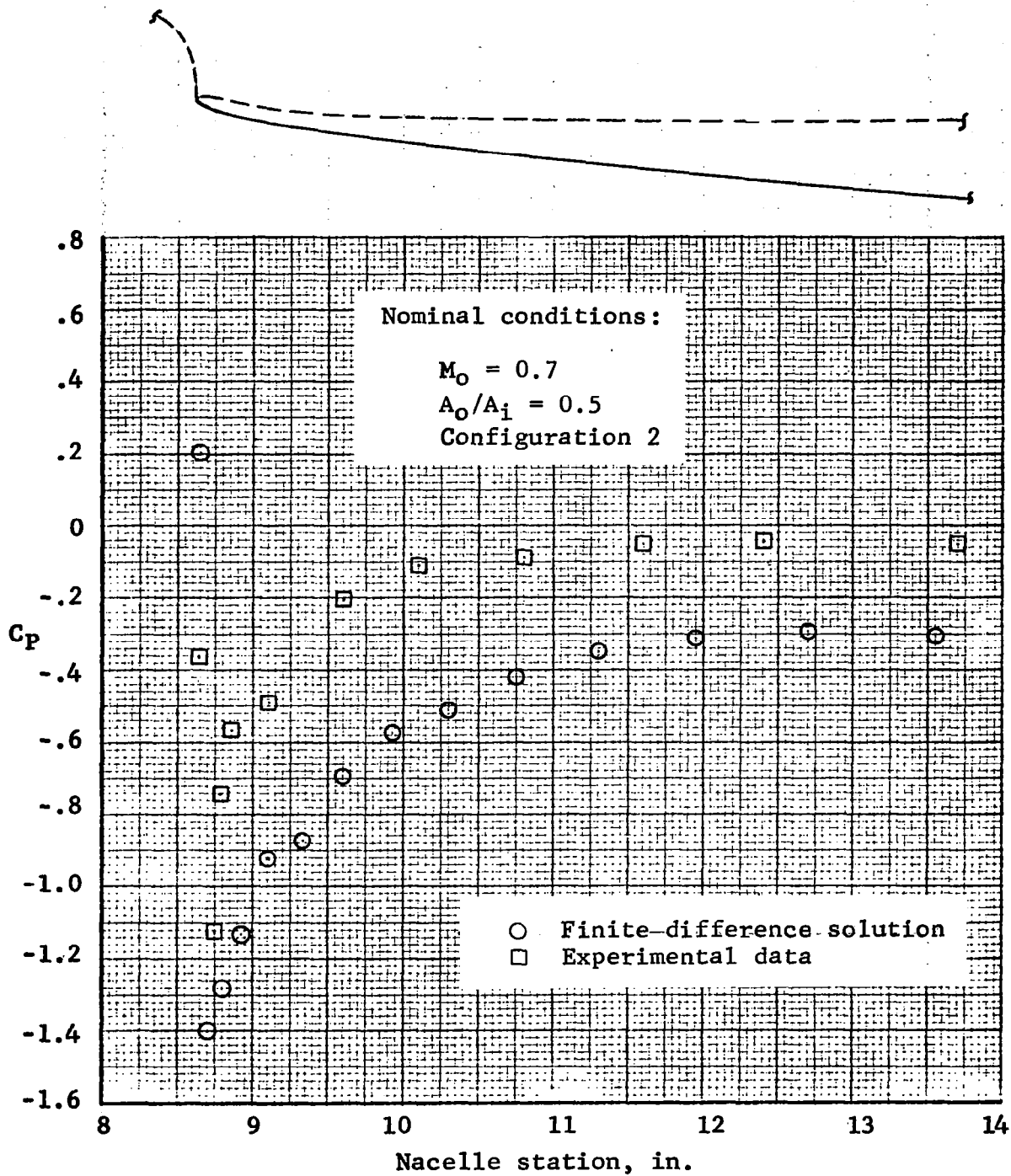


Figure 31.- Cowl external pressure distribution, Case I.

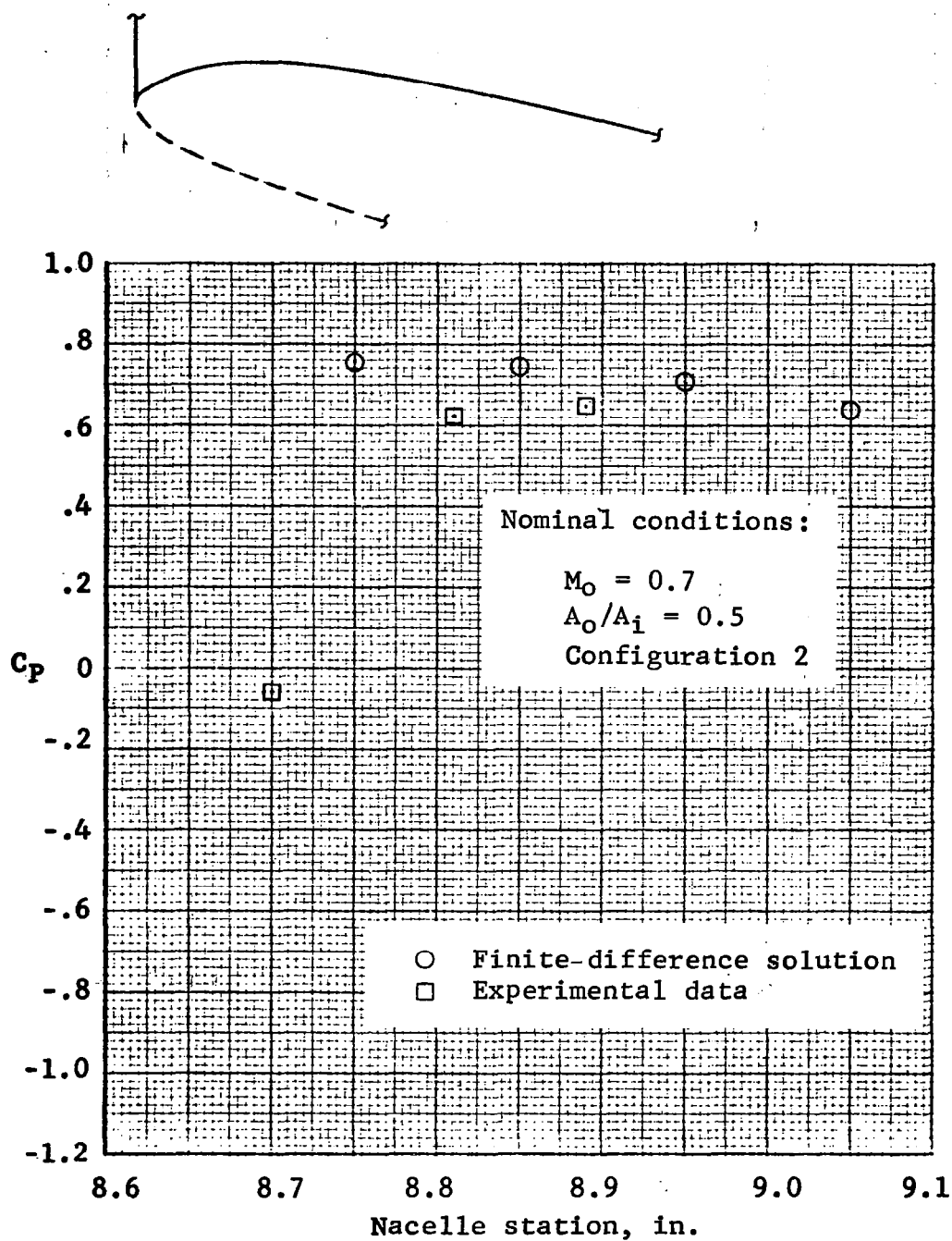


Figure 32.- Cowl lip internal pressure distribution, Case I.

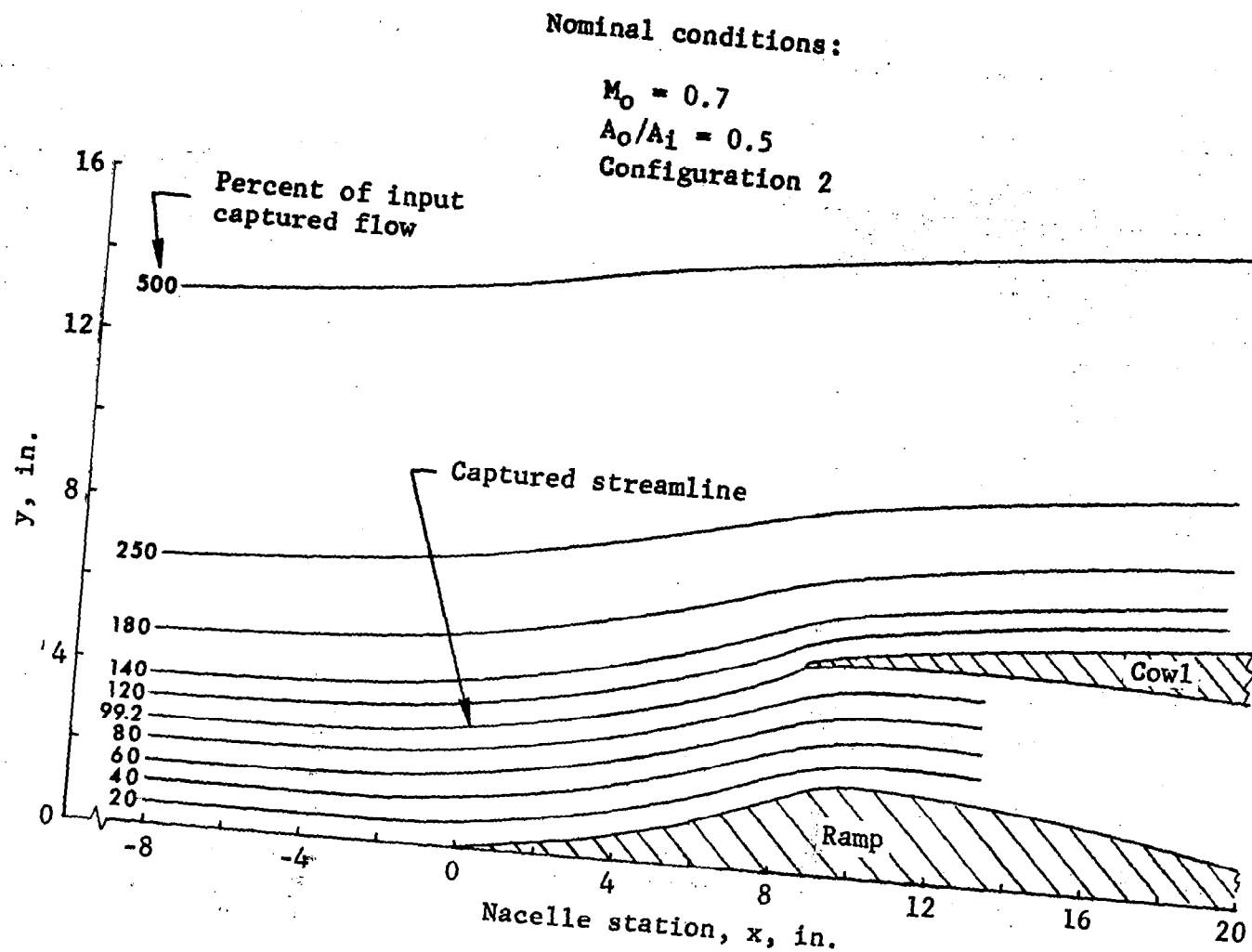


Figure 33.- Computed flow field streamlines, Case I.

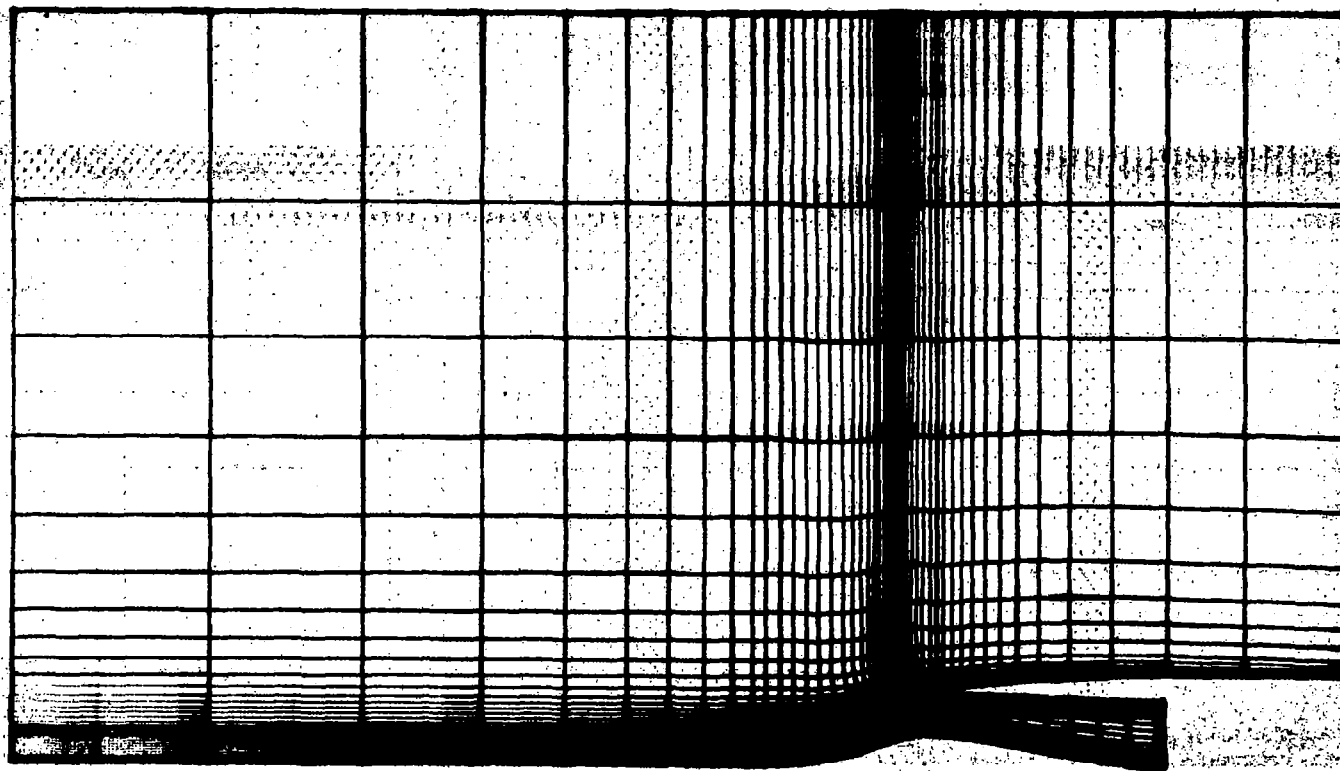


Figure 34.- Revised computational control volume.

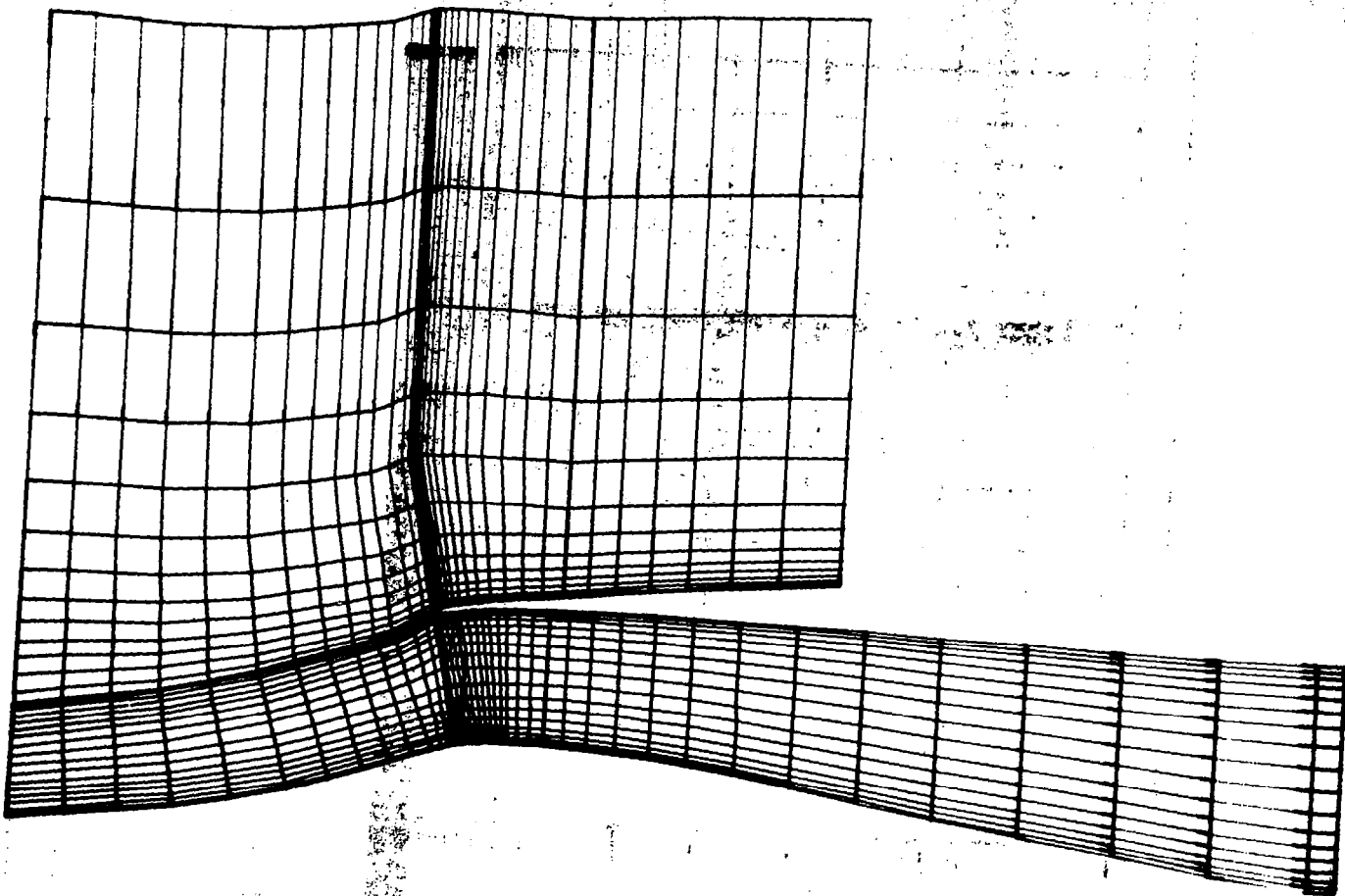


Figure 35.- Revised computational cell arrangement near inlet.

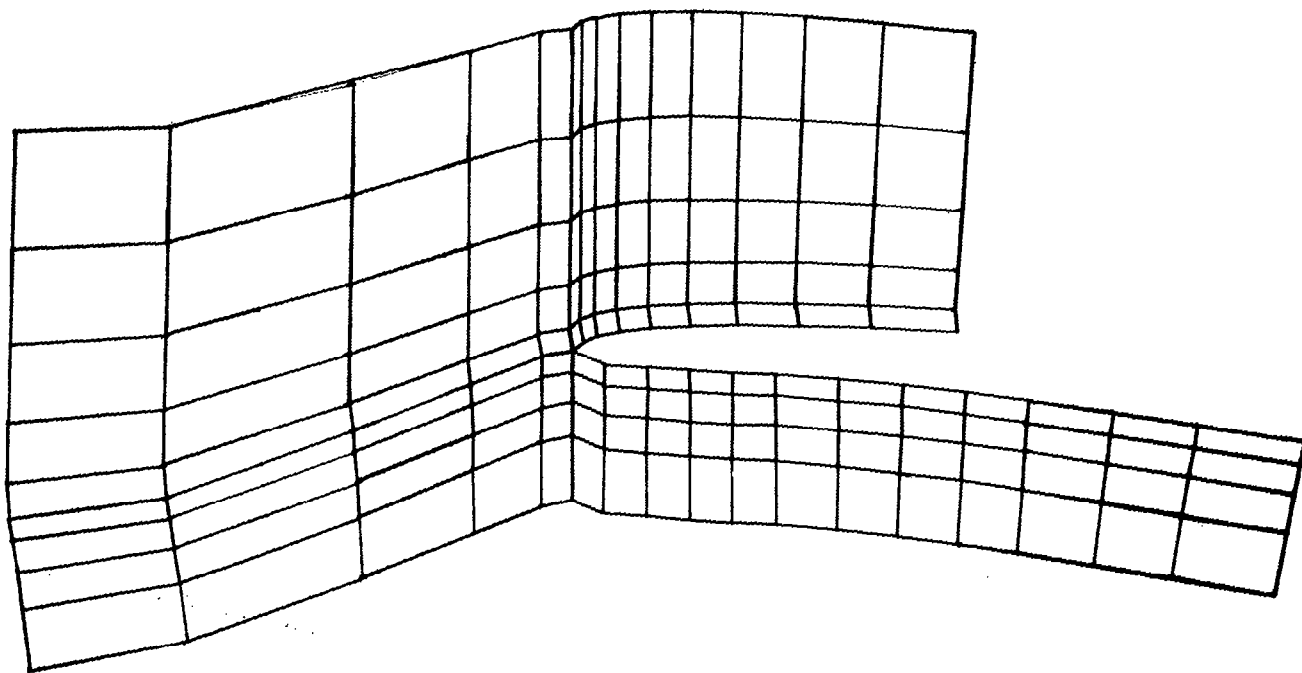


Figure 36.- Revised computational cell arrangement near cowl lip.

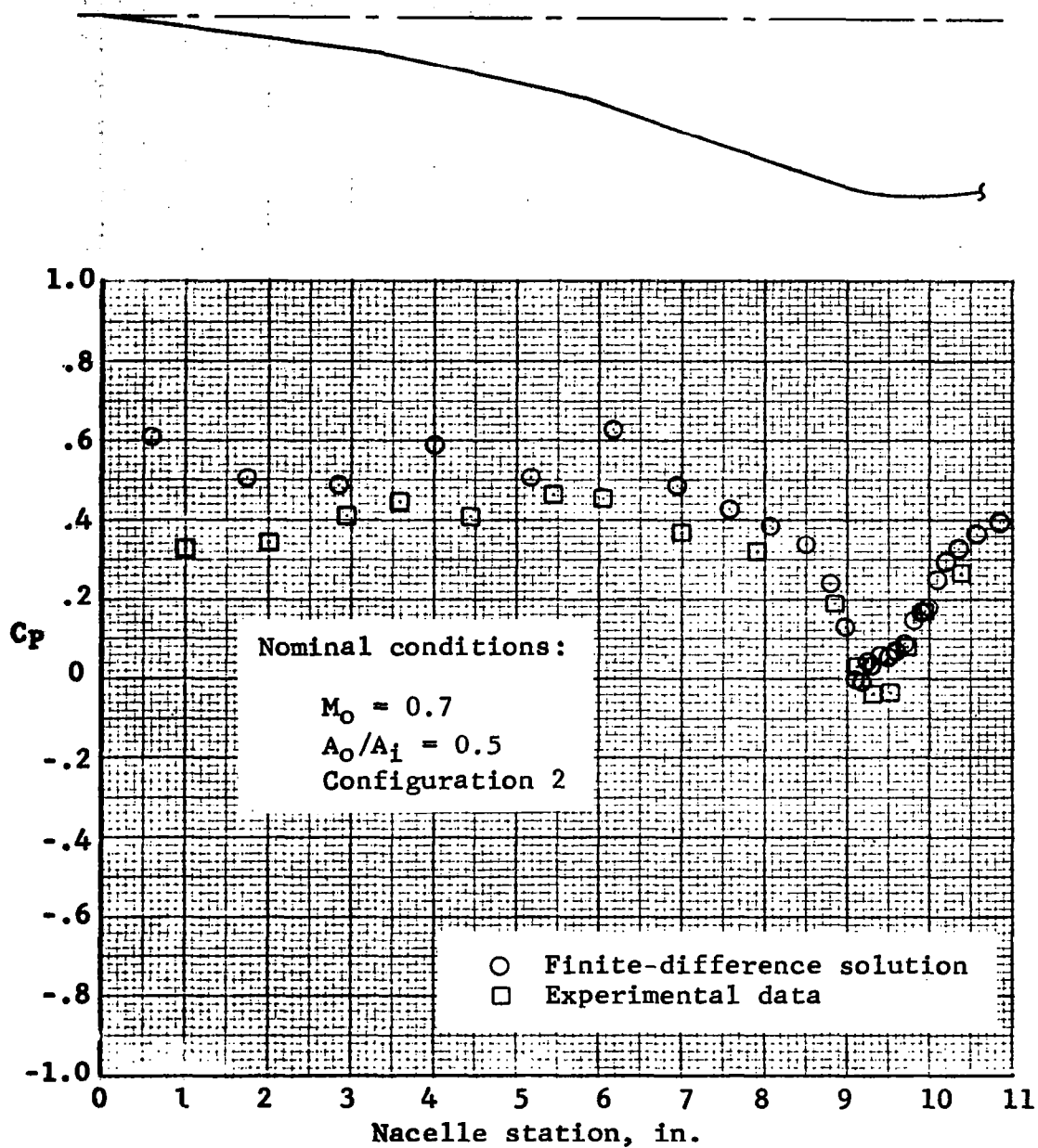


Figure 37.- Ramp pressure distribution, Case Ia.

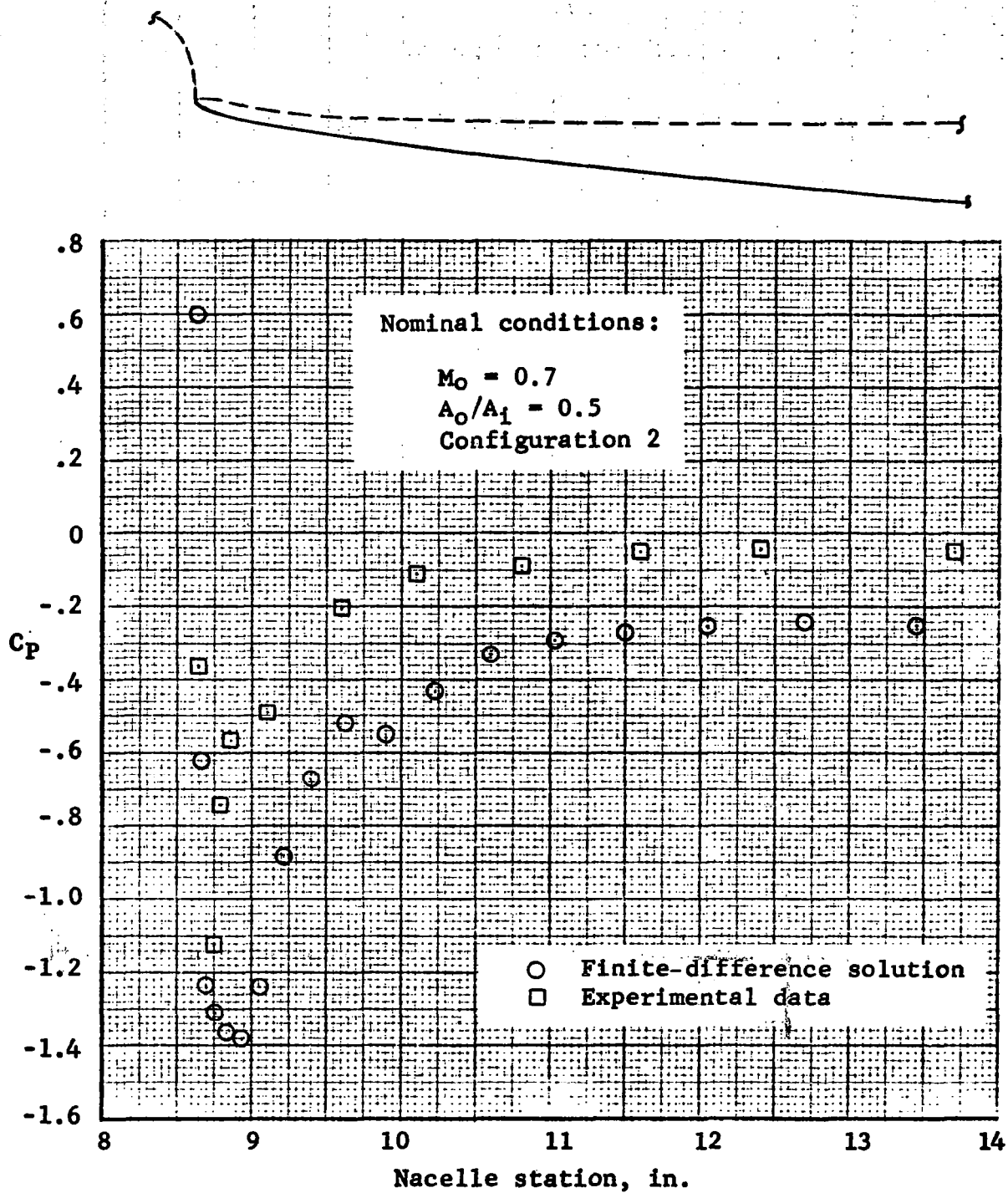


Figure 38.- Cowl external pressure distribution, Case Ia.

Data at Nacelle Station 9.6 on the Lower Cowl

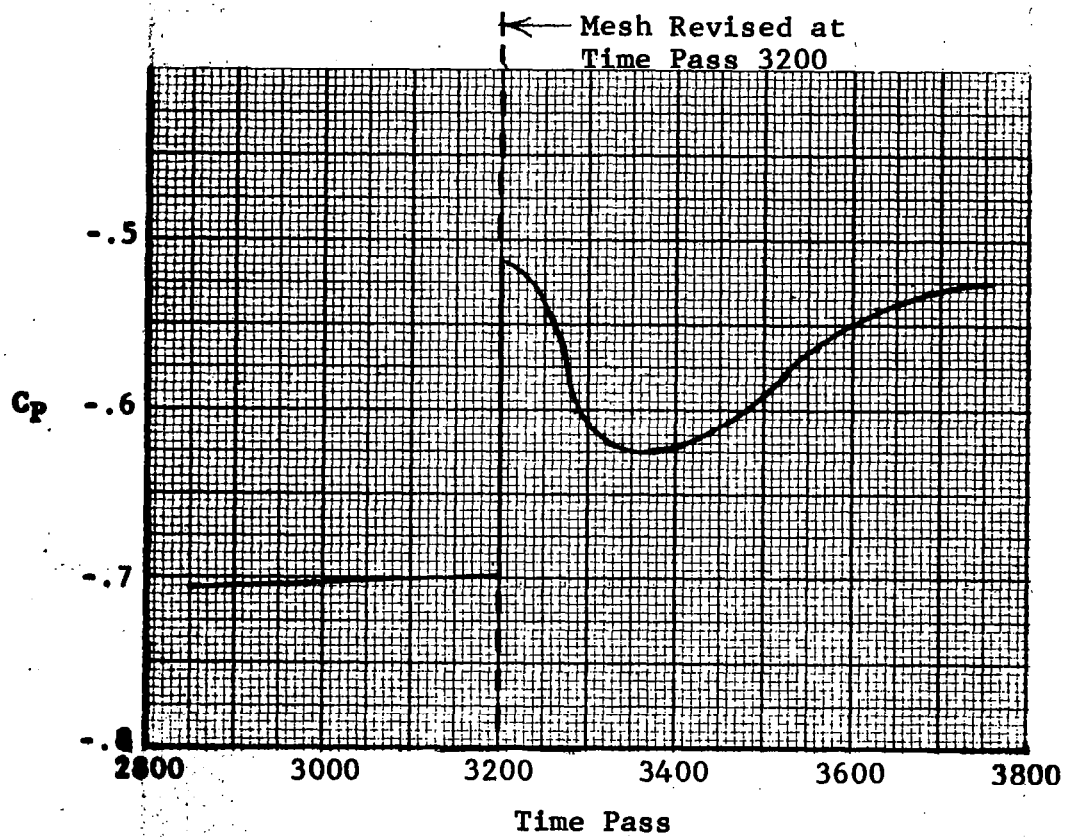


Figure 39 Effect of Mesh Change on Cowl Pressure

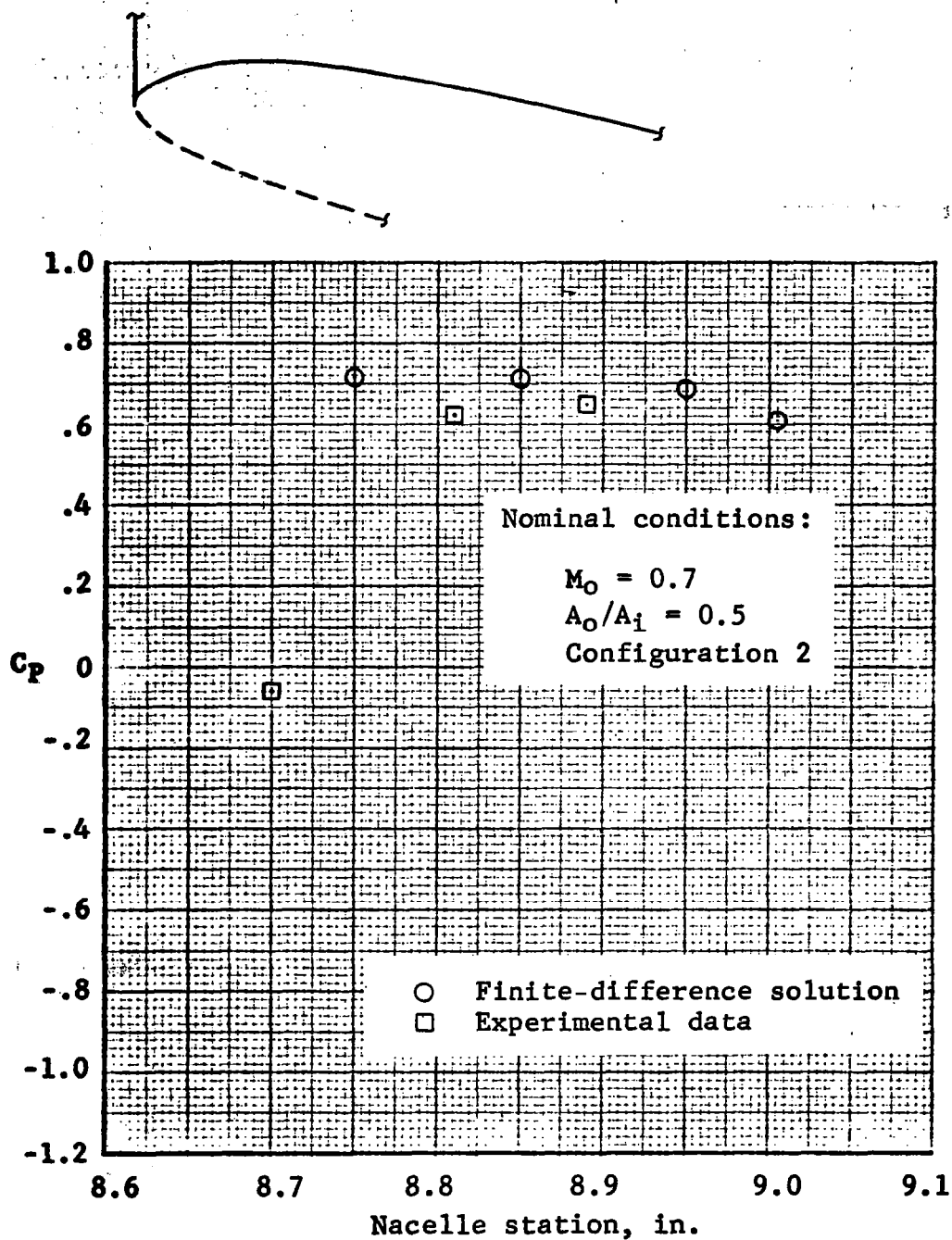


Figure 40.- Cowl lip internal pressure distribution, Case Ia.

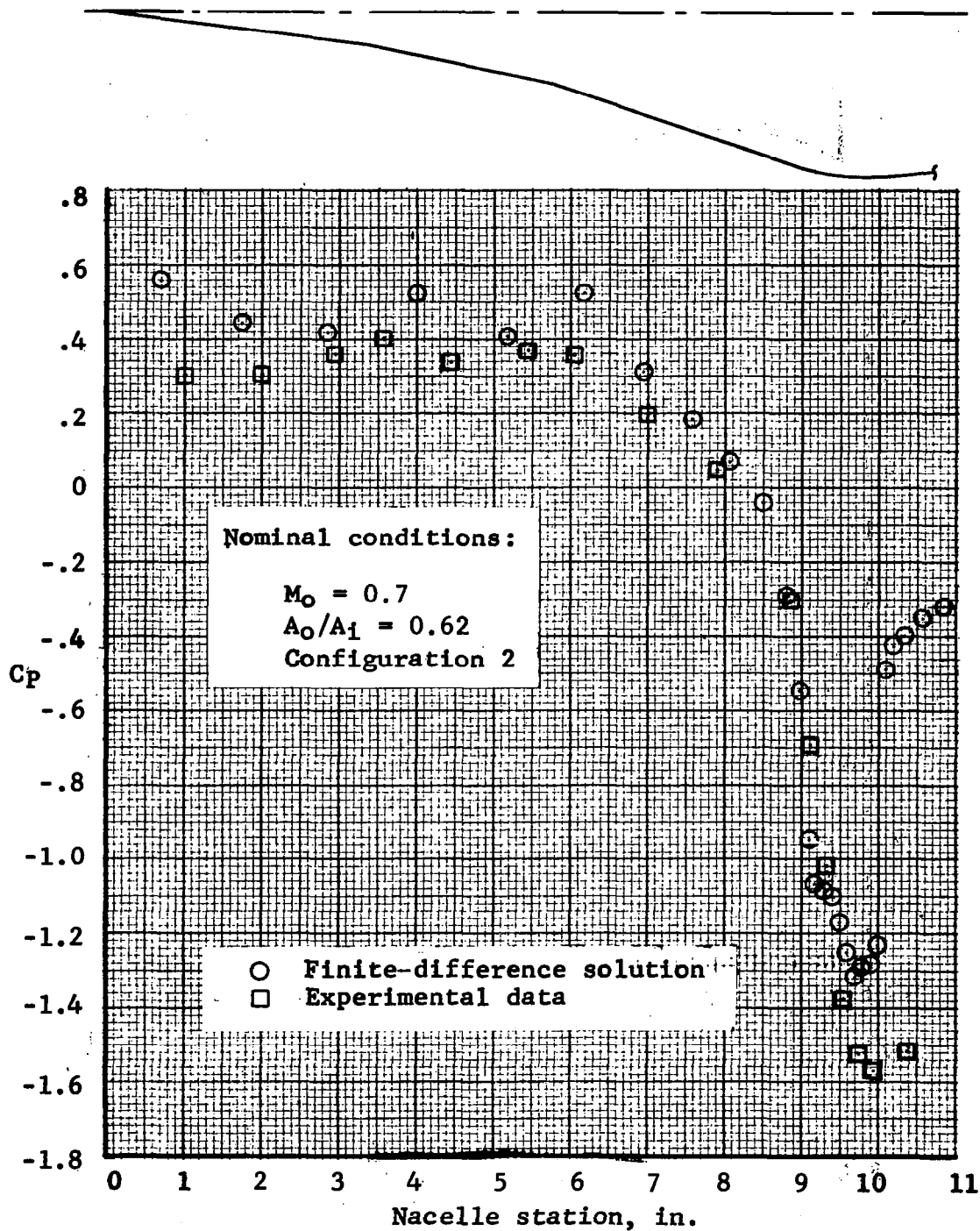


Figure 41.- Ramp pressure distribution, Case II.

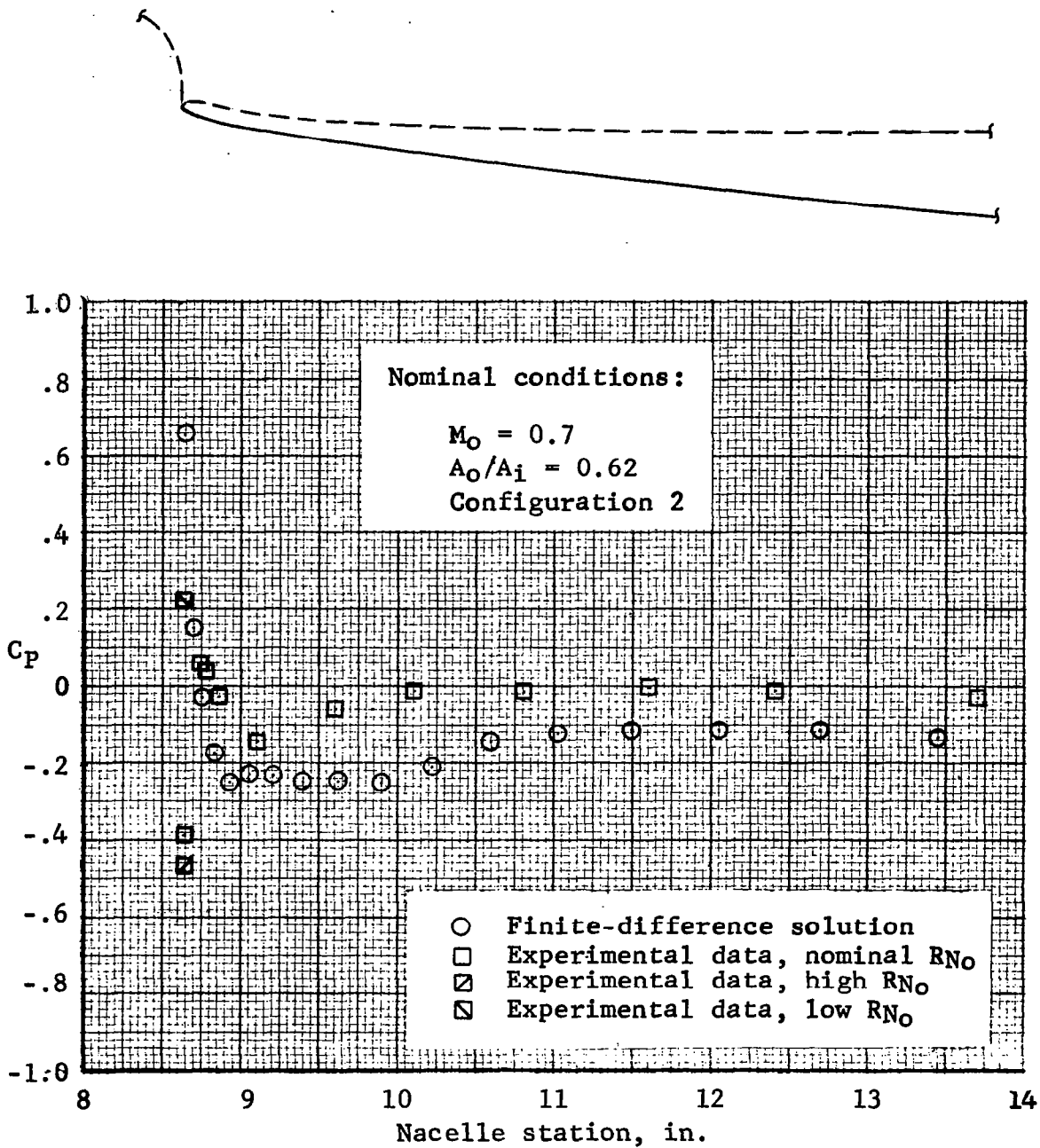


Figure 42.- Cowl external pressure distribution, Case II.

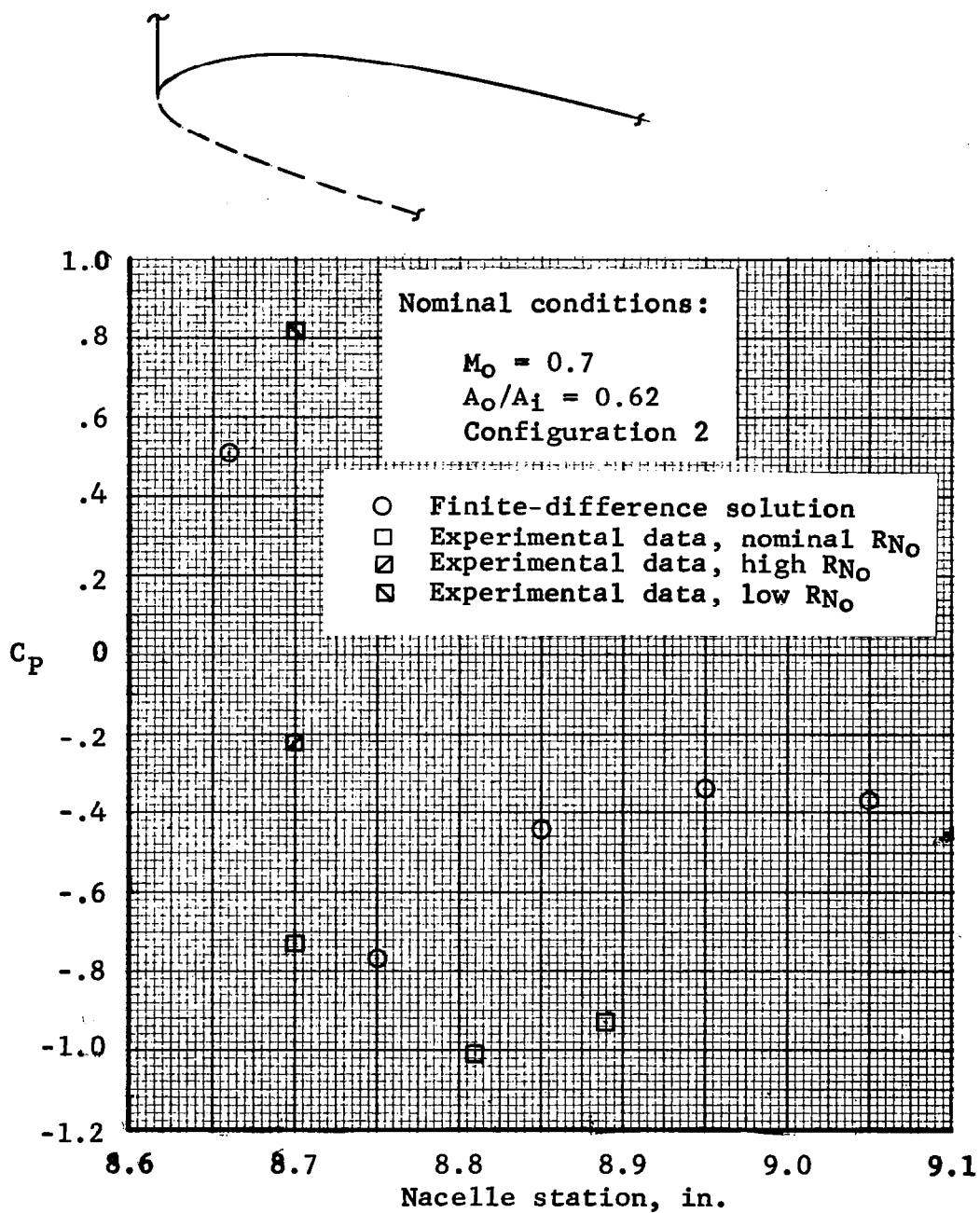


Figure 43.- Cowl lip internal pressure distribution, Case II.

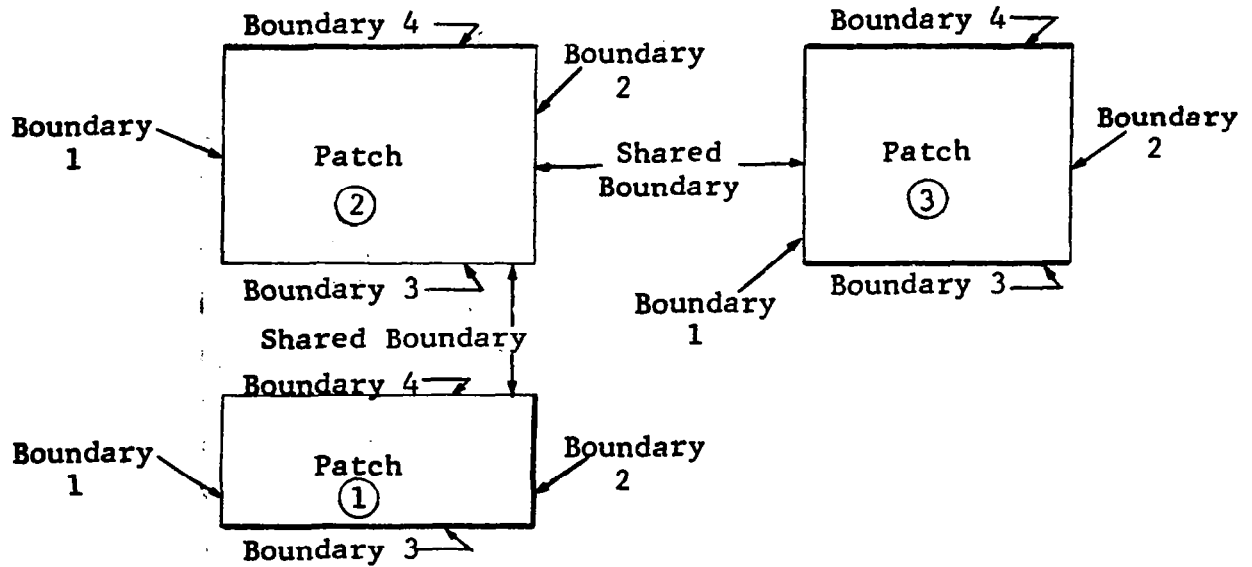
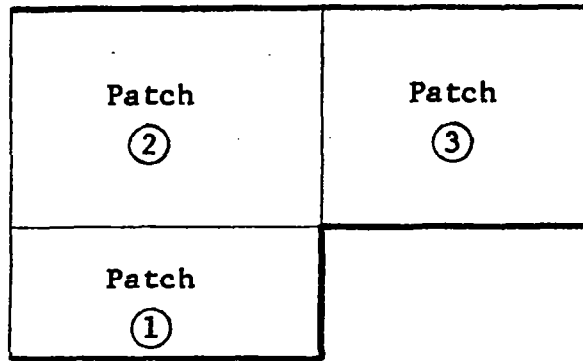
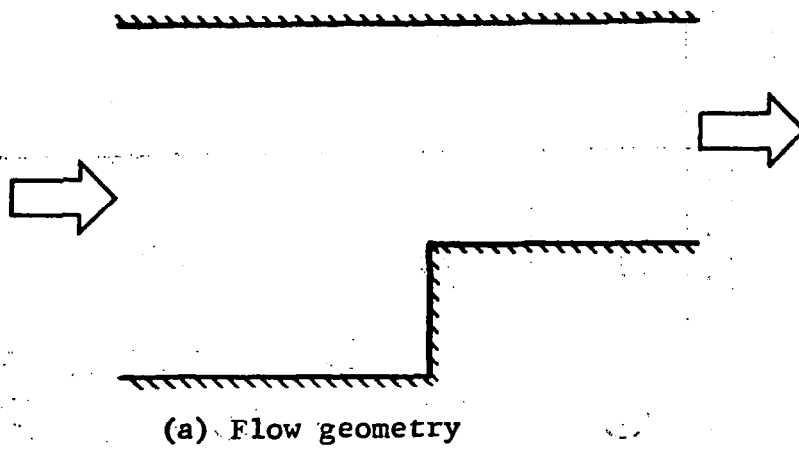
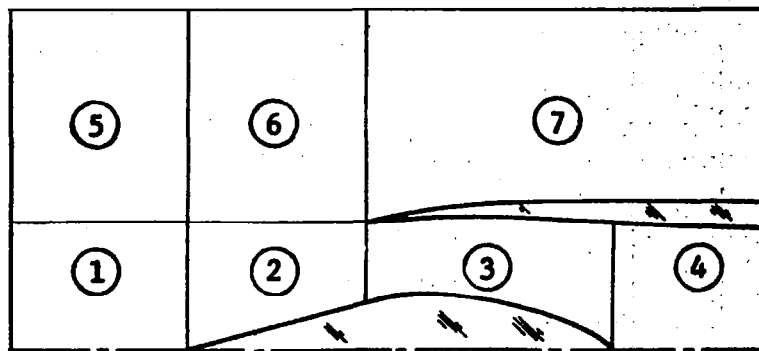
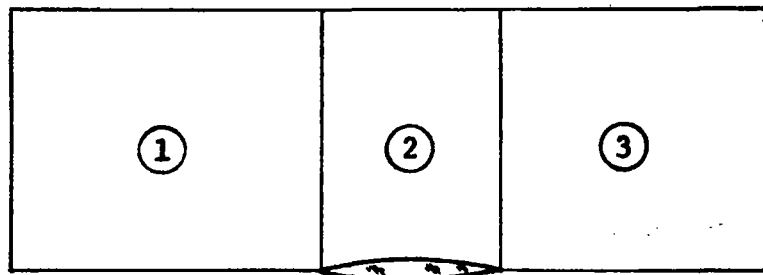


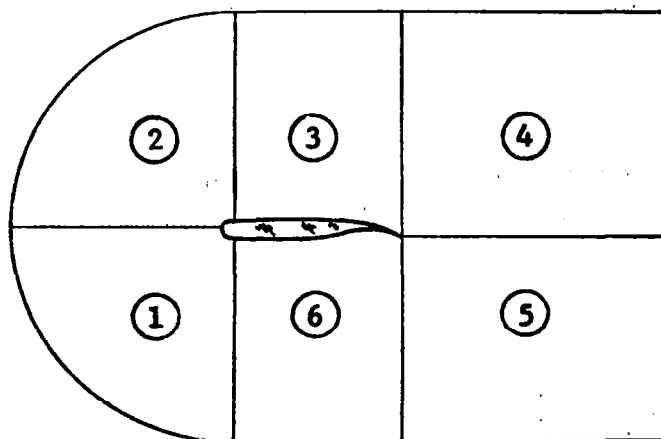
Figure 44.- Example mesh-patch arrangement.



(a) Inlet flow field



(b) Non-lifting airfoil



(c) Lifting airfoil

Figure 45.- Sample mesh-patch arrangement.

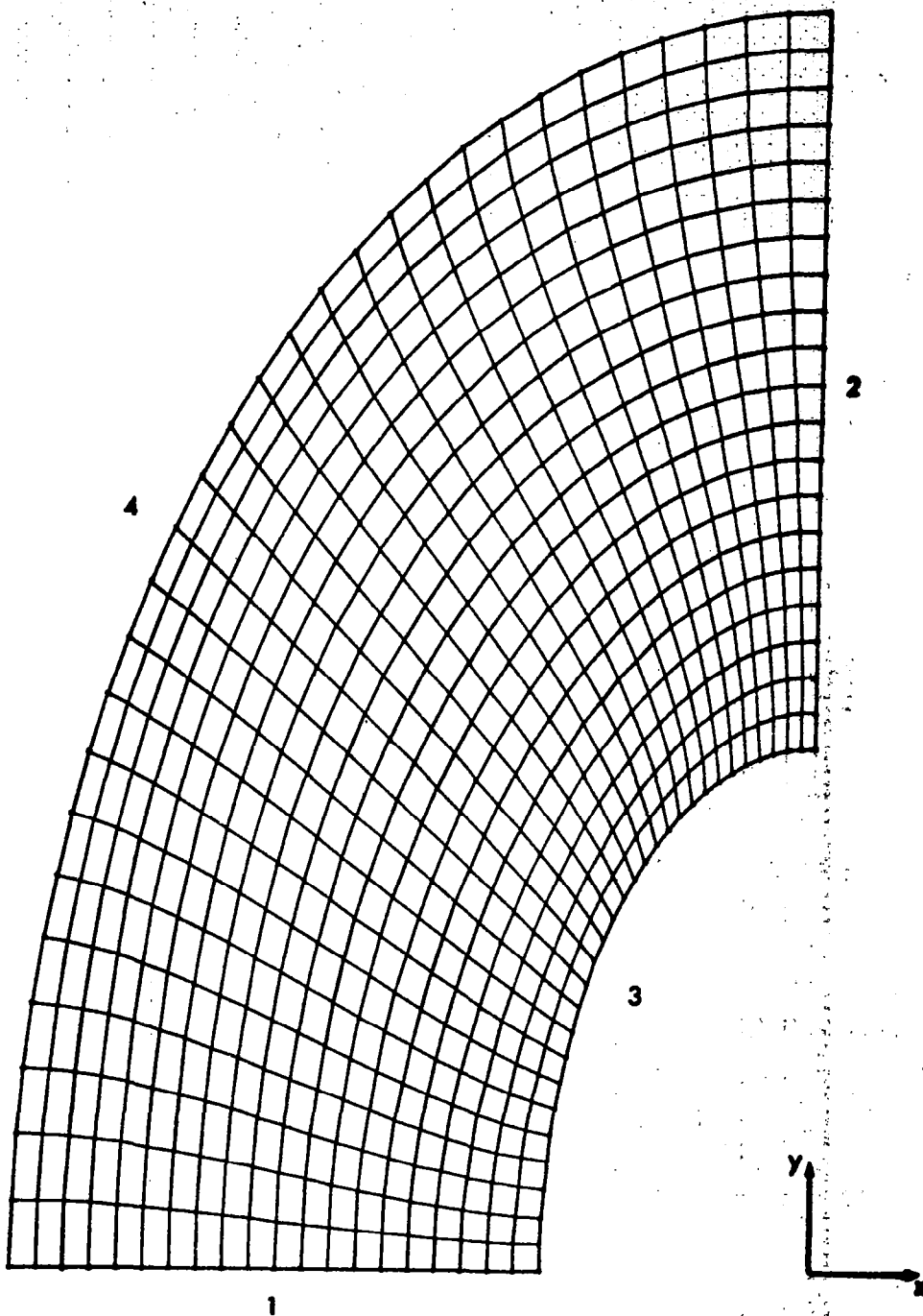


Figure 46.- Sample computer-generated mesh geometry.